

Piezoelectric Tonpilz Transducer

Introduction

The tonpilz (sound mushroom) transducer is a transducer for relatively low frequency, high power sound emission (Ref. 1 and Ref. 2). It is one of the popular acoustic transducer designs that are used for underwater sonar applications. The tonpilz transducer modeled in this tutorial consists of piezoceramic rings stacked between an aluminum head mass and a steel tail mass connected by a steel bolt. This central bolt could be prestressed to control the transducer response. The tail and head mass are used to lower the resonance frequency of the device to the desired level.



Figure 1: A tonpilz transducer. The aluminum head mass is shown in dark gray, the central steel bolt and steel tail mass are shown in light gray and the piezostack actuator with four disks of PZT-4 is shown in green.

Model Definition

In this model, the frequency response of the transducer shown in Figure 1 is studied. In this version of the tutorial, the effect of prestress in the bolt is not considered. The outer curved surface of the steel tail mass is assumed to be fixed. Each of the piezo disks are excited with a 1 V RMS electrical signal. The model determines the deformation in the device, the radiated pressure field and sound pressure level, as well as the spatial radiation pattern sensitivity, the transmitting voltage response (TVR) curve of the transducer, and the directivity index (DI) of the sound beam within the frequency range of 1 kHz to 40 kHz.

The parameters used in this model are shown in Table 1.

NAME	EXPRESSION	DESCRIPTION
Rwater	40[mm]	Water domain radius
Rpml	10[mm]	PML layer thickness
a	25[mm]	Piston head radius
Zeval	-10[m]	Directivity evaluation distance
Vrms	I[V]	RMS drive voltage
V0	sqrt(2)*Vrms	Zero-to-peak drive voltage
f0min	I[kHz]	Minimum operating frequency
f0max	40[kHz]	Maximum operating frequency
f0step	I[kHz]	Frequency step

TABLE I: LIST OF MODELING PARAMETERS.



Figure 2: An axisymmetric section of the model geometry.

Figure 2 shows an axisymmetric view of the actual 3D model. The head mass is exposed to an unbounded region of water. A perfectly matched layer (PML) is used to model the absorption of sound waves as they propagate far away from the sound source. Note that

although the modeling geometry has a rotational symmetry, using a full 3D geometry allows us to capture any vibration mode of the transducer and acoustic mode of the fluid region that do not possess such a symmetry in the solution.

PHYSICS IMPLEMENTATION

The Acoustic-Piezoelectric Interaction, Frequency Domain interface available in the Acoustics Module is used to simulate the multiphysics interactions. This predefined multiphysics interface includes the necessary fundamental physics which are Pressure Acoustics, Solid Mechanics, and Electrostatics. The Pressure Acoustics interface is used to solve the wave equation in the water domain. The Solid Mechanics interface is solved on all structural materials including the PZT-4 disks. The Electrostatics interface is solved on the PZT-4 disks. The multiphysics couplings necessary to model this system are available as predefined nodes under the Multiphysics branch. These couplings are:

Acoustic-Structure Boundary: This node is active on the boundaries that are at the interface of the water domain and transducer head mass. On these boundaries a bidirectional coupling is automatically set up. The fluid pressure evaluated by the Pressure Acoustics interface is applied as a mechanical load in the Solid Mechanics interface. Furthermore, the normal component of the structural acceleration is used as a sound source.

Piezoelectric Effect: This node is active on the PZT-4 domains only and couple the Solid Mechanics and Electrostatics equations solved in these domains via the linear constitutive equations that model the piezoelectric effect by coupling stresses and strains with electric field and electric displacement.

MATERIAL ORIENTATION

The piezoelectric disks are stacked in a way such that alternate disks are poled along opposite directions as shown in Figure 3. This allows us to use a single electrical terminal at the interface of each pair of disks and obtain the piezoelectric actuation effect in each of

the disks along the same direction. Having the piezoelectric strain in-phase in all the disks maximizes the actuation.



Figure 3: Schematic representation of the poling direction and electrical connections in the piezo disks. The blue arrows in the light gray disks indicate a + Z polarization. The red arrows in the dark gray disks indicate a - Z polarization.

In this model, the PZT-4 disks actuate in the d₃₃-mode. Hence two of the disks are poled along the +Z-direction while the other two are poled along the -Z-direction. The default definition of the piezoelectric material properties in the Global Coordinate System automatically creates a +Z polarization. In order to create a -Z polarization, a user-defined Rotated Coordinate System is used. In this coordinate system, the Euler angles are set to $\alpha = 0$, $\beta = \pi$, and $\gamma = 0$. Note that the COMSOL Multiphysics software uses the Z-X-Z convention for Euler angles.

EXTERIOR FIELD CALCULATION

An exterior field calculation is set up on the interface boundaries between the inner water domain and the PML domains. The exterior-field integral type is set to *full integral* which allows computation of both amplitude and phase of the acoustic pressure and sound pressure level (SPL) at any point in space outside the computational domain. These quantities are later used in postprocessing to plot the on-axis pressure variation outside the computational domain as well as to visualize the beam pattern in 2D and 3D polar plots.

MESHING CONSIDERATIONS

In order to accurately resolve the pressure waves within the inner water domain, the maximum mesh element size is specified as $1/5^{\text{th}}$ of the smallest wavelength of interest.

The smallest wavelength is the speed of sound in water (1500 m/s) divided by the largest frequency used in the frequency sweep. The PML is meshed using the Swept feature to create five layers of structured mesh. Additionally a single layer of Boundary Layer Mesh is created within the inner water domain adjacent to the exterior field boundaries. The thickness of this layer is set to $1/100^{\text{th}}$ of the smallest wavelength of interest. The single boundary layer creates a smooth transition between the inner free tetrahedral mesh and the outer structured prism mesh elements thereby yielding a more precise exterior field calculation.





Figure 4: The modeling geometry with the computational mesh.

COMPUTING TRANSDUCER CHARACTERISTICS

The frequency response of the following important transducer characteristics are investigated in this tutorial.

Specific Acoustic Impedance

The specific acoustic impedance Z_{aco} is computed as the ratio of the impedance of the head mass surface exposed to water to the characteristic impedance Z_0 of water as shown in Equation 1. The value of Z_0 is computed from the product of density and speed of sound in water at a temperature of 293.15 K.

$$Z_{\rm aco} = \frac{\left(\int_{A} p da\right) / \left(\int_{A} v_n da\right)}{Z_0} \tag{1}$$

The impedance of the head mass surface is computed from the ratio of the area integral of the acoustic pressure p to the area integral of the normal component of the structural velocity v_n of the surface. In this case the normal velocity is the same as the Z-component of the velocity v_z which in frequency domain can be represented as the product of the Z-component of the structural displacement w, the variable j which is the imaginary number square root of -1 and the angular frequency of vibration ω . The area integral is computed by introducing a nonlocal integration coupling that is assigned to the surface of interest. To avoid division by zero, it is advised to add the built-in constant eps, which denotes the machine precision limit, to the variable w as shown in Table 2 in the definition of the variable Z_{aco}.

