

# Loudspeaker Spider Optimization

# Introduction

The suspension of a loudspeaker is designed to keep the cone and dust cap in place and avoid any rocking movement of the voice coil. At low frequencies, where the displacement of the cone and dust cap is significant, the stiffness of the suspension varies along the stroke of the voice coil. The parameter is known as  $C_{MS}(x)$ . This variation, or nonlinearity, can play a significant role in the distortion created by the speaker.

This model performs shape optimization of the design of the spider, a thin membrane-like mechanical element, which is a part of the suspension of the loudspeaker. By changing the shape of the spider, it is possible to create a suspension system that behaves linearly (having a nearly constant stiffness  $C_{\rm MS}$ ) all through the range of movement of the voice coil.

The model sets up constraints in the shape variables to make sure that the thickness of the spider remains constant. The final design features a loudspeaker suspension that behaves marginally nonlinearly through the voice coil stroke. This is an improvement over the traditional design, as nonlinearities in the mechanical behavior of the suspension introduce distortion when reproducing signals of large amplitude.

The geometry and simulation parameters are similar to those used in the tutorial *Loudspeaker Driver — Frequency-Domain Analysis* available in the Acoustics Module Application Library. More insight into the vibroacoustics analysis of that geometry can be gained by reading the documentation of that model.

This tutorial model illustrates:

- How to obtain the compliance curve of a loudspeaker
- How geometric nonlinearity can be used in an optimization problem
- How the shape of a boundary can be carried over to another boundary to reflect manufacturing constraints, in this case a constant thickness
- How a single component and physics can be used to capture two different designs

# Model Definition

The suspension of a loudspeaker is designed to keep the speaker cone in place and avoid any rocking movement of the voice coil. Figure 1 shows the main components of a loudspeaker, whit the suspension system composed of the surround and the spider.

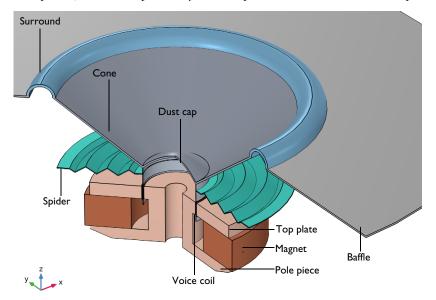


Figure 1: Geometry of the modeled loudspeaker driver.

When loudspeakers are excited at high frequencies, the displacement of the voice coil is relatively small. This means that at high frequencies, the compliance of the suspension along the path of the coil is almost constant. At low frequencies, the voice coil displacement becomes significant and the variation of the compliance,  $C_{MS}(x)$ , becomes more relevant. This variation of the compliance along the path of the coil is an undesired effect as it introduces distortion in the speaker. This effect is further explained in Figure 2.

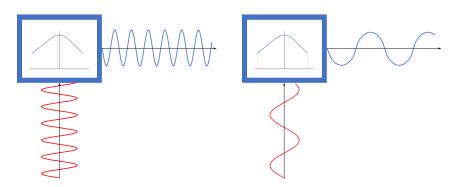


Figure 2: Displacement of the voice coil (blue curves along the horizontal axis) when a high frequency (left) and low frequency (right) current travels through the voice coil (red curves along the vertical axis). At high frequencies, the displacement of the voice coil is small, meaning that the compliance remains almost constant through the movement, producing negligible distortion. At low frequencies, the large displacement of the coil means that the compliance will vary along the path, generating distortion in the displacement and the generated sound.

Although novelty designs exist, most loudspeakers use spiders made of a cloth membrane formed in a concentric zigzag pattern around the former. This pattern is included in the cloth to increase the compliance of the spider and make possible a large stroke of the voice coil, which is needed to produce sound at low frequencies. Figure 1 shows a traditional spider design.

The first study in the model analyzes a traditional design to include as a reference. In the second study, starting from a flat design, a shape optimization problem is defined to obtain a suspension system that behaves linearly through the stroke of the voice coil, that is, it ideally has a constant compliance  $C_{\rm MS}$ .

# Results and Discussion

The von Mises stress distribution of a traditional spider is shown in Figure 3. Note that both the surround and the spider show deformation, as both elements contribute to the total compliance of the suspension system.

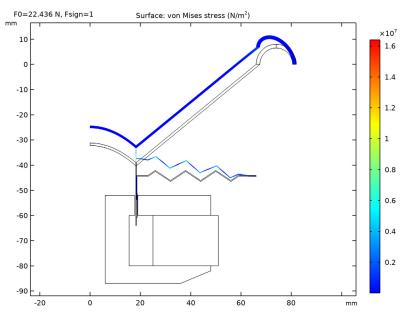


Figure 3: von Mises stress of the traditional spider.

The shape optimization study starts with a flat design of the spider and improves the objective by changing the shape of the bottom boundary. The objective is to minimize the difference between the compliance of the suspension system from an ideal linear suspension system. The change of the bottom boundary is carried over to the top boundary using a *General Extrusion* operator. The deformation of the top boundary accounts for the displacement and change in the normal direction of the bottom surface to guarantee that the thickness of the spider remains constant.

Figure 4 shows the relative normal boundary displacement and the von Mises stress of the optimized design.

