

ID Plane Slider Bearing

Introduction

This benchmark model computes the load-carrying capacity of a one-dimensional hydrodynamic slider bearing. The results are compared with analytic expressions obtained by solving the Reynolds equations directly in this simple case (Ref. 1 provides the derivation of the results used).

Model Definition

Although the model is defined in 2D within COMSOL, the Thin-Film Flow, Edge interface is used, which means that it is effectively 1D. The Thin-Film Flow interfaces are defined in this manner to facilitate easy coupling to structural problems in higher dimensions.

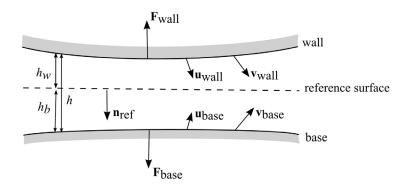


Figure 1: An example illustrating the configuration and definitions used in the Thin-Film Flow, Edge interface. Here \mathbf{u} denotes a displacement vector and \mathbf{v} a velocity vector.

When Thin-Film Flow is assigned to a boundary, the boundary represents a reference surface in the physical device. In practice a small gap exists at the boundary and two impermeable structures, the wall and the base, are located either side of it. The problem formulation, including definitions of the terms used, is shown in Figure 1.

In this example, the model geometry consists of a single line, with length, L (set to 1 mm in the model parameters). The line is located at the origin and aligned with the *x*-axis. The base is coincident with the reference surface and the height of the wall varies linearly along the line. At the origin the wall height is h_0+s_h (2.2 µm in the initial configuration) and at position (L,0) it is h_0 (0.2 µm in the initial configuration). The wall height can therefore be written as:

$$h_w = h_0 + s_h \left(1 - \frac{x}{L}\right)$$

The model defines a number of dimensionless parameters to facilitate easy comparison with theory, and h_0 and s_h are defined in terms of these parameters. A pressure is generated in the bearing by a tangential velocity of the base along the reference plane ($v_{b,x}$).

For no slip boundary conditions at the wall and the base, the Reynolds equation takes the following form for a general stationary problem:

$$\nabla_t \cdot (h \rho \mathbf{v}_{av}) - \rho(\mathbf{v}_w \cdot \nabla_t h_w + \mathbf{v}_b \cdot \nabla_t h_b) = 0$$
$$\mathbf{v}_{av} = \frac{1}{2} (\mathbf{I} - \mathbf{n}_r \mathbf{n}_r^T) (\mathbf{v}_w + \mathbf{v}_b) - \frac{h^2}{12\mu} \nabla_t p_f$$

here ρ is the fluid density, μ is its viscosity and p_f is the pressure developed as a result of the flow (this is the dependent variable in COMSOL). Other terms are defined in Figure 1. For this 1D problem the Reynolds equation is greatly simplified and can be written as:

$$\frac{d}{dx}\left(\frac{\rho h v_{b,x}}{2} - \frac{\rho h^3}{12\mu}\frac{dp_f}{dx}\right) = 0$$

This equation can be integrated directly to give:

$$\frac{1}{\mu}\frac{dp_f}{dx} = \frac{6v_{b,x}}{h^2} + \frac{C}{\rho h^3}$$

where *C* is a constant of integration. *C* is sometimes expressed in terms of the density (ρ_m) and height (h_m) at the position in the bearing (x_m) which the pressure gradient is zero. So given that:

$$\frac{dp_f}{dx} = 0$$
 when $x = x_m$, $\rho = \rho_m$ and $h = h_m$

C is given by:

$$C = -6\rho_m v_{b,x} h_m$$

Using this notation the Reynolds equation becomes:

$$\frac{dp_f}{dx} = 6\mu v_{b,x} \frac{\rho h - \rho_m h_m}{\rho h^3}$$

If ρ and μ are assumed to be independent of the pressure p_f then:

$$\frac{dp_f}{dx} = 6\mu v_{b,x} \frac{h - h_m}{h^3}$$

In Ref. 1 this equation is solved in a dimensionless form, using the dimensionless variables:

$$P = \frac{ps_h^2}{\mu v_{b,x}L} \qquad H = \frac{h}{s_h} \qquad H_m = \frac{h_m}{s_h} \qquad H_0 = \frac{h_0}{s_h} \qquad X = \frac{x}{L}$$

The dimensionless form of the Reynolds equation is therefore:

$$\frac{dP}{dx} = 6\frac{H - H_m}{H^3}$$

Which can be solved to give:

0

$$P = \frac{6X(1-X)}{(H_0 + 1 - X)^2(1 + 2H_0)}$$

With this pressure distribution it is straightforward to show that the pressure takes its maximum value (P_m) at position X_m , where X_m and P_m are given by:

$$X_m = \frac{1 + H_0}{1 + 2H_0} \qquad P_m = \frac{3}{2H_0(1 + H_0)(1 + 2H_0)}$$

The dimensionless flow rate $(Q=2q/(s_hv_{b,x}))$, where q is the flow rate per unit depth, $q=v_{av}\times h$) can be shown to be:

$$Q = \frac{2H_0(1+H_0)}{1+2H_0}$$

Finally the dimensionless total vertical load (L_v) and the horizontal loads acting on the wall $(L_{w,h})$ and the base $(L_{b,h})$ are given by:

4 | ID PLANE SLIDER BEARING

$$\begin{split} L_v &= -\frac{s_h^2}{\mu v_{b,x} L^2} \int_0^L F_{w,v} dx = 6 \ln \left(\frac{1+H_0}{H_0}\right) - \frac{12}{1+2H_0} \\ L_{w,h} &= \frac{s_h}{\mu v_{b,x} L} \int_0^L F_{w,x} dx = 4 \ln \left(\frac{1+H_0}{H_0}\right) - \frac{6}{1+2H_0} \\ L_{b,h} &= \frac{s_h}{\mu v_{b,x} L} \int_0^L F_{b,x} dx = 4 \ln \left(\frac{H_0}{1+H_0}\right) + \frac{6}{1+2H_0} \end{split}$$

The vertical load results from the pressure, while the horizontal loads result from the shear forces from the fluid and the in plane components of the pressure forces. For details of the derivation of these loads, see Ref. 1.

In this model COMSOL solves the bearing problem on a specific geometry, but the results are expressed in the dimensionless forms given above, for ease of comparison with the expressions and plots shown in Ref. 1.

Results and Discussion

The results of the simulation are compared with the analytic expressions discussed above in Figure 2 to Figure 7. In all cases the agreement between COMSOL and the analytic results is excellent. The ratio $H_0=h_0/s_h$ is a measure of the slope of the wall surface-for smaller values of H_0 the slope is greater in relation to the exit height of the bearing. For smaller H_0 larger pressures can be produced in the bearing and higher loads can be sustained by the bearing. The flow rate of gas through the bearing increases with increasing H_0 as the flow tends toward a pure Couette flow, which produces no back pressure.

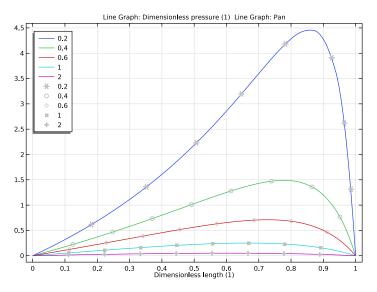


Figure 2: Nondimensional pressure vs distance along the bearing, plotted for different values of the film thickness ratio, $H_0=b_0/s_h$. The computed results are shown as the continuous curves and the theoretical results as the gray symbols.

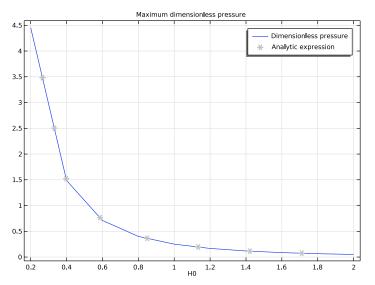


Figure 3: Nondimensional maximum pressure vs film thickness ratio, $H_0=h_0/s_h$. The computed results are shown as the continuous curve and the theoretical result as the gray symbols.

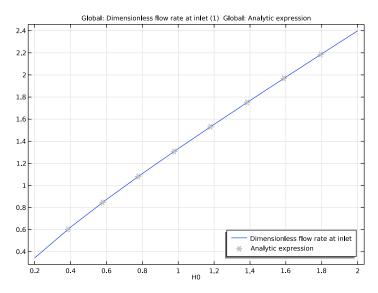


Figure 4: Nondimensional flow rate vs film thickness ratio, $H_0=h_0/s_h$. The computed results are shown as the continuous curve and the theoretical result as the gray symbols.