Transmitting Voltage Response (TVR)

The TVR represents the sensitivity of the transducer measured at a distance of 1 m and driven at 1 V RMS. This definition can be mathematically expressed using Equation 2.

$$TVR = 20\log \frac{p_{\rm RMS} / V_{\rm RMS}}{1\mu PaV^{-1}}$$
(2)

The RMS pressure at 1 m from the transducer head mass surface can be computed from Equation 3.

$$p_{\rm RMS} = \sqrt{\frac{1}{2}pp^*} \tag{3}$$

In Equation 3, the pressure p at 1 m ahead of the transducer is obtained using the exterior field pressure variable pext by using an expression pext(0,0,-1). Note that the computational domain is much smaller than 1 m but by performing the exterior field calculation you are able to compute the pressure and phase at any point outside the domain. The variable p^* is the complex conjugate of the pressure p and can be expressed in COMSOL using the expression conj(pext(0,0,-1)).

Directivity Index (DI)

The directivity index (DI) gives a measure of how directional the transmitted acoustic beam is (Ref. 3). It is a measure of the ratio of the intensity I_{front} in front of the transducer at a certain distance in the far-field (beyond the Rayleigh radius) to the intensity I_{ave}

transmitted by an omnidirectional source of the same strength. The DI is evaluated at a distance of 10 m in front of the transducer. You can alter the value of the parameter $Z_{\rm eval}$ to compute the DI at a different location. The DI can be computed from Equation 4.

$$DI = 10\log \frac{I_{\text{front}}}{I_{\text{ave}}}$$
 (4)

The quantity I_{front} can be computed from the ratio of the RMS pressure at the desired distance from the transducer to the characteristic impedance Z_0 . The quantity I_{ave} can be computed from the ratio of total radiated power P_{tot} , to the surface area of a sphere having the same radius as the distance from the transducer at which the DI is computed.

The user-defined variables used to compute the transducer characteristics are shown in Table 2.

NAME	EXPRESSION	DESCRIPTION
rho0	intop3(acpr.rho)	Density of water at room temperature
c0	intop3(acpr.c)	Speed of sound in water at room temperature
Zaco	intop2(p)/intop2(acpr.iomega*(w+eps))/ (rho0*c0)	Specific acoustic impedance
pext_l	pext(0,0,-1)	Exterior field pressure at 1 m
prms	<pre>sqrt(0.5*pext_l*conj(pext_l))[Pa]</pre>	RMS pressure at 1 m
TVR	20*log10(prms/Vrms/1[uPa/V])	Transmitting Voltage Response (TVR)
pext_Zeval	pext(0,0,Zeval[1/m])	Exterior field pressure at Zeval
lfront	0.5*pext_Zeval*conj(pext_Zeval)[Pa^2]/ (rho0*c0)	On-axis intensity at Zeval
Ptot	intop l (down(acpr.lx)*acpr.nx+ down(acpr.ly)*acpr.ny+down(acpr.lz)* acpr.nz)	Total radiated power
lave	Ptot/(4*pi*Zeval^2)	Average intensity of monopole source at Zeval
Di	lfront/lave	Intensity directivity

TABLE 2: LIST OF VARIABLES.

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NAME	EXPRESSION	DESCRIPTION
DI	10*log10(Di)	Directivity index of the tonpilz transducer
k0	2*pi*freq/c0	Wave number
DI_fl_pist	10*log10((k0*a)^2/(1-2*besselj(1,2*k0* a)/(2*k0*a)))	Directivity index of flanged piston
SPL_Zeval	intop3(subst(acpr.ffc1.Lp_pext,x,0,y,0,z, Zeval))	SPL at Zeval
SPL_rel	acpr.ffc1.Lp_pext-SPL_Zeval	SPL relative to 0 dB at Zeval

Results and Discussion



Figure 5: A multislice plot showing the total acoustic pressure variation in the water domain and PML at 40 kHz.

Figure 5 shows the total acoustic pressure in the water domain for 40 kHz excitation. The inner water domain in the model captures roughly half a wavelength at this frequency. As expected, the outer surface of the PML layer shows zero pressure which confirms that the PML layer effectively absorbs the outgoing waves. Setting the PML stretching type to

rational allows us to effectively use the PML over a large range of wavelengths and angles of incidence of the pressure waves as encountered in this model.



Figure 6: A multislice plot showing the sound pressure level (SPL) in the water domain and PML at 40 kHz.

Figure 6 shows the sound pressure level (SPL) in the water domain for 40 kHz excitation. Note the 45 dB difference between the SPL near the transducer head mass and the outer surface of the PML layer. This once again confirms the effectiveness of the damping induced by the PML.



Figure 7: A 3D radiation pattern plot showing the sound pressure level (SPL) for 40 kHz excitation at a distance of 10 m from the head mass. The SPL is computed relative to 0 dB at an on-axis distance of 10 m from the head mass.

Figure 7 shows a 3D radiation pattern plot of the exterior field SPL computed at a radial distance of 10 m from the transducer head mass for 40 kHz excitation. The SPL shown here is computed relative to a value of 0 dB at an on-axis distance of 10 m directly ahead of the transducer head mass. This baseline value is computed by the user-defined variable SPL_Zeval and the SPL relative to this quantity is computed by the variable SPL_rel. Note that you can control the baseline value by changing the value of the parameter Zeval. You could also create similar plot for any of the other frequencies solved for and also at other distances outside the computational domain. The later can be controlled by manually specifying the radius of the sphere as the absolute value of the parameter Zeval in the Evaluation section of the Radiation Pattern plot settings.



Figure 8: A polar beam sensitivity plot showing the sound pressure level (SPL) for 1, 10 and 40 kHz excitation at a distance of 10 m from the head mass. The SPL is computed relative to 0 dB at an on-axis distance of 10 m from the head mass.

Figure 8 shows a polar beam sensitivity plot which is also known as the beam pattern. Here the relative exterior field SPL is computed at a radial distance of 10 m from the transducer head mass for 1, 10, and 40 kHz excitation. The polar plot here shows the radiation pattern in the XZ-plane. The right half of the circle from 270° through 0 ° up to 90 ° correspond to the -Z hemisphere. The +Y direction is directed into the plane. The beam pattern shows that at a relatively lower frequency of 1 kHz, the sound emitted is fairly omnidirectional. This is because at low frequencies the transducer works in the piston mode.