Figure 5 shows the force versus displacement and compliance curves of the traditional design, an idealized design, and the optimized design.

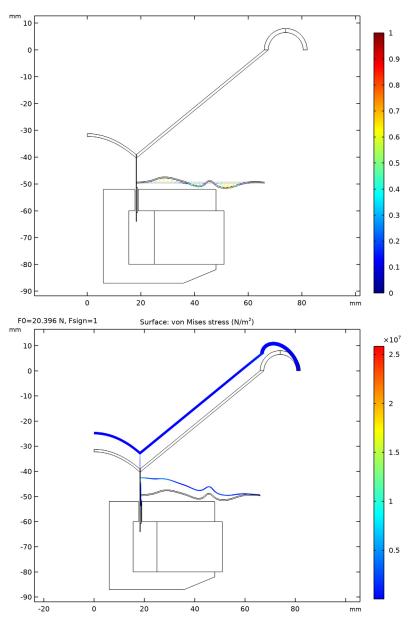


Figure 4: Relative normal displacement and von Mises stress of the optimized design.

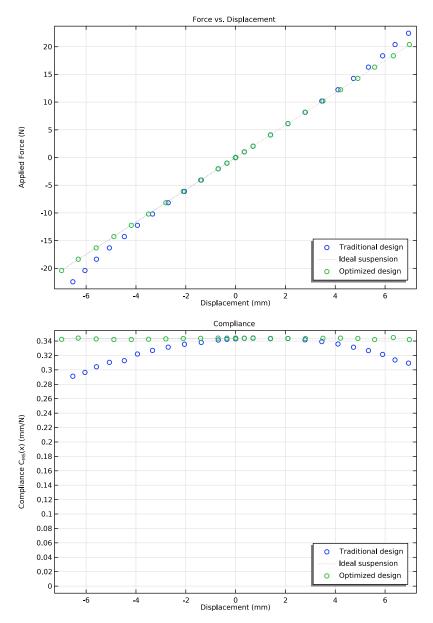


Figure 5: Force versus displacement and compliance curve of a traditional design (blue dots), an ideal suspension (gray line) and the optimized design (green dots). Note how the behavior of the traditional design deviates from the ideal behavior as the displacement increases.

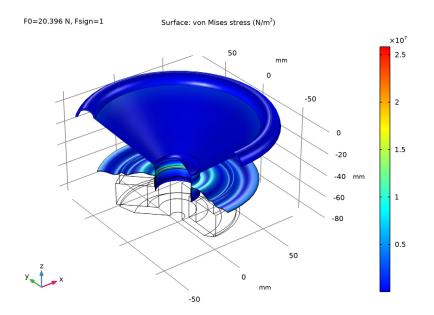


Figure 6: Revolved geometry of the optimized design.

The von Mises stress distribution of the revolved optimized design is shown in Figure 6.

This model demonstrates an efficient approach to optimize nonlinear structural problems where the performance through the different points of loading is also relevant to measure the overall performance. It also demonstrates how optimization can be used to obtain novel designs using traditional materials and manufacturing techniques, significantly improving the performance.

# Notes About the COMSOL Implementation

## USING A SINGLE COMPONENT AND PHYSICS TO REPRESENT TWO DESIGNS

The model uses different boundary conditions to represent two designs using a single geometry and the same physics. When using this type of approach, it is important to pay attention to the validity of the approach for the analysis of interest. In this case, for example, it is important to note that the approach is only valid for analyzing the static stiffness of the model, as the mass of the two designs will be included as moving masses if a frequency domain analysis was used instead. This model is also very light and runs fast,

but the computational implication of including part of the model that is not relevant for the analysis should be considered in larger models.

## USING GENERAL EXTRUSION

A general extrusion coupling operator is used to carry over the shape changes from the bottom boundary to the top boundary. A set of variables under the **Angle Variables** group compute the change in the normal direction of the bottom boundary to make sure that the thickness remains constant through the spider domain.

#### USING DOMAIN PROBES TO COMPUTE THE OPTIMIZATION OBJECTIVES

The objective of the optimization is based on a domain probe. The displacement and stiffness are computed through this domain probe and are used in the optimization objective.

**Application Library path:** Optimization\_Module/Shape\_Optimization/ loudspeaker\_spider\_optimization

# Modeling Instructions

From the File menu, choose New.

## N E W

In the New window, click 🙆 Model Wizard.

## MODEL WIZARD

- I In the Model Wizard window, click 🚈 2D Axisymmetric.
- 2 In the Select Physics tree, select Structural Mechanics>Solid Mechanics (solid).
- 3 Click Add.
- 4 Click 🔿 Study.
- 5 In the Select Study tree, select General Studies>Stationary.
- 6 Click **M** Done.

## GEOMETRY I

- I In the Model Builder window, under Component I (compl) click Geometry I.
- 2 In the Settings window for Geometry, locate the Units section.

- 3 From the Length unit list, choose mm.
- **4** In the **Geometry** toolbar, click **Insert Sequence**.
- 5 Browse to the model's Application Libraries folder and double-click the file loudspeaker\_spider\_optimization\_geom\_sequence.mph.
- 6 In the Geometry toolbar, click 🟢 Build All.

