

Assembly with a Hinge Joint

In mechanical assemblies, parts are sometimes connected so that they are free to move relative to each other with one or multiple degrees of freedom. Examples of such connections are ball joints, hinges, and different types of bearings. If the details of the connection are not the subjects of the analysis, it is possible to model the connection using joint nodes in the Multibody Dynamics Module.

This example illustrates how to model a barrel hinge connecting two solid objects in an assembly. Two different versions are studied. In one of them, both parts are flexible. In the other version, one of the parts is considered as rigid.

Model Definition

Figure 1 shows the model geometry.

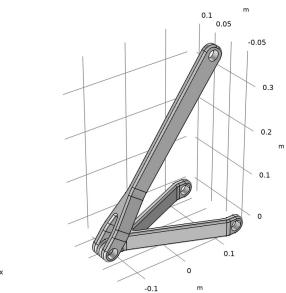


Figure 1: Model geometry.

Two parts of the assembly are connected through a barrel hinge that allows relative rotation as well as sliding along the axis of the pin hole. All the other degrees of freedom are shared between the two parts.

One hole of the forked bottom part is modeled as a hinge joint and other hole is modeled as a cylindrical joint allowing the axial motion.

The pin hole of the top part is constrained in the x direction so that it can slide in the yz plane.

A force of 1 kN is applied in z direction at a 10 cm distance in the negative y direction from the center of the upper pin hole. The offset of the load introduces both tension and bending of the member.

In a second study, the upper part is considered as rigid, and it only transmits the force through the hinge.

Results and Discussion

The default displacement plot for the flexible model is shown in Figure 2.

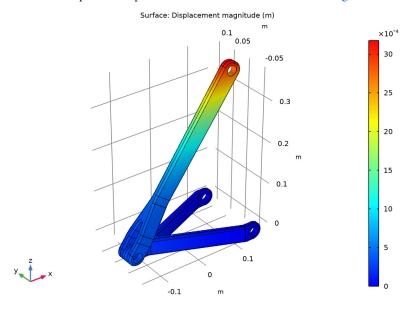


Figure 2: Displacement of the flexible structure.

In Figure 3 you can see the stress distribution.

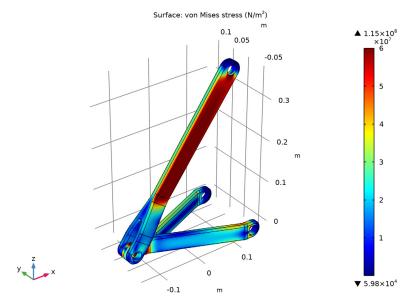


Figure 3: Equivalent stress in the flexible structure

In a model that consists of a mix of rigid and flexible parts, stresses can only be displayed in the flexible parts. This is shown in Figure 4.

If you compare Figure 3 and Figure 4, you can see that the stress distribution in the lower part is essentially unaffected by the fact that the load is transmitted through a rigid body.

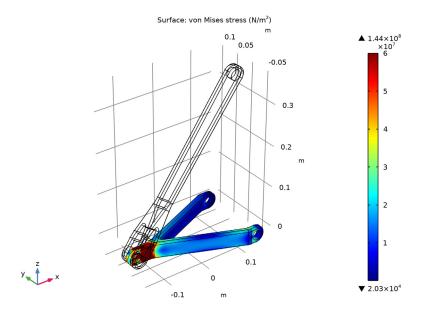


Figure 4: Equivalent stress in the flexible part when one component is rigid.

Notes About the COMSOL Implementation

When the flexibility of a component can be neglected and its stress distribution is not of interest, it is efficient to treat such a component as a rigid domain. The rigid domain is a domain that has a material model with only one material parameter, the density.

A Joint node can establish a connection directly between Rigid Domain nodes however **Attachment** nodes are needed, defining the connection boundaries, for flexible elements.

Application Library path: Multibody_Dynamics_Module/Tutorials/ hinge joint assembly

Modeling Instructions

From the File menu, choose New.

NEW

In the New window, click Model Wizard.

MODEL WIZARD

- I In the Model Wizard window, click 1 3D.
- 2 In the Select Physics tree, select Structural Mechanics>Multibody Dynamics (mbd).
- 3 Click Add.
- 4 Click Study.
- 5 In the Select Study tree, select General Studies>Stationary.
- 6 Click **Done**.

