

Fatigue Analysis of a Wheel Rim

Introduction

During the development of safety critical components like a car wheel rim, making sure fatigue cracks do not occur is one of the most important tasks. When a final prototype is available this is ensured by testing, but prototype production and testing are time consuming and expensive activities. Good predictions from simulations can keep down the number the number of prototypes to a minimum.

In this example, you do a fatigue evaluation on a model of a wheel rim, subjected to the load history from a simulated test.

Model Definition

For a definition of geometry, loads and boundary conditions, see the documentation for the model *Submodel in a Wheel Rim* in the Structural Mechanics Module Application Library.

The fatigue limit (in terms if the stress amplitude) is known for two cases with pure axial loading. For pure tension it is 95 MPa, and for fully reversed loading it is 125 MPa. In this model, you use the Findley criterion, so the Findley parameters have to be derived from these data.

In pure tension, the Findley criterion can be written as

$$\sqrt{\left(\frac{\Delta\sigma}{2}\right)^2 + \left(k \cdot \sigma_{\max}\right)^2} + k \cdot \sigma_{\max} = 2f$$

This means that you have to solve the simultaneous equations

$$\sqrt{95^{2} + (k \cdot 190)^{2}} + k \cdot 190 = 2f$$

$$\sqrt{125^{2} + (k \cdot 125)^{2}} + k \cdot 125 = 2f$$

to get the Findley parameters f and k. The result is f = 84 MPa and k = 0.30.

Results and Discussion

The fatigue usage factor distribution is shown in Figure 1. The maximum value is about 0.67, which should indicate that the design is good when taking into account that the required safety factor has been included in the load. In Figure 2 the stress histories in the

critical point are displayed. The loading is slightly nonproportional, and has a compressive mean stress, which is captured by the Findley criterion.



Surface: Fatigue usage factor (1) Max/Min Surface: Fatigue usage factor (1)

Figure 1: Fatigue usage factor using the Findley criterion.



Figure 2: Stress histories in the critical point.

Notes About the COMSOL Implementation

In this example, you perform the fatigue analysis as an additional study step in a model which already contains the results from a stress analysis. Since the critical points for fatigue crack initiation is on the free surface of the body, it is sufficient to do the fatigue evaluation on the boundary, and not in the domain. This reduces CPU and memory requirements significantly.

Application Library path: Fatigue_Module/Stress_Based/rim_fatigue

ROOT

In this example you will start from an existing model which is an example in the Structural Mechanics Module.

- I From the File menu, choose Open.
- 2 From the Application Libraries root, browse to the folder Structural_Mechanics_Module/Tutorials and double-click the file rim_submodel.mph.

RESULTS

Stress in Submodel

If the model was stored without solutions, you will now have to run **Study 1** and **Study 2** before continuing.

ADD PHYSICS

- I In the Home toolbar, click 🙀 Add Physics to open the Add Physics window.
- 2 Go to the Add Physics window.
- 3 In the tree, select Structural Mechanics>Fatigue (ftg).
- 4 Find the Physics interfaces in study subsection. In the table, clear the Solve check boxes for Study 1 and Study 2.
- 5 Click Add to Component 2 in the window toolbar.
- 6 In the Home toolbar, click 🙀 Add Physics to close the Add Physics window.

FATIGUE (FTG)

Stress-Based I

- I Right-click **Component 2 (comp2)>Fatigue (ftg)** and choose the boundary evaluation **Stress-Based**.
- **2** Select Boundaries 2–5 only.
- 3 In the Settings window for Stress-Based, locate the Solution Field section.
- 4 From the Physics interface list, choose Solid Mechanics 2 (solid2).

MATERIALS

Material 3 (mat3)

- I In the Model Builder window, expand the Component 2 (comp2)>Materials node.
- 2 Right-click Component 2 (comp2)>Materials and choose Blank Material.
- 3 In the Settings window for Material, locate the Geometric Entity Selection section.
- **4** From the **Geometric entity level** list, choose **Boundary**.
- 5 From the Selection list, choose All boundaries.
- 6 Locate the Material Contents section. In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Normal stress sensitivity coefficient	k_Findley	0.30	1	Findley
Limit factor	f_Findley	84[MPa]	Pa	Findley

ADD STUDY

- I In the Home toolbar, click $\sim\sim$ Add Study to open the Add Study window.
- 2 Go to the Add Study window.
- **3** Find the **Physics interfaces in study** subsection. In the table, clear the **Solve** check boxes for **Solid Mechanics (solid)** and **Solid Mechanics 2 (solid2)**.
- 4 Find the Studies subsection. In the Select Study tree, select

Preset Studies for Selected Physics Interfaces>Fatigue.

- 5 Click Add Study in the window toolbar.
- 6 In the Home toolbar, click $\sim\sim$ Add Study to close the Add Study window.