## **GLOBAL DEFINITIONS**

Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- 3 Click 📂 Load from File.
- **4** Browse to the model's Application Libraries folder and double-click the file loudspeaker\_spider\_optimization\_parameters.txt.

## DEFINITIONS

## **Optimization Objective**

- I In the Model Builder window, under Component I (compl) right-click Definitions and choose Variables.
- 2 In the Settings window for Variables, type Optimization Objective in the Label text field.
- 3 Locate the Variables section. In the table, enter the following settings:

Name	Expression	Unit	Description
obj_1	(susp_stiff- susp_stiff0)^2		Optimization objective

Angle Variables

- I In the Model Builder window, right-click Definitions and choose Variables.
- 2 In the Settings window for Variables, type Angle Variables in the Label text field.
- **3** Locate the Geometric Entity Selection section. From the Geometric entity level list, choose Boundary.
- 4 Select Boundary 32 only.

Name	Expression	Unit	Description
angle	atan2(nZ,nR)	rad	Normal angle
step	if(s<0.1,s/0.1, if(0.9 <s,(1-s) 0.1,1))<="" td=""><td></td><td>Angle smoothing step</td></s,(1-s)>		Angle smoothing step
nRApprox	cos(-pi/2*(1-step)+ angle*step)		R component of the normal
nZApprox	<pre>sin(-pi/2*(1-step)+ angle*step)</pre>		Z component of the normal

5 Locate the Variables section. In the table, enter the following settings:

## Domain Probe 1 (dom1)

- I In the Definitions toolbar, click probes and choose Domain Probe.
- 2 In the Settings window for Domain Probe, type susp\_disp in the Variable name text field.
- 3 Locate the Source Selection section. From the Selection list, choose Voice Coil.
- 4 Locate the Expression section. In the Expression text field, type w.
- **5** Select the **Description** check box.
- **6** In the associated text field, type **Displacement**.

## Domain Probe 2 (dom2)

- I In the Definitions toolbar, click probes and choose Domain Probe.
- 2 In the Settings window for Domain Probe, type susp\_stiff in the Variable name text field.
- 3 Locate the Source Selection section. From the Selection list, choose Voice Coil.
- 4 Locate the Expression section. In the Expression text field, type F0\*Fsign/w.
- 5 From the Table and plot unit list, choose N/mm.
- 6 Select the **Description** check box.
- 7 In the associated text field, type Suspension stiffness.

## General Extrusion 1 (genext1)

- I In the Definitions toolbar, click 🖉 Nonlocal Couplings and choose General Extrusion.
- 2 In the Settings window for General Extrusion, locate the Source Selection section.
- **3** From the **Geometric entity level** list, choose **Boundary**.
- 4 Select Boundary 32 only.
- 5 Locate the Destination Map section. In the r-expression text field, type Rg.

- 6 In the z-expression text field, type -49.7[mm].
- 7 Locate the Source section. From the Source frame list, choose Geometry (Rg, PHIg, Zg).

## Free Shape Domain 1

- I In the Definitions toolbar, click 😰 Optimization and choose Free Shape Domain.
- 2 In the Settings window for Free Shape Domain, locate the Domain Selection section.
- **3** From the Selection list, choose Shape Optimization.

## Free Shape Boundary I

- I In the Definitions toolbar, click 😥 Optimization and choose Free Shape Boundary.
- 2 Select Boundary 32 only.
- **3** In the Settings window for Free Shape Boundary, locate the Control Variable Settings section.
- **4** In the  $d_{\max}$  text field, type max\_disp.
- **5** Locate the **Filtering** section. In the  $R_{\min}$  text field, type max\_disp.

#### Prescribed Deformation 1

- I In the Definitions toolbar, click • • Deformed Geometry and choose Prescribed Deformation.
- **2** In the **Settings** window for **Prescribed Deformation**, locate the **Geometric Entity Selection** section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 Select Boundary 34 only.
- 5 Locate the Prescribed Deformation section. Specify the dx vector as

genext1(fsd1.dRg-nRApprox*0.4[mm])	
<pre>genext1(fsd1.dZg-nZApprox*0.4[mm])-0.4[mm]</pre>	z

6 Right-click Prescribed Deformation I and choose Browse Materials.

## MATERIAL BROWSER

- I In the Material Browser window, click III Import Material Library.
- 2 From the Application Libraries root, browse to the folder Acoustics\_Module/ Electroacoustic\_Transducers and double-click the file loudspeaker driver materials.mph.
- 3 Click M Done.

## MATERIALS

In the Home toolbar, click 📑 Windows and choose Add Material from Library.

## ADD MATERIAL

- I Go to the Add Material window.
- 2 In the tree, select loudspeaker driver materials>Composite.
- 3 Click Add to Component in the window toolbar.

## MATERIALS

## Composite (mat1)

- I In the Settings window for Material, locate the Geometric Entity Selection section.
- 2 From the Selection list, choose Composite.

## ADD MATERIAL

- I Go to the Add Material window.
- 2 In the tree, select loudspeaker driver materials>Cloth.
- 3 Click Add to Component in the window toolbar.

## MATERIALS

Cloth (mat2)

- I In the Settings window for Material, locate the Geometric Entity Selection section.
- 2 From the Selection list, choose Cloth.