Figure 9: Total structural displacement of the transducer at 40 kHz. An exaggerated deformation has been used for better visualization.

Figure 9 shows the total structural displacement of the tonpilz transducer at 40 kHz excitation. At this frequency, the head mass vibrates in a mode whose shape is somewhat toroidal. This produces the lobes in the sound radiation pattern as can be seen in Figure 7 and Figure 8. If you observe the vibration mode for relatively lower frequencies, say 1 kHz, observe that the head mass vibrates mainly along its axis similar to a flanged piston.



Figure 10: A multislice plot of the electric potential distribution within the four PZT-4 disks.

Figure 10 zooms on the piezo stacks. It shows the electric potential distribution through the thickness of the PZT-4 disks. The color bands indicate a successful implementation of the idea described in Figure 3.



Figure 11: A line plot of acoustic pressure variation directly ahead of the transducer head mass up to an on-axis distance of 500 mm.

Figure 11 shows a plot of acoustic pressure vs. z-coordinate along the axis of the transducer for an excitation frequency of 40 kHz. The blue curve shows the pressure variable p that is solved for vs. the z-coordinate of the geometric edge that passes through the radius of the inner water domain vertically downward from the head mass along the transducer axis. The green curve shows how the pressure outside the computational domain can be computed using the exterior-field variable pext by using an expression pext(x,y,z). The variable is evaluated using a parameterized curve dataset to evaluate the variable outside the computational mesh.

This approach shows how to use a relatively small computational domain for efficient modeling and still use the results from the exterior field calculation in postprocessing to visualize both the magnitude and phase of the pressure outside the computational domain.



Figure 12: Frequency response plot of the absolute value, real and imaginary components of the specific acoustic impedance at the interface between the head mass and water.

Figure 12 shows the frequency response of the specific acoustic impedance of the head mass surface that is exposed to water. A resonance and anti-resonance is observed within the frequency range of 20 kHz and 35 kHz.



Figure 13: Transmitting Voltage Response (TVR) as a function of frequency obtained at an on-axis distance of 1 m ahead of the head mass and computed relative to $1 \mu Pa/V$.

Figure 13 shows the variation in the TVR of the transducer as a function of operating frequency. The fairly flat region between 15 kHz and 25 kHz can be particularly useful for sensing applications.



Figure 14: Frequency response of the Directivity Index (DI) computed at an on-axis distance of 10 m from the head mass. The DI of the tonpilz transducer is compared to that of a flanged piston.

Figure 14 shows the Directivity Index (DI) of the tonpilz transducer (blue curve) and compares it with the DI of a flanged piston (green curve). The latter can be computed from analytical expression as shown in Table 2. It is defined by the variable DI_f1_pist. Note that when the tonpilz transducer operates like a piston at lower frequencies, its DI becomes very similar to that of a flanged piston. Another feature worth noting is that within the range of 15 kHz and 25 kHz, the DI of the tonpilz transducer changes from nearly 0 dB to 11 dB while its TVR remains nearly constant. This can make the transducer quite versatile within this operating range.



Figure 15: Total radiated power from the tonpilz transducer within the operating frequency range of 1 kHz to 40 kHz.

Figure 15 shows the total radiated power as a function of the operating frequency of the tonpilz transducer. Note that the acoustic radiated power should be always positive. In order to correctly compute the radiated power at the low frequency range you need to make some changes to the default settings of the Perfectly Matched Layer (PML).

The default settings produce negative radiated power at low frequencies. This happens because at low frequencies the evanescent waves extend into the PML layer. The interaction between the scaled coordinate system in the PML and these waves may create an erroneous energy contribution in the model (can be either positive or negative).

A good way to investigate the performance of the PML is to make a sensitivity analysis on some parameter (for example, the total radiated acoustic power) with respect to changes in the PML parameters. In this model such a rigorous sensitivity analysis is not performed. In general, increasing the curvature factor effectively shifts the resolving power of the PML toward the physical domain, which is necessary in this case since the evanescent components decay in only a fraction of a wavelength. However, if you increase it too much, you may lose resolution in the other end, that is, of the free space wavelength. Assuming that the PMLs work properly at high frequencies for a curvature parameter of 1, you can

in principle perform a convergence study by increasing the value of the curvature parameter until the low-frequency (for 1 kHz) result converges while making sure that the high-frequency (for 40 kHz) result is not affected. It turns out that a value of 5 for the curvature parameter yields good results in this model. A scaling factor of 0.5 further improves the results, but only by a small amount. Decreasing the scaling factor corresponds to compressing the PML layer (shortening it), which in turn effectively increases the mesh resolution within the PML region.



Figure 16: Pressure evaluated outside the computational mesh using the exterior field operator pext(x,y,z).

As discussed above, the pressure (including phase) is evaluated outside the computational domain using the exterior field computation feature. The feature can evaluates the pressure (and phase) at any distance outside computational domain (as already seen in Figure 11) both in the near and in the far-field. The pressure outside the computational mesh is depicted in Figure 16 using the Grid 3D dataset. The sound pressure level is depicted in Figure 17.



Figure 17: Sound pressure level evaluated outside the computational domain using the exterior field feature as well as the sound pressure level inside the domain.

References

1. C. H. Sherman and J. L. Butler, *Transducers and Arrays for Underwater Sound*, Springer, New York, 2007.

2. M. Lasky, "Review of Undersea Acoustics to 1950," J. Acoust. Soc. Am., vol. 61, pp. 283–297, 1976.

3. D. T. Blackstock, Fundamentals of Physical Acoustics, John Wiley & Sons, 2000.

Application Library path: Acoustics_Module/Piezoelectric_Devices/ tonpilz_transducer

Modeling Instructions

From the File menu, choose New.

NEW

In the New window, click 🔗 Model Wizard.

MODEL WIZARD

- I In the Model Wizard window, click 间 3D.
- 2 In the Select Physics tree, select Acoustics>Acoustic-Structure Interaction>Acoustic-Piezoelectric Interaction, Frequency Domain.
- 3 Click Add.
- 4 Click \bigcirc Study.
- 5 In the Select Study tree, select General Studies>Frequency Domain.
- 6 Click **M** Done.

GLOBAL DEFINITIONS

Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- 3 Click 📂 Load from File.
- **4** Browse to the model's Application Libraries folder and double-click the file tonpilz transducer parameters.txt.

GEOMETRY I

- I In the Model Builder window, under Component I (compl) click Geometry I.
- 2 In the Settings window for Geometry, locate the Units section.
- 3 From the Length unit list, choose mm.

Build a plane section of the geometry before revolving it.