GLOBAL DEFINITIONS

Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- **3** In the table, enter the following settings:

Name	Expression	Value	Description
F	1e3[N]	1000 N	Applied load

GEOMETRY I

Import I (impl)

- I In the **Home** toolbar, click **Import**.
- 2 In the Settings window for Import, locate the Import section.
- 3 Click Browse.
- **4** Browse to the model's Application Libraries folder and double-click the file hinge assembly.mphbin.
- 5 Click Import.

Form Union (fin)

- I In the Model Builder window, under Component I (compl)>Geometry I click Form Union (fin).
- 2 In the Settings window for Form Union/Assembly, locate the Form Union/Assembly section.
- 3 From the Action list, choose Form an assembly.
- 4 Clear the Create pairs check box.

5 Click Pauld Selected.

ADD MATERIAL

- I In the Home toolbar, click **Add Material** to open the Add Material window.
- 2 Go to the Add Material window.
- 3 In the tree, select Built-in>Structural steel.
- 4 Click Add to Component in the window toolbar.
- 5 In the Home toolbar, click **‡ Add Material** to close the **Add Material** window.

MULTIBODY DYNAMICS (MBD)

Fixed Constraint I

- I In the Model Builder window, under Component I (compl) right-click Multibody Dynamics (mbd) and choose Fixed Constraint.
- 2 Select Boundaries 133–136 only.

Use **Rigid Connector** node to constrain the motion and apply load.

Rigid Connector 1

- I In the Physics toolbar, click **Boundaries** and choose **Rigid Connector**.
- **2** Select Boundaries 67 and 68 only.
- 3 In the Settings window for Rigid Connector, locate the Prescribed Displacement at Center of Rotation section.
- 4 Select the Prescribed in x direction check box.

Applied Force 1

- I In the Physics toolbar, click 🕞 Attributes and choose Applied Force.
- 2 In the Settings window for Applied Force, locate the Location section.
- **3** Select the **Offset** check box.
- **4** Specify the $\mathbf{X}_{\text{offset}}$ vector as

0	x
-0.1	у
0	z

The center of rotation for a rigid connector is available in the variables xcx_tag, xcy_tag, and xcz_tag. The default position is the center of gravity of the attached boundaries, which in this case will be the center of the hole.

5 Locate the **Applied Force** section. Specify the \mathbf{F} vector as

0	x
0	у
F	z

Attachment I

- I In the Physics toolbar, click **Boundaries** and choose **Attachment**.
- 2 Select Boundaries 16 and 17 only.
- 3 In the Settings window for Attachment, locate the Connection Type section.
- 4 From the list, choose Flexible.

Attachment 2

- I In the Physics toolbar, click **Boundaries** and choose **Attachment**.
- 2 Select Boundaries 18 and 19 only.
- 3 In the Settings window for Attachment, locate the Connection Type section.
- 4 From the list, choose Flexible.

Attachment 3

- I In the Physics toolbar, click **Boundaries** and choose **Attachment**.
- **2** Select Boundaries 75 and 76 only.
- 3 In the Settings window for Attachment, locate the Connection Type section.
- **4** From the list, choose **Flexible**.

Hinge Joint 1

- I In the Physics toolbar, click Global and choose Hinge Joint.
- 2 In the Settings window for Hinge Joint, locate the Attachment Selection section.
- 3 From the Source list, choose Attachment 1.
- **4** From the **Destination** list, choose **Attachment 3**.
- **5** Locate the **Axis of Joint** section. Specify the e_0 vector as

0	x
1	у
0	z

Cylindrical Joint 1

I In the Physics toolbar, click Global and choose Cylindrical Joint.

- 2 In the Settings window for Cylindrical Joint, locate the Attachment Selection section.
- **3** From the **Source** list, choose **Attachment 2**.
- 4 From the **Destination** list, choose **Attachment 3**.
- **5** Locate the **Axis of Joint** section. Specify the e_0 vector as

0	x
1	у
0	z

If you want accurate stress results, then quadratic shape functions should be used for the displacements. The default value in the Multibody Dynamics interface is linear shape functions.

- 6 In the Model Builder window, click Multibody Dynamics (mbd).
- 7 In the Settings window for Multibody Dynamics, click to expand the Discretization section.
- 8 From the Displacement field list, choose Quadratic Lagrange.

MESH I

Free Tetrahedral I

In the Mesh toolbar, click A Free Tetrahedral.

Size 1

- I Right-click Free Tetrahedral I and choose Size.
- 2 In the Settings window for Size, locate the Geometric Entity Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 Select Boundaries 41, 42, and 53 only.
- **5** Locate the **Element Size** section. Click the **Custom** button.
- 6 Locate the Element Size Parameters section. Select the Maximum element size check box.
- 7 In the associated text field, type 0.002.