## ADD MATERIAL

- I Go to the Add Material window.
- 2 In the tree, select loudspeaker driver materials>Foam.
- 3 Click Add to Component in the window toolbar.

## MATERIALS

Foam (mat3)

- I In the Settings window for Material, locate the Geometric Entity Selection section.
- 2 From the Selection list, choose Foam.

## ADD MATERIAL

I Go to the Add Material window.

- 2 In the tree, select loudspeaker driver materials>Coil.
- 3 Click Add to Component in the window toolbar.

## MATERIALS

## Coil (mat4)

- I In the Settings window for Material, locate the Geometric Entity Selection section.
- 2 From the Selection list, choose Voice Coil.

## ADD MATERIAL

- I Go to the Add Material window.
- 2 In the tree, select loudspeaker driver materials>Glass Fiber.
- 3 Click Add to Component in the window toolbar.
- 4 In the Home toolbar, click 🙀 Add Material to close the Add Material window.

## MATERIALS

Glass Fiber (mat5)

- I In the Settings window for Material, locate the Geometric Entity Selection section.
- 2 From the Selection list, choose Former.

## SOLID MECHANICS (SOLID)

- I In the Model Builder window, under Component I (compl) click Solid Mechanics (solid).
- 2 In the Settings window for Solid Mechanics, locate the Domain Selection section.
- 3 From the Selection list, choose Structural Domains.

Body Load I

- I In the Physics toolbar, click **Domains** and choose **Body Load**.
- 2 In the Settings window for Body Load, locate the Domain Selection section.
- 3 From the Selection list, choose Voice Coil.
- 4 Locate the Force section. From the Load type list, choose Total force.
- **5** Specify the **F**<sub>tot</sub> vector as

0	r
F0*Fsign	z

Fixed Constraint 1

I In the Physics toolbar, click — Boundaries and choose Fixed Constraint.

- 2 In the Settings window for Fixed Constraint, locate the Boundary Selection section.
- 3 From the Selection list, choose Fixed Boundaries I.

## Fixed Constraint 2

- I In the Physics toolbar, click Boundaries and choose Fixed Constraint.
- 2 In the Settings window for Fixed Constraint, locate the Boundary Selection section.
- 3 From the Selection list, choose Fixed Boundaries 2.

## MESH I

Mapped I

- I In the Mesh toolbar, click Mapped.
- 2 In the Settings window for Mapped, locate the Domain Selection section.
- 3 From the Geometric entity level list, choose Domain.
- 4 From the Selection list, choose Structural Domains.

# Size I

- I Right-click Mapped I and choose Size.
- 2 In the Settings window for Size, locate the Element Size section.
- 3 Click the **Custom** button.
- 4 Locate the Element Size Parameters section. Select the Maximum element size check box.
- 5 In the associated text field, type mesh\_size.

#### Distribution I

- I In the Model Builder window, right-click Mapped I and choose Distribution.
- 2 In the Settings window for Distribution, locate the Boundary Selection section.
- 3 From the Selection list, choose Fixed Boundaries I.
- 4 Locate the Distribution section. In the Number of elements text field, type 2.

#### Distribution 2

- I Right-click Mapped I and choose Distribution.
- 2 In the Settings window for Distribution, locate the Boundary Selection section.
- 3 From the Selection list, choose Fixed Boundaries 2.
- 4 Locate the Distribution section. In the Number of elements text field, type 2.
- 5 Click 📗 Build All.

Free Triangular 1

- I In the Mesh toolbar, click K Free Triangular.
- 2 In the Settings window for Free Triangular, click 📗 Build All.

## STUDY I - TRADITIONAL DESIGN

- I In the Model Builder window, click Study I.
- 2 In the Settings window for Study, type Study 1 Traditional Design in the Label text field.

Step 1: Stationary

- I In the Model Builder window, under Study I Traditional Design click Step I: Stationary.
- 2 In the Settings window for Stationary, locate the Study Settings section.
- **3** Select the **Include geometric nonlinearity** check box.
- 4 Click to expand the Results While Solving section. From the Probes list, choose None.
- 5 Locate the Physics and Variables Selection section. Select the Modify model configuration for study step check box.
- 6 In the Physics and variables selection tree, select Component I (compl)> Solid Mechanics (solid), Controls spatial frame>Fixed Constraint 2.
- 7 Click 🖉 Disable.
- 8 Click to expand the Study Extensions section. Select the Auxiliary sweep check box.
- **9** From the Sweep type list, choose All combinations.
- 10 Click + Add.

**II** In the table, enter the following settings:

Parameter name	Parameter value list	Parameter unit
Fsign (Force sign)	-1 1	

**I2** Click + Add.

**I3** In the table, enter the following settings:

Parameter name	Parameter value list	Parameter unit
F0 (Force parameter for the sweep)	Fmax/1000 Fmax/20 range(Fmax/10,Fmax/10, Fmax*1.15)	Ν

**I4** In the **Home** toolbar, click **= Compute**.

## RESULTS

Study I - Traditional Design/Solution I (soll)

In the Model Builder window, expand the Results>Datasets node, then click Study I - Traditional Design/Solution I (sol1).

#### Selection

- I In the Results toolbar, click 🖣 Attributes and choose Selection.
- 2 In the Settings window for Selection, locate the Geometric Entity Selection section.
- 3 From the Geometric entity level list, choose Domain.
- **4** Select Domains 1–11 and 13–18 only.