Work Plane I (wp1)

- I In the Geometry toolbar, click 📥 Work Plane.
- 2 In the Model Builder window, expand the Geometry I node, then click Work Plane I (wp1).
- 3 In the Settings window for Work Plane, locate the Plane Definition section.
- 4 From the Plane list, choose xz-plane.

Start with the solid domain.

5 Click 📥 Show Work Plane.

Work Plane I (wpl)>Rectangle I (rl)

- I In the Work Plane toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- **3** In the **Width** text field, type 4.
- 4 In the **Height** text field, type 2.
- 5 Locate the **Position** section. In the **yw** text field, type 25.

Work Plane 1 (wp1)>Rectangle 2 (r2)

- I In the Work Plane toolbar, click 📃 Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- **3** In the **Width** text field, type 2.
- 4 In the **Height** text field, type 20.
- 5 Locate the **Position** section. In the **yw** text field, type 5.

Work Plane 1 (wp1)>Rectangle 3 (r3)

- I In the Work Plane toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- 3 In the Width text field, type 6.
- **4** In the **Height** text field, type 10.
- 5 Locate the **Position** section. In the **xw** text field, type 2.
- 6 In the **yw** text field, type 15.

Work Plane I (wp1)>Rectangle 4 (r4)

- I In the Work Plane toolbar, click 📃 Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- **3** In the **Width** text field, type 2.
- 4 In the **Height** text field, type 8.
- 5 Locate the **Position** section. In the **xw** text field, type 4.
- **6** In the **yw** text field, type **7**.
- 7 Click to expand the Layers section. In the table, enter the following settings:

Layer name	Thickness (mm)
Layer 1	2
Layer 2	2
Layer 3	2

Work Plane I (wpl)>Polygon I (poll)

I In the Work Plane toolbar, click / Polygon.

2 In the Settings window for Polygon, locate the Coordinates section.

3 In the table, enter the following settings:

xw (mm)	yw (mm)
0	0
0	5
2	5
2	7
20	7
а	0

Then build the water domain.

Work Plane I (wpI)>Circle I (cI)

I In the Work Plane toolbar, click 🕑 Circle.

2 In the Settings window for Circle, locate the Size and Shape section.

- 3 In the Radius text field, type Rwater+Rpml.
- 4 In the Sector angle text field, type 90.
- 5 Locate the Rotation Angle section. In the Rotation text field, type -90.

6 Click to expand the Layers section. In the table, enter the following settings:

Layer name	Thickness (mm)
Layer 1	Rpml

7 In the Work Plane toolbar, click 🟢 Build All.

8 Click the 🕂 Zoom Extents button in the Graphics toolbar.

Revolve 1 (rev1)

- I In the Model Builder window, right-click Geometry I and choose Revolve.
- 2 In the Settings window for Revolve, locate the Revolution Angles section.
- 3 Click the Angles button.
- 4 In the End angle text field, type 90.
- 5 Click 틤 Build Selected.

Rotate | (rot |)

I In the Geometry toolbar, click 💭 Transforms and choose Rotate.

- 2 Select the object **rev1** only.
- 3 In the Settings window for Rotate, locate the Rotation section.
- 4 In the Angle text field, type range (90,90,270).
- **5** Locate the **Input** section. Select the **Keep input objects** check box.
- 6 Click 🟢 Build All Objects.
- 7 Click the **Zoom Extents** button in the **Graphics** toolbar.



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8 Click the 🔁 Wireframe Rendering button in the Graphics toolbar.

DEFINITIONS

Create domain selections that will be used in the modeling.

Water domain - Inner

- I In the Definitions toolbar, click 🐚 Explicit.
- 2 In the Settings window for Explicit, type Water domain Inner in the Label text field.

3 Select Domains 3, 4, 22, and 32 only.



Water domain - PML

- I In the Definitions toolbar, click 🐚 Explicit.
- ${\bf 2}~$ In the Settings window for ${\bf Explicit},$ type Water domain PML in the ${\bf Label}$ text field.
- **3** Select Domains 1, 2, 21, and 31 only.



Aluminum

- I In the Definitions toolbar, click 🐚 Explicit.
- 2 In the Settings window for Explicit, type Aluminum in the Label text field.
- **3** Select Domains 5, 6, 23, and 33 only.



Steel

- I In the Definitions toolbar, click 🐚 Explicit.
- 2 In the Settings window for Explicit, type Steel in the Label text field.

3 Select Domains 7, 8, 17–20, 24, 29, 30, and 34–36 only.



+Z poled Piezo

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- I In the Definitions toolbar, click 🛯 🐂 Explicit.
- 2 In the Settings window for Explicit, type +Z poled Piezo in the Label text field.
- **3** Select Domains 9, 10, 13, 14, 25, 27, 37, and 39 only.



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-Z poled Piezo

- I In the Definitions toolbar, click 🗞 Explicit.
- 2 In the Settings window for Explicit, type -Z poled Piezo in the Label text field.
- **3** Select Domains 11, 12, 15, 16, 26, 28, 38, and 40 only.



Water domains

- I In the **Definitions** toolbar, click 📑 **Union**.
- 2 In the Settings window for Union, type Water domains in the Label text field.
- 3 Locate the Input Entities section. Under Selections to add, click + Add.
- 4 In the Add dialog box, in the Selections to add list, choose Water domain Inner and Water domain PML.
- 5 Click OK.

Piezo domains

- I In the **Definitions** toolbar, click 💾 **Union**.
- 2 In the Settings window for Union, type Piezo domains in the Label text field.
- 3 Locate the Input Entities section. Under Selections to add, click + Add.
- 4 In the Add dialog box, in the Selections to add list, choose +Z poled Piezo and -Z poled Piezo.
- 5 Click OK.

Solid domains

- I In the **Definitions** toolbar, click **here complement**.
- 2 In the Settings window for Complement, type Solid domains in the Label text field.
- 3 Locate the Input Entities section. Under Selections to invert, click + Add.
- 4 In the Add dialog box, select Water domains in the Selections to invert list.
- 5 Click OK.

Non-PML domains

- I In the **Definitions** toolbar, click **here complement**.
- 2 In the Settings window for Complement, type Non-PML domains in the Label text field.
- 3 Locate the Input Entities section. Under Selections to invert, click + Add.
- 4 In the Add dialog box, select Water domain PML in the Selections to invert list.
- 5 Click OK.

Create boundary selections that will be used further in the modeling.

Ground boundaries

- I In the **Definitions** toolbar, click 🗞 **Explicit**.
- 2 In the Settings window for Explicit, type Ground boundaries in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- **4** Select Boundaries 28, 29, 38, 39, 45, 46, 93, 99, 103, 145, 151, and 155 only.