Size

- I In the Model Builder window, click Size.
- 2 In the Settings window for Size, locate the Element Size section.
- 3 From the Predefined list, choose Fine.
- 4 Click Build All.

STUDY I

In the **Home** toolbar, click **Compute**.

RESULTS

Displacement (mbd)

I In the Displacement (mbd) toolbar, click **Plot**.

The default plot shows the displacement of the assembly. Compare with Figure 2.

Reproduce the stress plot shown in Figure 3 with the following steps.

Stress

- I In the Home toolbar, click **Add Plot Group** and choose **3D Plot Group**.
- 2 In the Settings window for 3D Plot Group, type Stress in the Label text field.

Surface 1

- I Right-click Stress and choose Surface.
- 2 In the Settings window for Surface, click Replace Expression in the upper-right corner of the Expression section. From the menu, choose Component I (compl)> Multibody Dynamics>Stress>mbd.mises - von Mises stress - N/m2.
- 3 In the Stress toolbar, click Plot.
- 4 Click to expand the Range section. Select the Manual color range check box.
- 5 In the Maximum text field, type 6E7.

Stress

- I In the Model Builder window, click Stress.
- 2 In the Settings window for 3D Plot Group, locate the Color Legend section.
- 3 Select the Show maximum and minimum values check box.
- 4 In the Stress toolbar, click Plot.

Make the upper lever rigid and analyze that configuration in a new study.

MULTIBODY DYNAMICS (MBD)

Rigid Domain I

- I In the Physics toolbar, click **Domains** and choose Rigid Domain.
- 2 Select Domain 1 only.

Use Rigid Domain subnodes to constrain the motion and apply load.

Prescribed Displacement/Rotation I

- I In the Physics toolbar, click 🕞 Attributes and choose Prescribed Displacement/Rotation.
- 2 In the Settings window for Prescribed Displacement/Rotation, locate the Prescribed Displacement at Center of Rotation section.
- 3 Select the Prescribed in x direction check box.
- 4 Locate the Center of Rotation section. From the list, choose Centroid of selected entities.

Center of Rotation: Boundary I

- I In the Model Builder window, expand the Prescribed Displacement/Rotation I node, then click Center of Rotation: Boundary 1.
- **2** Select Boundaries 67 and 68 only.

Rigid Domain 1

In the Model Builder window, click Rigid Domain 1.

Applied Force 1

- I In the Physics toolbar, click 🕞 Attributes and choose Applied Force. Similar to the previous case, point of load application can be written like as follows:
- 2 In the Settings window for Applied Force, locate the Location section.
- **3** From the list, choose **User defined**.
- **4** Specify the \mathbf{X}_p vector as

mbd.rd1.pdr1.xcx	x
mbd.rd1.pdr1.xcy-0.1	у
mbd.rd1.pdr1.xcz	z

5 Locate the **Applied Force** section. Specify the \mathbf{F} vector as

0	x
0	у
F	z

Add an extra **Hinge Joint** node, so that it is easy to rerun the original version of the model. Normally it would have been easier just to utilize the existing joint.

ADD STUDY

- I In the Home toolbar, click Add Study to open the Add Study window.
- 2 Go to the Add Study window.

- 3 Find the Studies subsection. In the Select Study tree, select General Studies>Stationary.
- 4 Click Add Study in the window toolbar.
- 5 In the Home toolbar, click Add Study to close the Add Study window.

STUDY 2

In the **Home** toolbar, click **Compute**.

RESULTS

Stress

Duplicate the stress plot used for the fully flexible model to get Figure 4.

Stress 1

- I In the Model Builder window, right-click Stress and choose Duplicate.
- 2 In the Settings window for 3D Plot Group, locate the Data section.
- 3 From the Dataset list, choose Study 2/Solution 2 (sol2).
- 4 In the Stress I toolbar, click Plot.

Make sure that the first study still solves for the flexible structure.

STUDY I

Step 1: Stationary

- I In the Model Builder window, under Study I click Step I: Stationary.
- 2 In the Settings window for Stationary, locate the Physics and Variables Selection section.
- 3 Select the Modify model configuration for study step check box.
- 4 In the Physics and variables selection tree, select Component I (compl)> Multibody Dynamics (mbd), Controls spatial frame>Rigid Domain I.
- 5 Click / Disable.