## Applied Loads (solid), Shape Optimization

- I In the Model Builder window, under Results, Ctrl-click to select Applied Loads (solid) and Shape Optimization.
- 2 Right-click and choose Delete.

#### Stress (solid), Stress, 3D (solid)

- In the Model Builder window, under Results, Ctrl-click to select Stress (solid) and Stress, 3D (solid).
- 2 Right-click and choose Group.

#### Traditional Design Results

In the **Settings** window for **Group**, type Traditional Design Results in the **Label** text field.

## ROOT

In the Home toolbar, click 📑 Windows and choose Add Study.

## ADD STUDY

- I Go to the Add Study window.
- 2 Find the Studies subsection. In the Select Study tree, select Empty Study.
- 3 Click Add Study in the window toolbar.
- 4 In the Home toolbar, click Add Study to close the Add Study window.

## STUDY 2

## Shape Optimization

I In the Study toolbar, click of Optimization and choose Shape Optimization.

- 2 In the Settings window for Shape Optimization, locate the Objective Function section.
- **3** From the Solution list, choose Maximum of objectives.
- 4 Locate the Optimization Solver section. Clear the Move limits check box.

**5** Locate the **Objective Function** section. In the table, enter the following settings:

Expression	Description
log10(comp1.obj_1)	

6 Locate the Output While Solving section. From the Probes list, choose None.

## STUDY I - TRADITIONAL DESIGN

#### Step 1: Stationary

In the Model Builder window, under Study I - Traditional Design right-click Step I: Stationary and choose Copy.

## STUDY 2

In the Model Builder window, right-click Study 2 and choose Paste Stationary.

## Step 1: Stationary

- I In the Settings window for Stationary, locate the Physics and Variables Selection section.
- 2 In the Physics and variables selection tree, select Component I (compl)> Solid Mechanics (solid), Controls spatial frame>Fixed Constraint I.
- 3 Click 💋 Disable.
- 4 In the Physics and variables selection tree, select Component I (compl)> Solid Mechanics (solid), Controls spatial frame>Fixed Constraint 2.
- 5 Click 🔘 Enable.
- 6 Locate the Study Extensions section. In the table, enter the following settings:

Parameter name	Parameter value list	Parameter unit
F0 (Force parameter for the sweep)	Fmax/1000 Fmax/20 range(Fmax/10,Fmax/10, Fmax)	Ν

- 7 In the Model Builder window, click Study 2.
- 8 In the Settings window for Study, type Study 2 Optimization in the Label text field.
- **9** In the Study toolbar, click  $t_{=0}^{\cup}$  Get Initial Value.

## RESULTS

Applied Loads (solid)

In the Model Builder window, under Results right-click Applied Loads (solid) and choose Delete.

Study 2 - Optimization/Solution 2 (sol2)

In the Model Builder window, click Study 2 - Optimization/Solution 2 (sol2).

Selection

- I In the Results toolbar, click 🖣 Attributes and choose Selection.
- 2 In the Settings window for Selection, locate the Geometric Entity Selection section.
- 3 From the Geometric entity level list, choose Domain.
- **4** Select Domains 1–12 and 14–18 only.

Arrow Line 1

- I In the Model Builder window, expand the Results>Shape Optimization node, then click Arrow Line I.
- 2 In the Settings window for Arrow Line, locate the Arrow Positioning section.
- 3 From the Placement list, choose Gauss points.

Shape Optimization, Stress (solid) 1, Stress, 3D (solid) 1

- In the Model Builder window, under Results, Ctrl-click to select Stress (solid) 1, Stress,
   3D (solid) 1, and Shape Optimization.
- 2 Right-click and choose Group.

**Optimization Results** 

In the Settings window for Group, type Optimization Results in the Label text field.

## STUDY 2 - OPTIMIZATION

Shape Optimization

- I In the Model Builder window, under Study 2 Optimization click Shape Optimization.
- 2 In the Settings window for Shape Optimization, locate the Output While Solving section.
- **3** Select the **Plot** check box.
- 4 From the Plot group list, choose Shape Optimization.
- **5** In the **Home** toolbar, click **= Compute**.

## RESULTS

## Force vs. Displacement

- I In the Home toolbar, click 🔎 Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Force vs. Displacement in the Label text field.
- 3 Locate the Plot Settings section. Select the x-axis label check box.
- **4** In the associated text field, type Displacement (mm).
- 5 Select the y-axis label check box.
- 6 In the associated text field, type Applied Force (N).
- 7 Click to expand the Title section. From the Title type list, choose Label.
- 8 Locate the Legend section. From the Position list, choose Lower right.

## Global I

- I Right-click Force vs. Displacement and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
F0*Fsign		

- 4 Locate the x-Axis Data section. From the Axis source data list, choose All solutions.
- 5 From the Parameter list, choose Expression.
- **6** In the **Expression** text field, type susp\_disp.
- 7 Click to expand the Legends section. From the Legends list, choose Manual.
- 8 In the table, enter the following settings:

#### Legends

## Traditional design

- 9 Click to expand the Coloring and Style section. Find the Line style subsection. From the Line list, choose None.
- 10 Find the Line markers subsection. From the Marker list, choose Circle.
- II From the **Positioning** list, choose **In data points**.
- **12** In the Force vs. Displacement toolbar, click **I** Plot.