Voltage boundaries

- I In the Definitions toolbar, click 🐂 Explicit.
- 2 In the Settings window for Explicit, type Voltage boundaries in the Label text field.
- **3** Locate the **Input Entities** section. From the **Geometric entity level** list, choose **Boundary**.
- 4 Select Boundaries 33, 34, 43, 44, 96, 102, 148, and 154 only.





Fixed boundaries

- I In the **Definitions** toolbar, click **here Explicit**.
- 2 In the Settings window for Explicit, type Fixed boundaries in the Label text field.
- **3** Locate the **Input Entities** section. From the **Geometric entity level** list, choose **Boundary**.

4 Select Boundaries 19, 20, 88, and 164 only.



Exterior Field boundaries

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- I In the **Definitions** toolbar, click **herefore Explicit**.
- 2 In the Settings window for Explicit, type Exterior Field boundaries in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.

4 Select Boundaries 9, 10, 82, and 125 only.



Define a nonlocal integration coupling on the exterior-field boundary.

Integration 1 (intop1)

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- I In the Definitions toolbar, click *P* Nonlocal Couplings and choose Integration.
- 2 In the Settings window for Integration, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 From the Selection list, choose Exterior Field boundaries.

Define a nonlocal integration coupling on the acoustic-structure interface.

Integration 2 (intop2)

- I In the Definitions toolbar, click nonlocal Couplings and choose Integration.
- 2 In the Settings window for Integration, locate the Source Selection section.
- **3** From the **Geometric entity level** list, choose **Boundary**.

4 Select Boundaries 14, 15, 85, and 128 only.





Define a nonlocal integration coupling to retrieve values at the summit point of the water domain.

Integration 3 (intop3)

- I In the Definitions toolbar, click *P* Nonlocal Couplings and choose Integration.
- 2 In the Settings window for Integration, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Point.

4 Select Point 46 only.



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Define a rotated system that will be used for the poling of the -Z poled piezoelectric disks.

Rotated System 2 (sys2)

- I In the Definitions toolbar, click \bigvee_{x}^{y} Coordinate Systems and choose Rotated System.
- 2 In the Settings window for Rotated System, locate the Rotation section.
- **3** Find the **Euler angles (Z-X-Z)** subsection. In the β text field, type pi.

Insert a Perfectly Matched Layer to model the absorption of acoustic wave far away from the source.

Perfectly Matched Layer I (pml1)

- I In the Definitions toolbar, click W Perfectly Matched Layer.
- 2 In the Settings window for Perfectly Matched Layer, locate the Domain Selection section.
- 3 From the Selection list, choose Water domain PML.
- 4 Locate the Geometry section. From the Type list, choose Spherical.
- 5 Locate the Scaling section. From the Coordinate stretching type list, choose Rational.
- 6 In the PML scaling factor text field, type 0.5.
- 7 In the PML scaling curvature parameter text field, type 5.

MATERIALS

Add water, aluminum, steel, and piezoelectric materials from the Material Library.

ADD MATERIAL

- I In the Home toolbar, click 👬 Add Material to open the Add Material window.
- 2 Go to the Add Material window.
- 3 In the tree, select Built-in>Water, liquid.
- 4 Click Add to Component in the window toolbar.
- 5 In the tree, select Built-in>Aluminum.
- 6 Click Add to Component in the window toolbar.
- 7 In the tree, select Built-in>Steel AISI 4340.
- 8 Click Add to Component in the window toolbar.
- 9 In the tree, select Piezoelectric>Lead Zirconate Titanate (PZT-4).
- **IO** Click **Add to Component** in the window toolbar.
- II In the Home toolbar, click 🙀 Add Material to close the Add Material window.

MATERIALS

Water, liquid (mat1)

- I In the Model Builder window, under Component I (compl)>Materials click Water, liquid (matl).
- 2 In the Settings window for Material, locate the Geometric Entity Selection section.
- 3 From the Selection list, choose Water domains.

Aluminum (mat2)

- I In the Model Builder window, click Aluminum (mat2).
- 2 In the Settings window for Material, locate the Geometric Entity Selection section.
- 3 From the Selection list, choose Aluminum.

Steel AISI 4340 (mat3)

- I In the Model Builder window, click Steel AISI 4340 (mat3).
- 2 In the Settings window for Material, locate the Geometric Entity Selection section.
- **3** From the **Selection** list, choose **Steel**.

Lead Zirconate Titanate (PZT-4) (mat4)

I In the Model Builder window, click Lead Zirconate Titanate (PZT-4) (mat4).

- 2 In the Settings window for Material, locate the Geometric Entity Selection section.
- **3** From the Selection list, choose Piezo domains.

PRESSURE ACOUSTICS, FREQUENCY DOMAIN (ACPR)

Define physics settings and boundary conditions.

- In the Model Builder window, under Component I (compl) click Pressure Acoustics, Frequency Domain (acpr).
- 2 In the Settings window for Pressure Acoustics, Frequency Domain, locate the Domain Selection section.
- 3 From the Selection list, choose Water domains.
- 4 Locate the Sound Pressure Level Settings section. From the Reference pressure for the sound pressure level list, choose Use reference pressure for water.
- 5 Locate the Typical Wave Speed for Perfectly Matched Layers section. In the c_{ref} text field, type 1500[m/s].

Exterior Field Calculation 1

- I In the Physics toolbar, click 🔚 Boundaries and choose Exterior Field Calculation.
- **2** In the Settings window for Exterior Field Calculation, locate the Boundary Selection section.
- **3** From the Selection list, choose Exterior Field boundaries.
- 4 Locate the Exterior Field Calculation section. From the Condition in the $z = z_0$ plane list, choose Symmetric/Infinite sound hard boundary.

SOLID MECHANICS (SOLID)

- I In the Model Builder window, under Component I (comp I) click Solid Mechanics (solid).
- 2 In the Settings window for Solid Mechanics, locate the Domain Selection section.
- **3** From the Selection list, choose Solid domains.

Piezoelectric Material I

- I In the Model Builder window, under Component I (comp1)>Solid Mechanics (solid) click Piezoelectric Material I.
- 2 In the Settings window for Piezoelectric Material, locate the Domain Selection section.
- 3 From the Selection list, choose +Z poled Piezo.

Piezoelectric Material 2

I In the Physics toolbar, click 🔚 Domains and choose Piezoelectric Material.

- 2 In the Settings window for Piezoelectric Material, locate the Domain Selection section.
- 3 From the Selection list, choose -Z poled Piezo.
- 4 Locate the Coordinate System Selection section. From the Coordinate system list, choose Rotated System 2 (sys2).