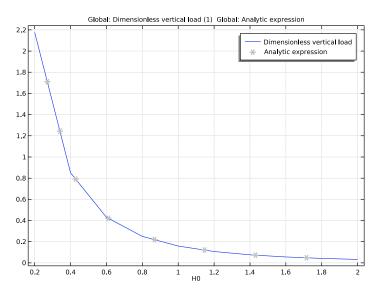


Figure 5: Nondimensional vertical load vs film thickness ratio, $H_0=h_0/s_h$. The computed results are shown as the continuous curve and the theoretical result as the gray symbols.

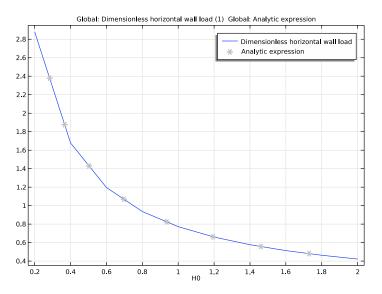


Figure 6: Nondimensional horizontal wall load vs film thickness ratio, $H_0=h_0/s_h$. The computed results are shown as the continuous curve and the theoretical result as the gray symbols.

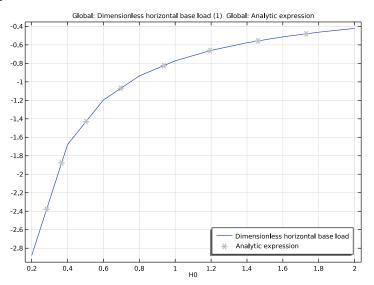


Figure 7: Nondimensional horizontal base load vs film thickness ratio, $H_0=b_0/s_h$. The computed results are shown as the continuous curve and the theoretical result as the gray symbols.

Reference

1. B.J. Hamrock, S.R. Schmid, and B.O. Jacobson, *Fundamentals of Fluid Film Lubrication*, Marcel Dekker, New York, 2004.

This example is based on the discussion entitled *Fixed-Incline Slider Bearing* in section 8.5 of the above reference.

Application Library path: CFD_Module/Thin-Film_Flow/slider_bearing_1d

Modeling Instructions

From the **File** menu, choose **New**.

NEW

In the New window, click 🙆 Model Wizard.

MODEL WIZARD

- I In the Model Wizard window, click **2D**.
- 2 In the Select Physics tree, select Fluid Flow>Thin-Film Flow>Thin-Film Flow, Edge (tffs).
- 3 Click Add.
- 4 Click 🔿 Study.
- 5 In the Select Study tree, select General Studies>Stationary.
- 6 Click 🗹 Done.

GLOBAL DEFINITIONS

Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- **3** In the table, enter the following settings:

Name	Expression	Value	Description	
L	1[mm]	0.001 m	Bearing length	
HO	0.1	0.1	Dimensionless height at end	
sh	2[um]	2E-6 m	Additional height at start	

Name	Expression	Value	Description
h0	sh*H0	2E-7 m	Height at end
Vb	0.1[mm/s]	IE-4 m/s	Velocity of base
muO	0.8[Pa*s]	0.8 Pa·s	Fluid viscosity
rho0	900[kg/m^3]	900 kg/m³	Fluid density

GEOMETRY I

Line Segment I (Is I)

- I In the Geometry toolbar, click 🚧 More Primitives and choose Line Segment.
- 2 In the Settings window for Line Segment, locate the Starting Point section.
- 3 From the Specify list, choose Coordinates.
- 4 Locate the Endpoint section. From the Specify list, choose Coordinates.
- **5** In the **x** text field, type L.
- 6 Click 틤 Build Selected.

DEFINITIONS

Integration 1 (intop1)

- I In the Definitions toolbar, click Nonlocal Couplings and choose Integration.
- 2 In the Settings window for Integration, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 Select Boundary 1 only.

Integration 2 (intop2)

- I In the Definitions toolbar, click Nonlocal Couplings and choose Integration.
- 2 In the Settings window for Integration, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Point.
- 4 Select Point 1 only.

Variables I

- I In the **Definitions** toolbar, click $\partial =$ **Local Variables**.
- 2 In the Settings window for Variables, locate the Variables section.