Global 2

- I Right-click Global I and choose Duplicate.
- 2 In the Settings window for Global, locate the Data section.
- 3 From the Dataset list, choose Study 2 Optimization/Solution 2 (sol2).
- 4 Locate the x-Axis Data section. From the Axis source data list, choose All solutions.
- 5 Locate the y-Axis Data section. In the table, enter the following settings:

Expression	Unit	Description
<pre>susp_disp/susp_comp0</pre>	N	

- 6 Locate the Coloring and Style section. Find the Line style subsection. From the Line list, choose Dotted.
- 7 From the Color list, choose Gray.
- 8 Find the Line markers subsection. From the Marker list, choose None.
- 9 Locate the Legends section. In the table, enter the following settings:

#### Legends

Ideal suspension

Global 3

- I In the Model Builder window, under Results>Force vs. Displacement right-click Global I and choose Duplicate.
- 2 In the Settings window for Global, locate the Data section.
- 3 From the Dataset list, choose Study 2 Optimization/Solution 2 (sol2).
- 4 Locate the x-Axis Data section. From the Axis source data list, choose All solutions.
- 5 Locate the Legends section. In the table, enter the following settings:

#### Legends

Optimized design

6 In the Force vs. Displacement toolbar, click 🗿 Plot.

The image should look like Figure 5.

#### Compliance

- I In the Model Builder window, right-click Force vs. Displacement and choose Duplicate.
- 2 In the Settings window for ID Plot Group, type Compliance in the Label text field.

3 Locate the Plot Settings section. In the y-axis label text field, type Compliance C<sub> MS</sub>(x) (mm/N).

Global I

- I In the Model Builder window, expand the Compliance node, then click Global I.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
1/susp_stiff	mm / N	

Global 2

I In the Model Builder window, click Global 2.

2 In the Settings window for Global, locate the y-Axis Data section.

**3** In the table, enter the following settings:

Expression	Unit	Description
1/susp_stiff0	mm / N	

Global 3

I In the Model Builder window, click Global 3.

2 In the Settings window for Global, locate the y-Axis Data section.

**3** In the table, enter the following settings:

Expression	Unit	Description
1/susp_stiff	mm / N	

**4** In the **Compliance** toolbar, click **I Plot**.

Annotation I

- I In the Model Builder window, right-click Compliance and choose Annotation.
- 2 In the Settings window for Annotation, locate the Coloring and Style section.
- **3** Clear the **Show point** check box.
- **4** In the **Compliance** toolbar, click **O Plot**.

The image should look like Figure 5.

Stress, 3D (solid)

In the Model Builder window, expand the Results>Traditional Design Results>Stress, 3D (solid) node.

## Deformation

- In the Model Builder window, expand the Results>Traditional Design Results>Stress, 3D (solid)>Surface I node.
- 2 Right-click **Deformation** and choose **Delete**.

#### Stress, 3D (solid) 1

In the Model Builder window, expand the Results>Optimization Results>Stress, 3D (solid) I node.

Deformation

- In the Model Builder window, expand the Results>Optimization Results>Stress,
   3D (solid) I>Surface I node.
- 2 Right-click **Deformation** and choose **Delete**.
- 3 In the Stress, 3D (solid) I toolbar, click 💽 Plot.

The image should look like Figure 6.

# Geometry Modeling Instructions

If you want to create the geometry yourself, follow these steps.

From the File menu, choose New.

## NEW

In the New window, click Slank Model.

## ADD COMPONENT

In the Home toolbar, click 🛞 Add Component and choose 2D Axisymmetric.

#### GEOMETRY I

Circle 1 (c1)

- I In the **Geometry** toolbar, click  $\cdot$  **Circle**.
- 2 In the Settings window for Circle, locate the Size and Shape section.
- 3 In the Radius text field, type 130[mm].
- 4 Click to expand the Layers section. In the table, enter the following settings:

Layer name	Thickness (m)
Layer 1	15[mm]

Circle 2 (c2)

I In the **Geometry** toolbar, click  $\bigcirc$  **Circle**.

2 In the Settings window for Circle, locate the Size and Shape section.

3 In the Radius text field, type 8[mm].

4 In the Sector angle text field, type 180.

**5** Locate the **Position** section. In the **r** text field, type 74[mm].

6 Locate the Layers section. In the table, enter the following settings:

Layer name	Thickness (m)
Layer 1	1.5[mm]

Delete Entities I (dell)

I In the Model Builder window, right-click Geometry I and choose Delete Entities.

- 2 On the object c2, select Boundaries 2–4 only.
- **3** In the **Settings** window for **Delete Entities**, locate the **Selections of Resulting Entities** section.
- 4 Select the **Resulting objects selection** check box.

Rectangle 1 (r1)

I In the Geometry toolbar, click 📃 Rectangle.

- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- 3 In the Width text field, type 70[mm].
- 4 In the **Height** text field, type 1[mm].
- **5** Locate the **Position** section. In the **r** text field, type **80.5**[mm].
- 6 In the z text field, type -1[mm].