Fixed Constraint I

- I In the Physics toolbar, click 🔚 Boundaries and choose Fixed Constraint.
- 2 In the Settings window for Fixed Constraint, locate the Boundary Selection section.
- 3 From the Selection list, choose Fixed boundaries.

ELECTROSTATICS (ES)

- I In the Model Builder window, under Component I (compl) click Electrostatics (es).
- 2 In the Settings window for Electrostatics, locate the Domain Selection section.
- 3 From the Selection list, choose Piezo domains.

Ground I

- I In the Physics toolbar, click 🔚 Boundaries and choose Ground.
- 2 In the Settings window for Ground, locate the Boundary Selection section.
- **3** From the Selection list, choose Ground boundaries.

Electric Potential 1

- I In the Physics toolbar, click 🔚 Boundaries and choose Electric Potential.
- 2 In the Settings window for Electric Potential, locate the Boundary Selection section.
- **3** From the Selection list, choose Voltage boundaries.
- **4** Locate the **Electric Potential** section. In the V_0 text field, type V0.

Mesh the geometry; create a tetrahedral mesh in the solid and the water-inner domains and create a swept mesh in the PML.

MESH I

Free Tetrahedral I

- I In the Mesh toolbar, click \land Free Tetrahedral.
- 2 In the Settings window for Free Tetrahedral, locate the Domain Selection section.
- **3** From the **Geometric entity level** list, choose **Domain**.
- 4 From the Selection list, choose Non-PML domains.

Define a mesh size in the water domain to ensure that the smallest wavelength is resolved by at least 5 elements (here we select lambda0/6).

Size 1

- I Right-click Free Tetrahedral I and choose Size.
- 2 In the Settings window for Size, locate the Geometric Entity Selection section.
- 3 From the Selection list, choose Water domains.
- 4 Locate the **Element Size** section. Click the **Custom** button.
- 5 Locate the Element Size Parameters section. Select the Maximum element size check box.
- 6 In the associated text field, type 1500[m/s]/f0max/6.

Size 2

- I In the Model Builder window, right-click Free Tetrahedral I and choose Size.
- 2 In the Settings window for Size, locate the Geometric Entity Selection section.
- **3** From the Selection list, choose Solid domains.
- 4 Locate the **Element Size** section. Click the **Custom** button.
- 5 Locate the Element Size Parameters section. Select the Resolution of narrow regions check box.
- 6 In the associated text field, type 2.

This setting ensures at least two mesh elements in the narrow layers of the piezo disks and bolt.

Swept I

In the Mesh toolbar, click A Swept.

Distribution I

I Right-click Swept I and choose Distribution.

Create a boundary layer at the external boundaries of the water domain. This will ensure numerically well defined normal gradients used in the exterior-field calculation feature. Turn off the smoothing option.

Boundary Layers 1

- I In the Mesh toolbar, click Moundary Layers.
- 2 In the Settings window for Boundary Layers, locate the Geometric Entity Selection section.
- 3 From the Geometric entity level list, choose Domain.
- 4 From the Selection list, choose Water domain Inner.
- **5** Click to expand the **Transition** section. Clear the **Smooth transition to interior mesh** check box.

Boundary Layer Properties

- I In the Model Builder window, click Boundary Layer Properties.
- **2** In the **Settings** window for **Boundary Layer Properties**, locate the **Boundary Selection** section.
- **3** From the Selection list, choose Exterior Field boundaries.
- 4 Locate the Layers section. In the Number of layers text field, type 1.
- 5 From the Thickness specification list, choose First layer.
- 6 In the Thickness text field, type 1500[m/s]/f0max/6/20.
- 7 In the Model Builder window, right-click Mesh I and choose Build All.
- 8 Click the V Go to Default View button in the Graphics toolbar.

DEFINITIONS

Import the file containing the variable definitions. These variables will mainly be used for postprocessing calculations.

Variables 1

- I In the Model Builder window, under Component I (compl) right-click Definitions and choose Variables.
- 2 In the Settings window for Variables, locate the Variables section.
- **3** Click **b** Load from File.
- **4** Browse to the model's Application Libraries folder and double-click the file tonpilz_transducer_variables.txt.

STUDY I

Step 1: Frequency Domain

- I In the Model Builder window, under Study I click Step I: Frequency Domain.
- 2 In the Settings window for Frequency Domain, locate the Study Settings section.
- 3 From the Frequency unit list, choose kHz.
- 4 In the **Frequencies** text field, type range(fOmin,fOstep,fOmax).
- **5** In the **Home** toolbar, click **= Compute**.

RESULTS

Acoustic Pressure (acpr)

I In the Settings window for 3D Plot Group, locate the Plot Settings section.

2 From the View list, choose View I.

Delete the default surface plot and replace it by a multislice plot, which shows the value of the acoustic pressure inside the water domain.

Surface 1

- I In the Model Builder window, expand the Acoustic Pressure (acpr) node.
- 2 Right-click Results>Acoustic Pressure (acpr)>Surface I and choose Delete.

Acoustic Pressure (acpr)

In the Model Builder window, under Results click Acoustic Pressure (acpr).

Multislice 1

- I In the Acoustic Pressure (acpr) toolbar, click 间 More Plots and choose Multislice.
- 2 In the Settings window for Multislice, locate the Coloring and Style section.
- 3 From the Color table list, choose Wave.
- 4 From the Scale list, choose Linear symmetric.
- 5 In the Acoustic Pressure (acpr) toolbar, click **I** Plot.
- 6 Click the **Zoom Extents** button in the **Graphics** toolbar.

This plot should look like Figure 5.

Sound Pressure Level (acpr)

- I In the Model Builder window, under Results click Sound Pressure Level (acpr).
- 2 In the Settings window for 3D Plot Group, locate the Plot Settings section.
- 3 From the View list, choose View I.

Do the same as for the acoustic pressure plot.

Surface 1

- I In the Model Builder window, expand the Sound Pressure Level (acpr) node.
- 2 Right-click Results>Sound Pressure Level (acpr)>Surface I and choose Delete.

Sound Pressure Level (acpr)

In the Model Builder window, under Results click Sound Pressure Level (acpr).

Multislice 1

- I In the Sound Pressure Level (acpr) toolbar, click 间 More Plots and choose Multislice.
- In the Settings window for Multislice, click Replace Expression in the upper-right corner of the Expression section. From the menu, choose Component I (compl)>
 Pressure Acoustics, Frequency Domain>Pressure and sound pressure level>acpr.Lp_t Total sound pressure level dB.

3 In the Sound Pressure Level (acpr) toolbar, click 💿 Plot.

This plot should look like Figure 6.