Name	Expression	Unit	Description
hw	h0+sh*(1-x/L)	m	Wall height
Xd	x/L		Dimensionless length
Pd	pfilm*sh^2/(muO*Vb*L)		Dimensionless pressure
Hd	hw/sh		Dimensionless height
Qd	2*intop2(tffs.vavex* tffs.h)/(sh*Vb)		Dimensionless flow rate at inlet
VLd	<pre>-intop1(tffs.fwally)*sh^2/ (mu0*Vb*L^2)</pre>		Dimensionless vertical load
HLwd	<pre>intop1(tffs.fwallx)*sh/ (mu0*Vb*L)</pre>		Dimensionless horizontal wall load
HLbd	intop1(tffs.fbasex)*sh/ (muO*Vb*L)		Dimensionless horizontal base load
Pmaxan	3/(2*H0*(1+H0)*(1+2*H0))		Analytic maximum pressure
Qan	2*H0*(1+H0)/(1+2*H0)		Analytic dimensionless flow rate
VLan	6*log((H0+1)/H0)-12/(1+2* H0)		Analytic dimensionless vertical load
HLwan	4*log((H0+1)/H0)-6/(1+2*H0)		Analytic dimensionless horizontal wall load
HLban	4*log(H0/(H0+1))+6/(1+2*H0)		Analytic dimensionless horizontal base load

3 In the table, enter the following settings:

Pan

I In the **Definitions** toolbar, click $\begin{bmatrix} f \otimes \\ Q \end{bmatrix}$ **Analytic**.

- 2 In the Settings window for Analytic, type Pan in the Function name text field.
- **3** Locate the **Definition** section. In the **Expression** text field, type $6*X0*(1-X0)/((h0+1-X0)^2*(1+2*h0))$.
- 4 In the **Arguments** text field, type h0, X0.
- 5 Right-click Analytic I (anI) and choose Rename.
- 6 In the Rename Analytic dialog box, type Pan in the New label text field.
- 7 Click OK.

MATERIALS

Material I (mat1)

- I In the Model Builder window, under Component I (compl) right-click Materials and choose Blank Material.
- 2 In the Settings window for Material, locate the Material Contents section.
- **3** In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Dynamic viscosity	mu	muO	Pa∙s	Basic
Density	rho	rho0	kg/m³	Basic

THIN-FILM FLOW, EDGE (TFFS)

Fluid-Film Properties 1

- I In the Model Builder window, under Component I (compl)>Thin-Film Flow, Edge (tffs) click Fluid-Film Properties I.
- 2 In the Settings window for Fluid-Film Properties, locate the Wall Properties section.
- **3** In the h_{w1} text field, type hw.
- 4 Locate the Base Properties section. From the v_b list, choose User defined. Specify the vector as

Vb	x
0	у

STUDY I

Step 1: Stationary

- I In the Model Builder window, under Study I click Step I: Stationary.
- 2 In the Settings window for Stationary, click to expand the Study Extensions section.
- 3 Select the Auxiliary sweep check box.
- 4 Click + Add.
- 5 From the list in the Parameter name column, choose H0 (Dimensionless height at end).
- 6 Click Range.
- 7 In the Range dialog box, type 0.2 in the Start text field.
- 8 In the **Step** text field, type 0.2.
- 9 In the **Stop** text field, type 2.

IO Click Replace.

II In the **Home** toolbar, click **= Compute**.

RESULTS

ID Plot Group 2

- I In the Home toolbar, click 🚛 Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, locate the Data section.
- 3 From the Parameter selection (H0) list, choose From list.
- 4 In the Parameter values (H0) list, choose 0.2, 0.4, 0.6, I, and 2.
- 5 Locate the Legend section. From the Position list, choose Upper left.

Line Graph I

- I Right-click ID Plot Group 2 and choose Line Graph.
- **2** Select Boundary 1 only.
- 3 In the Settings window for Line Graph, locate the y-Axis Data section.
- 4 In the **Expression** text field, type Pd.
- 5 Locate the x-Axis Data section. From the Parameter list, choose Expression.
- 6 In the **Expression** text field, type Xd.
- 7 Click to expand the Legends section. Select the Show legends check box.

Line Graph 2

- I Right-click Line Graph I and choose Duplicate.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- 3 In the **Expression** text field, type Pan(H0, Xd).
- 4 Click to expand the Coloring and Style section. From the Color list, choose Gray.
- 5 Find the Line markers subsection. From the Marker list, choose Cycle.
- 6 Find the Line style subsection. From the Line list, choose None.

Pressure Distribution

- I In the Model Builder window, right-click ID Plot Group 2 and choose Rename.
- 2 In the Rename ID Plot Group dialog box, type Pressure Distribution in the New label text field.
- 3 Click OK.

Maximum 1

I In the **Results** toolbar, click **More Datasets** and choose **Evaluation>Maximum**.

- 2 In the Settings window for Maximum, locate the Settings section.
- **3** From the **Geometry level** list, choose **Line**.