STUDY 3

Step 1: Fatigue

- I In the Settings window for Fatigue, locate the Values of Dependent Variables section.
- 2 Find the Values of variables not solved for subsection. From the Settings list, choose User controlled.
- 3 From the Method list, choose Solution.
- 4 From the Study list, choose Study 2, Stationary.
- **5** In the **Home** toolbar, click **= Compute**.

RESULTS

Max/Min Surface 1

- I In the Fatigue Usage Factor (ftg) toolbar, click i More Plots and choose Max/ Min Surface.
- 2 In the Settings window for Max/Min Surface, locate the Expression section.
- 3 In the **Expression** text field, type ftg.fus.
- 4 Click to expand the Advanced section. Locate the Display section. From the Display list, choose Max.
- 5 Locate the Text Format section. In the Display precision text field, type 2.

The maximum value of fatigue usage factor may be outside the expected area because a singularity can be present on the edge of the boundary where the fixed constraint is applied. Add a selection to ensure to display the maximum value of the surface of interest.

Selection 1

- I In the Fatigue Usage Factor (ftg) toolbar, click 🝖 Selection.
- **2** Select Boundary 5 only.
- 3 In the Fatigue Usage Factor (ftg) toolbar, click 🗿 Plot.

Fatigue Usage Factor (ftg)

- I In the Model Builder window, click Fatigue Usage Factor (ftg).
- 2 In the Settings window for 3D Plot Group, locate the Plot Settings section.
- 3 From the View list, choose View 4.

In order to get the location of the point with maximum fatigue usage, zoom in on the maximum marker and click on it in the graphics window. You will then see the value and the coordinates in the Table window. The location will be approximately (0.016, 0.092, 0.088). Use these coordinates to create a Cut Point 3D dataset for detailed evaluation of the stress history in the critical point.

Cut Point 3D 1

- I In the **Results** toolbar, click **Cut Point 3D**.
- 2 In the Settings window for Cut Point 3D, locate the Data section.
- 3 From the Dataset list, choose Study 2/Solution 2 (3) (sol2).
- 4 Locate the **Point Data** section. In the **X** text field, type **0.0164**.
- **5** In the **Y** text field, type **0.0924**.
- **6** In the **Z** text field, type **0.0884**.

7 Select the Snap to closest boundary check box.

ID Plot Group 5

In the **Results** toolbar, click \sim **ID Plot Group**.

Point Graph 1

- I Right-click ID Plot Group 5 and choose Point Graph.
- 2 In the Settings window for Point Graph, locate the Data section.
- 3 From the Dataset list, choose Cut Point 3D I.
- 4 Click Replace Expression in the upper-right corner of the y-Axis Data section. From the menu, choose Component 2 (comp2)>Solid Mechanics 2>Stress>
 Stress tensor (spatial frame) N/m²>solid2.sx Stress tensor, x component.
- 5 Locate the y-Axis Data section. From the Unit list, choose MPa.
- 6 Click to expand the **Coloring and Style** section. Find the **Line style** subsection. From the **Line** list, choose **Cycle**.
- 7 Locate the x-Axis Data section. From the Axis source data list, choose All solutions.
- 8 Locate the Coloring and Style section. In the Width text field, type 3.
- 9 Click to expand the Legends section. Select the Show legends check box.
- **IO** From the Legends list, choose Manual.

II In the table, enter the following settings:

Legends

SX

Point Graph 2

- I Right-click Point Graph I and choose Duplicate.
- 2 In the Settings window for Point Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type solid2.sy.
- 4 Locate the Legends section. In the table, enter the following settings:

Legends

sy

Point Graph 3

- I Right-click Point Graph 2 and choose Duplicate.
- 2 In the Settings window for Point Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type solid2.sz.

4 Locate the Legends section. In the table, enter the following settings:

Legends

sz

Point Graph 4

- I Right-click Point Graph 3 and choose Duplicate.
- 2 In the Settings window for Point Graph, locate the y-Axis Data section.
- 3 In the **Expression** text field, type solid2.sxy.
- 4 Locate the Legends section. In the table, enter the following settings:

Legends

sxy

Point Graph 5

- I Right-click Point Graph 4 and choose Duplicate.
- 2 In the Settings window for Point Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type solid2.syz.
- 4 Locate the Legends section. In the table, enter the following settings:

Legends

syz

Point Graph 6

- I Right-click Point Graph 5 and choose Duplicate.
- 2 In the Settings window for Point Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type solid2.sxz.
- 4 Locate the Legends section. In the table, enter the following settings:

Legends

sxz

Stress History

- I In the Model Builder window, under Results click ID Plot Group 5.
- 2 In the Settings window for ID Plot Group, type Stress History in the Label text field.
- **3** Click to expand the **Title** section. From the **Title type** list, choose **Manual**.
- 4 In the Title text area, type Stress history in critical point.

5 In the **Stress History** toolbar, click **O Plot**.