Exterior-Field Sound Pressure Level (acpr)

Plot the sound pressure level at 10 meters relative to the value on the Z-axis

Radiation Pattern 1

- I In the Model Builder window, click Radiation Pattern I.
- 2 In the Settings window for Radiation Pattern, locate the Expression section.
- 3 In the **Expression** text field, type SPL_rel.
- **4** Locate the **Evaluation** section. Find the **Angles** subsection. From the **Restriction** list, choose **Manual**.
- **5** In the θ start text field, type **90**.
- **6** In the θ range text field, type 90.
- 7 Find the Sphere subsection. From the Sphere list, choose Manual.
- 8 In the Radius text field, type abs(Zeval).
- 9 In the Exterior-Field Sound Pressure Level (acpr) toolbar, click 💽 Plot.

Change the view in the graphics window to obtain a plot similar to Figure 7.

Plot the Exterior-Field Sound Pressure Level in XZ-plane for several frequencies.

Relative polar beam sensitivity

- I In the Model Builder window, expand the Exterior-Field Pressure (acpr) node, then click Results>Exterior-Field Sound Pressure Level xy-plane (acpr).
- 2 In the Settings window for Polar Plot Group, type Relative polar beam sensitivity in the Label text field.
- 3 Locate the Data section. From the Parameter selection (freq) list, choose From list.
- 4 In the Parameter values (freq (kHz)) list, choose 1, 10, and 40.

Radiation Pattern 1

- I In the Model Builder window, expand the Relative polar beam sensitivity node, then click Radiation Pattern I.
- 2 In the Settings window for Radiation Pattern, locate the Expression section.
- **3** In the **Expression** text field, type SPL_rel.
- **4** Locate the **Evaluation** section. Find the **Angles** subsection. From the **Restriction** list, choose **Manual**.
- **5** In the ϕ start text field, type -90.

- **6** In the ϕ range text field, type 180.
- 7 Find the **Reference direction** subsection. In the **x** text field, type 0.
- **8** In the **z** text field, type -1.

The reference direction defines that 0 deg. in the polar plot corresponds to the negative z-axis direction.

9 Find the Normal vector subsection. In the y text field, type 1.

IO In the **z** text field, type 0.

II Find the Evaluation distance subsection. In the Radius text field, type abs(Zeval).

Relative polar beam sensitivity

- I In the Model Builder window, click Relative polar beam sensitivity.
- 2 In the **Relative polar beam sensitivity** toolbar, click **O** Plot.

This plot should look like Figure 8.

Replace the stress plot by displacement in solid.

Displacement

- I In the Model Builder window, under Results click Stress (solid).
- 2 In the Settings window for 3D Plot Group, type Displacement in the Label text field.
- 3 Locate the Plot Settings section. From the View list, choose View I.

Volume 1

- I In the Model Builder window, expand the Displacement node, then click Volume I.
- 2 In the Settings window for Volume, click Replace Expression in the upper-right corner of the Expression section. From the menu, choose Component I (compl)>Solid Mechanics> Displacement>solid.disp - Displacement magnitude - m.

Displacement

- I In the Model Builder window, click Displacement.
- **2** In the **Displacement** toolbar, click **Displacement** toolbar, click

This plot should look like Figure 9.

DEFINITIONS

Create a new view to plot the electric potential in the piezoelectric disks.

View 5

In the Model Builder window, under Component I (compl) right-click Definitions and choose View.

RESULTS

Electric Potential (es)

- I In the Model Builder window, expand the View 5 node, then click Results> Electric Potential (es).
- 2 In the Electric Potential (es) toolbar, click **O** Plot.

Use the **Zoom Box** in the graphics window to obtain a plot similar to Figure 10.

Insert a 1D plot group to obtain the on-axis pressure graph shown in Figure 11.

First, create a parameterized curve dataset to enable plotting of the exterior-field variable outside the computational mesh: from the edge of the exterior-field boundary to 500 mm in front of the transducer.

Parameterized Curve 3D I

- I In the Results toolbar, click More Datasets and choose Parameterized Curve 3D.
- 2 In the Settings window for Parameterized Curve 3D, locate the Expressions section.
- 3 In the z text field, type (1-s)*(-1.02*Rwater)-s*500[mm].
- **4** Select the **Only evaluate globally defined expressions** check box.

On-axis Pressure

- I In the Results toolbar, click \sim ID Plot Group.
- 2 In the Settings window for ID Plot Group, type On-axis Pressure in the Label text field.
- 3 Locate the Data section. From the Parameter selection (freq) list, choose Last.
- 4 Click to expand the **Title** section. From the **Title type** list, choose **Label**.
- 5 Locate the Plot Settings section. Select the x-axis label check box.
- 6 In the associated text field, type Distance from the transducer (mm).
- 7 Select the y-axis label check box.
- 8 In the associated text field, type Pressure (Pa).

Line Graph I

- I Right-click On-axis Pressure and choose Line Graph.
- 2 Select Edge 136 only.
- In the Settings window for Line Graph, click Replace Expression in the upper-right corner of the y-Axis Data section. From the menu, choose Component I (compl)>
 Pressure Acoustics, Frequency Domain>Pressure and sound pressure level>p Pressure Pa.

- 4 Locate the x-Axis Data section. From the Parameter list, choose Expression.
- **5** In the **Expression** text field, type **z**.
- 6 Click to expand the Legends section. Select the Show legends check box.
- 7 From the Legends list, choose Manual.
- 8 In the table, enter the following settings:

Legends

Domain evaluation

Line Graph 2

- I In the Model Builder window, right-click On-axis Pressure and choose Line Graph.
- 2 In the Settings window for Line Graph, locate the Data section.
- 3 From the Dataset list, choose Parameterized Curve 3D I.
- 4 From the Parameter selection (freq) list, choose Last.
- 5 Locate the y-Axis Data section. In the Expression text field, type pext(x,y,z).
- 6 Locate the x-Axis Data section. From the Parameter list, choose Expression.
- 7 In the **Expression** text field, type z.
- 8 Click to expand the Coloring and Style section. From the Color list, choose Cycle (reset).
- 9 Find the Line style subsection. From the Line list, choose Dashed.
- 10 Locate the Legends section. Select the Show legends check box.
- II From the Legends list, choose Manual.
- **12** In the table, enter the following settings:

Legends

Exterior field evaluation

I3 In the **On-axis Pressure** toolbar, click **OD Plot**.

Insert a 1D plot to obtain the calculated specific acoustic impedance shown in Figure 12.

Specific Acoustic Impedance

- I In the Home toolbar, click 🚛 Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Specific Acoustic Impedance in the Label text field.
- 3 Locate the Title section. From the Title type list, choose Label.
- 4 Locate the Plot Settings section. Select the y-axis label check box.