#### Difference I (dif1)

- I In the Geometry toolbar, click i Booleans and Partitions and choose Difference.
- 2 Select the object **cl** only.
- 3 In the Settings window for Difference, locate the Difference section.
- 4 Find the Objects to subtract subsection. Select the 🔲 Activate Selection toggle button.
- **5** Select the object **r1** only.

#### Rectangle 2 (r2)

- I In the **Geometry** toolbar, click **Rectangle**.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.

- **3** In the **Width** text field, type 42[mm].
- 4 In the **Height** text field, type 35[mm].
- **5** Locate the **Position** section. In the **r** text field, type 6[mm].
- 6 In the z text field, type -87[mm].

## Rectangle 3 (r3)

- I In the Geometry toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- 3 In the Width text field, type 35.5[mm].
- 4 In the **Height** text field, type 20[mm].
- 5 Locate the **Position** section. In the r text field, type 15.5[mm].
- 6 In the z text field, type -80[mm].

## Rectangle 4 (r4)

- I In the **Geometry** toolbar, click **Rectangle**.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- 3 In the Width text field, type 1.2[mm].
- 4 In the **Height** text field, type 8[mm].
- **5** Locate the **Position** section. In the **r** text field, type **17.8**[mm].
- 6 In the z text field, type -60[mm].

#### Rectangle 5 (r5)

- I In the Geometry toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- **3** In the **Width** text field, type 26[mm].
- 4 In the **Height** text field, type 20[mm].
- **5** Locate the **Position** section. In the **r** text field, type **25**[mm].
- 6 In the z text field, type -80[mm].

## Polygon I (poll)

- I In the Geometry toolbar, click / Polygon.
- 2 In the Settings window for Polygon, locate the Coordinates section.
- **3** From the **Data source** list, choose **Vectors**.
- **4** In the **r** text field, type 48[mm] 36[mm] 36[mm] 48[mm].
- **5** In the **z** text field, type -82[mm] -87[mm] -87[mm].

## Difference 2 (dif2)

- I In the Geometry toolbar, click Pooleans and Partitions and choose Difference.
- 2 Select the object r2 only.
- 3 In the Settings window for Difference, locate the Difference section.
- 4 Find the Objects to subtract subsection. Select the 🔲 Activate Selection toggle button.
- 5 Select the objects poll, r3, and r4 only.
- **6** Locate the **Selections of Resulting Entities** section. Select the **Resulting objects selection** check box.

Rectangle 6 (r6)

- I In the **Geometry** toolbar, click **Rectangle**.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- **3** In the **Width** text field, type **0.2**[mm].
- 4 In the **Height** text field, type 25[mm].
- 5 Locate the Position section. In the r text field, type 18.2[mm].
- 6 In the z text field, type -64[mm].
- 7 Click to expand the Layers section. In the table, enter the following settings:

Layer name	Thickness (m)
Layer 1	1.26[mm]
Layer 2	3.84[mm]
Layer 3	0.4[mm]

- 8 Clear the Layers on bottom check box.
- 9 Select the Layers on top check box.

Rectangle 7 (r7)

- I In the Geometry toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- **3** In the **Width** text field, type **0.6**[mm].
- 4 In the **Height** text field, type 9.4[mm].
- 5 Locate the Position section. In the r text field, type 18.2[mm].
- 6 In the z text field, type -60.7[mm].

Rectangle 8 (r8)

I In the Geometry toolbar, click Rectangle.

- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- 3 In the Width text field, type 4.6[mm].
- 4 In the **Height** text field, type 0.4[mm].
- 5 Locate the **Position** section. In the **r** text field, type 18.4[mm].
- 6 In the z text field, type -44.5[mm].

Rectangle 9 (r9)

- I In the Geometry toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- **3** In the **Width** text field, type 7[mm].
- 4 In the **Height** text field, type 0.4[mm].
- **5** Locate the **Position** section. In the **r** text field, type **59**[mm].
- 6 In the z text field, type -44.5[mm].

Polygon 2 (pol2)

- I In the Geometry toolbar, click / Polygon.
- 2 In the Settings window for Polygon, locate the Coordinates section.
- **3** From the **Data source** list, choose **Vectors**.
- 4 In the r text field, type 23[mm] 26[mm] 26[mm] 32[mm] 32[mm] 38[mm] 38[mm] 44[mm] 44[mm] 50[mm] 50[mm] 56[mm] 56[mm] 59[mm] 59[mm] 59[mm] 59[mm] 56[mm] 50[mm] 50[mm] 44[mm] 44[mm] 38[mm] 38[mm] 32[mm] 32[mm] 26[mm] 26[mm] 23[mm] 23[mm] 23[mm].
- 5 In the z text field, type -44.1[mm] -42.1[mm] -42.1[mm] -46.1[mm] -46.1[mm] -42.1[mm] -42.1[mm] -42.1[mm] -42.1[mm] -42.1[mm] -42.1[mm] -42.1[mm] -46.1[mm] -46.1[mm] -44.1[mm] -44.1[mm] -44.5[mm] -44.5[mm] -46.5[mm] -42.5[mm] -42.5[mm] -42.5[mm] -42.5[mm] -42.5[mm] -42.5[mm] -42.5[mm] -42.5[mm] -42.5[mm] -44.5[mm] -44.5[mm] -44.1.