ID Plot Group 3

- I In the Results toolbar, click \sim ID Plot Group.
- 2 In the Settings window for ID Plot Group, locate the Data section.
- 3 From the Dataset list, choose Maximum I.
- 4 Click to expand the Title section. From the Title type list, choose Manual.
- 5 In the Title text area, type Maximum dimensionless pressure.

Global I

- I Right-click ID Plot Group 3 and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
Pd	1	Dimensionless pressure

Global 2

I In the Model Builder window, right-click ID Plot Group 3 and choose Global.

2 In the Settings window for Global, locate the y-Axis Data section.

3 In the table, enter the following settings:

Expression	Unit	Description
Pmaxan		Analytic expression

4 Click to expand the Coloring and Style section. From the Color list, choose Gray.

5 Find the Line markers subsection. From the Marker list, choose Cycle.

6 Find the Line style subsection. From the Line list, choose None.

Maximum Pressure

- I Right-click ID Plot Group 3 and choose Rename.
- 2 In the Rename ID Plot Group dialog box, type Maximum Pressure in the New label text field.
- 3 Click OK.

ID Plot Group 4

I In the **Results** toolbar, click \sim **ID Plot Group**.

- 2 In the Settings window for ID Plot Group, locate the Legend section.
- **3** From the **Position** list, choose **Lower right**.

Global I

- I Right-click ID Plot Group 4 and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
Qd	1	Dimensionless flow rate at inlet

Global 2

- I In the Model Builder window, right-click ID Plot Group 4 and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
Qan		Analytic expression

- **4** Locate the **Coloring and Style** section. Find the **Line style** subsection. From the **Line** list, choose **None**.
- **5** From the **Color** list, choose **Gray**.
- 6 Find the Line markers subsection. From the Marker list, choose Cycle.

Flow Rate

- I Right-click ID Plot Group 4 and choose Rename.
- 2 In the Rename ID Plot Group dialog box, type Flow Rate in the New label text field.
- 3 Click OK.
- ID Plot Group 5

In the Results toolbar, click \sim ID Plot Group.

Global I

- I Right-click ID Plot Group 5 and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
VLd	1	Dimensionless vertical load

Global 2

- I In the Model Builder window, right-click ID Plot Group 5 and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
VLan		Analytic expression

- **4** Locate the **Coloring and Style** section. Find the **Line style** subsection. From the **Line** list, choose **None**.
- 5 From the Color list, choose Gray.
- 6 Find the Line markers subsection. From the Marker list, choose Cycle.

Vertical Load

- I Right-click ID Plot Group 5 and choose Rename.
- 2 In the Rename ID Plot Group dialog box, type Vertical Load in the New label text field.
- 3 Click OK.

ID Plot Group 6

In the **Results** toolbar, click \sim **ID Plot Group**.

Global I

- I Right-click ID Plot Group 6 and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
HLwd	1	Dimensionless horizontal wall load

Global 2

- I In the Model Builder window, right-click ID Plot Group 6 and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
HLwan		Analytic expression

4 Locate the **Coloring and Style** section. Find the **Line style** subsection. From the **Line** list, choose **None**.

- 5 From the Color list, choose Gray.
- 6 Find the Line markers subsection. From the Marker list, choose Cycle.

Horizontal Load, Wall

- I Right-click ID Plot Group 6 and choose Rename.
- 2 In the **Rename ID Plot Group** dialog box, type Horizontal Load, Wall in the **New label** text field.
- 3 Click OK.

ID Plot Group 7

In the **Results** toolbar, click \sim **ID Plot Group**.

Global I

- I Right-click ID Plot Group 7 and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description	
HLbd	1	Dimensionless horizontal base load	

Global 2

- I In the Model Builder window, right-click ID Plot Group 7 and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
HLban		Analytic expression

- **4** Locate the **Coloring and Style** section. Find the **Line style** subsection. From the **Line** list, choose **None**.
- 5 From the Color list, choose Gray.
- 6 Find the Line markers subsection. From the Marker list, choose Cycle.

Horizontal Load, Base

- I In the Model Builder window, click ID Plot Group 7.
- 2 In the Settings window for ID Plot Group, locate the Legend section.
- **3** From the **Position** list, choose **Lower right**.
- 4 Right-click ID Plot Group 7 and choose Rename.

- 5 In the Rename ID Plot Group dialog box, type Horizontal Load, Base in the New label text field.
- 6 Click OK.