- **5** In the associated text field, type $Z/(\rb c)$.
- 6 Locate the Legend section. From the Position list, choose Upper left.

Global I

- I Right-click Specific Acoustic Impedance and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
abs(Zaco)	1	
real(Zaco)	1	
imag(Zaco)	1	

- 4 Click to expand the Legends section. From the Legends list, choose Manual.
- **5** In the table, enter the following settings:

Legends

Absolute value

Real part

Imaginary part

6 In the Specific Acoustic Impedance toolbar, click 💿 Plot.

Specific Acoustic Impedance

Duplicate the last plot group and replace it by the transmitting voltage response to obtain the graph shown in Figure 13.

Transmitting Voltage Response

- I In the Model Builder window, right-click Specific Acoustic Impedance and choose Duplicate.
- 2 In the Settings window for ID Plot Group, type Transmitting Voltage Response in the Label text field.
- **3** Locate the **Plot Settings** section. In the **y-axis label** text field, type TVR (dB rel. 1\mu Pa/V).
- 4 Locate the Legend section. From the Position list, choose Lower right.

Global I

I In the Model Builder window, expand the Transmitting Voltage Response node, then click Global I.

- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** Ctrl-click to select table rows 2 and 3.
- 4 Click **Delete**.
- **5** In the table, enter the following settings:

Expression	Unit	Description
TVR		Transmitting Voltage Response (TVR)

- 6 Locate the Legends section. Clear the Show legends check box.
- 7 In the Transmitting Voltage Response toolbar, click 💿 Plot.

An alternative approach to create the above plot is to use the Octave Band Plot. Set the geometric-entity level to global, set the expression to pext(0,0,-1), reference to 1[uPa/V]/Vrms, and the style to continuous.

Specific Acoustic Impedance

Duplicate the previous plot group and modify it to plot the directivity index like in Figure 14.

Directivity Index (DI)

- I In the Model Builder window, right-click Specific Acoustic Impedance and choose Duplicate.
- 2 In the Settings window for ID Plot Group, type Directivity Index (DI) in the Label text field.
- 3 Locate the Plot Settings section. In the y-axis label text field, type DI (dB).
- 4 Locate the Legend section. From the Position list, choose Lower left.

Global I

- I In the Model Builder window, expand the Directivity Index (DI) node, then click Global I.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
DI		Directivity index of Tonpilz transducer
DI_fl_pist		Directivity index of flanged piston

- 4 Click to select row number 3 in the table.
- 5 Click **Delete**.
- 6 Locate the Legends section. From the Legends list, choose Automatic.

7 In the Directivity Index (DI) toolbar, click 💽 Plot.

Transmitting Voltage Response

Duplicate the previous plot group and modify it to total radiated power like in Figure 15.

Total Radiated Power

- I In the Model Builder window, right-click Transmitting Voltage Response and choose Duplicate.
- 2 In the Settings window for ID Plot Group, type Total Radiated Power in the Label text field.
- 3 Locate the Plot Settings section. Clear the y-axis label check box.
- **4** Locate the Legend section. From the **Position** list, choose **Upper left**.

Global I

- I In the Model Builder window, expand the Total Radiated Power node, then click Global I.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
Ptot	mW	Total radiated power

4 In the **Total Radiated Power** toolbar, click **I Plot**.

Create a grid dataset in order to plot the exterior-field pressure pext(x,y,z) outside of the computational mesh. This will reproduce Figure 16.

Grid 3D I

- I In the **Results** toolbar, click **More Datasets** and choose **Grid>Grid 3D**.
- 2 In the Settings window for Grid 3D, locate the Parameter Bounds section.
- **3** Find the **First parameter** subsection. In the **Maximum** text field, type **0**.
- 4 Find the Second parameter subsection. In the Minimum text field, type 200.
- 5 In the Maximum text field, type 200.
- 6 Find the Third parameter subsection. In the Maximum text field, type -200.
- 7 Click to expand the Grid section. In the x resolution text field, type 2.
- 8 In the y resolution text field, type 150.
- 9 In the z resolution text field, type 150.

Exterior Pressure Slice

I In the Results toolbar, click 间 3D Plot Group.

2 In the Settings window for 3D Plot Group, type Exterior Pressure Slice in the Label text field.

Surface 1

- I Right-click Exterior Pressure Slice and choose Surface.
- 2 In the Settings window for Surface, locate the Data section.
- 3 From the Dataset list, choose Grid 3D I.
- 4 Locate the Expression section. In the Expression text field, type if (sqrt(x²+y²+ z²)>Rwater, pext(x,y,z),NaN).

The if() statement ensures that pext(x,y,z) is only plotted outside of the exteriorfield calculation boundary where it makes mathematical sense to do so. The value NaN will plot nothing.

- 5 Locate the Coloring and Style section. From the Color table list, choose Wave.
- 6 From the Scale list, choose Linear symmetric.
- 7 In the Exterior Pressure Slice toolbar, click 💽 Plot.
- 8 Click the YZ Go to YZ View button in the Graphics toolbar.

Now, create a second plot that represents the sound pressure level outside of the transducer and reproduce Figure 17. The SPL is also plotted inside the transducer.

Exterior SPL Slice

- I In the Home toolbar, click 🚛 Add Plot Group and choose 3D Plot Group.
- 2 In the Settings window for 3D Plot Group, type Exterior SPL Slice in the Label text field.
- 3 Locate the Data section. From the Parameter value (freq (kHz)) list, choose 20.

Surface 1

- I Right-click Exterior SPL Slice and choose Surface.
- 2 In the Settings window for Surface, locate the Data section.
- 3 From the Dataset list, choose Grid 3D I.
- 4 From the Parameter value (freq (kHz)) list, choose 20.
- 5 Locate the Expression section. In the Expression text field, type if (sqrt(x²+y²+ z²)>Rwater,acpr.efc1.Lp_pext,NaN).

Slice 1

- I In the Model Builder window, right-click Exterior SPL Slice and choose Slice.
- 2 In the Settings window for Slice, locate the Expression section.

- **3** In the **Expression** text field, type acpr.Lp_t.
- 4 Locate the Plane Data section. In the Planes text field, type 1.
- 5 Click to expand the Inherit Style section. From the Plot list, choose Surface I.

Add a selection to the slice plot such that variables are not plotted in the PML domain. Use it to also plot the SPL inside the computational domain.

Selection 1

- I Right-click Slice I and choose Selection.
- 2 Select Domains 3–20, 22–30, and 32–40 only.

You have selected all domains except the PML domains. Simply select all domains (use Ctrl+A) and then deselect the PML.

3 In the Exterior SPL Slice toolbar, click 💿 Plot.