Union I (uniI)

- I In the Geometry toolbar, click Booleans and Partitions and choose Union.
- 2 Select the objects pol2, r8, and r9 only.
- 3 In the Settings window for Union, locate the Selections of Resulting Entities section.
- **4** Select the **Resulting objects selection** check box.
- 5 Locate the Union section. Clear the Keep interior boundaries check box.

Polygon 3 (pol3)

- I In the Geometry toolbar, click / Polygon.
- 2 In the Settings window for Polygon, locate the Coordinates section.
- 3 From the Data source list, choose Vectors.
- **4** In the r text field, type 18.4[mm] 66[mm] 66[mm] 67.5[mm] 67.5[mm] 18.4[mm] 18.4[mm].
- **5** In the **z** text field, type -39[mm] 0 0 0 0 -40.26[mm] -40.26[mm] -39[mm].

Quadratic Bézier I (qb1)

- I In the Geometry toolbar, click 🗱 More Primitives and choose Quadratic Bézier.
- 2 In the Settings window for Quadratic Bézier, locate the Control Points section.
- **3** In row **I**, set **r** to -18.2[mm].
- 4 In row 3, set r to 18.2[mm].
- **5** In row **I**, set **z** to -39[mm].
- 6 In row 2, set z to -23.5[mm].
- 7 In row 3, set z to -39[mm].
- 8 Locate the Weights section. In the 2 text field, type 1.

## Line Segment I (Is I)

- I In the Geometry toolbar, click 😕 More Primitives and choose Line Segment.
- 2 In the Settings window for Line Segment, locate the Starting Point section.
- **3** From the **Specify** list, choose **Coordinates**.
- **4** Locate the **Endpoint** section. From the **Specify** list, choose **Coordinates**.
- **5** Locate the **Starting Point** section. In the **r** text field, type 18.2[mm].
- 6 Locate the **Endpoint** section. In the **r** text field, type 18.2[mm].
- 7 Locate the Starting Point section. In the z text field, type -39[mm].
- 8 Locate the Endpoint section. In the z text field, type -40.26[mm].

## Quadratic Bézier 2 (qb2)

- I In the Geometry toolbar, click 🗱 More Primitives and choose Quadratic Bézier.
- 2 In the Settings window for Quadratic Bézier, locate the Control Points section.
- **3** In row **I**, set **r** to **18.2**[mm].
- 4 In row 3, set r to -18.2[mm].
- **5** In row **I**, set **z** to -40.26[mm].

- 6 In row 2, set z to -24.26[mm].
- 7 In row 3, set z to -40.26[mm].
- 8 Locate the Weights section. In the 2 text field, type 1.

Line Segment 2 (Is2)

- I In the Geometry toolbar, click 😕 More Primitives and choose Line Segment.
- 2 In the Settings window for Line Segment, locate the Starting Point section.
- **3** From the **Specify** list, choose **Coordinates**.
- 4 Locate the Endpoint section. From the Specify list, choose Coordinates.
- **5** Locate the **Starting Point** section. In the **r** text field, type -18.2[mm].
- 6 Locate the **Endpoint** section. In the **r** text field, type -18.2[mm].
- 7 Locate the Starting Point section. In the z text field, type -40.26[mm].
- 8 Locate the Endpoint section. In the z text field, type -39[mm].

Convert to Solid 1 (csol1)

- I In the Geometry toolbar, click 🕅 Conversions and choose Convert to Solid.
- 2 Select the objects Is1, Is2, qb1, and qb2 only.

Line Segment 3 (Is3)

- I In the Geometry toolbar, click 🗱 More Primitives and choose Line Segment.
- 2 In the Settings window for Line Segment, locate the Starting Point section.
- 3 From the Specify list, choose Coordinates.
- 4 In the z text field, type -52[mm].
- 5 Locate the Endpoint section. From the Specify list, choose Coordinates.
- 6 In the r text field, type sqrt((115[mm])<sup>2</sup> (52[mm])<sup>2</sup>).
- 7 In the z text field, type -52[mm].
- 8 Locate the Selections of Resulting Entities section. Select the Resulting objects selection check box.

## Union 2 (uni2)

- I In the Geometry toolbar, click P Booleans and Partitions and choose Union.
- 2 Click in the Graphics window and then press Ctrl+A to select all objects.

#### Delete Entities 2 (del2)

- I Right-click Geometry I and choose Delete Entities.
- 2 On the object uni2, select Boundaries 13, 19, 33, and 45 only.

Fillet I (fill)

- I In the **Geometry** toolbar, click **Fillet**.
- 2 On the object del2, select Points 14, 15, 35, and 36 only.
- 3 In the Settings window for Fillet, locate the Radius section.
- 4 In the Radius text field, type 0.2[mm].

#### Form Union (fin)

- I In the Model Builder window, click Form Union (fin).
- 2 In the Settings window for Form Union/Assembly, locate the Form Union/Assembly section.
- 3 From the Repair tolerance list, choose Relative.

## Ignore Vertices 1 (igv1)

- I In the Geometry toolbar, click 🏷 Virtual Operations and choose Ignore Vertices.
- 2 On the object fin, select Points 18, 26, and 33 only.
- 3 In the Geometry toolbar, click 🟢 Build All.