

Porous Absorber

Introduction

This is a model of acoustic absorption by a porous acoustic open cell foam. In porous materials the sound propagates in a network of small interconnected pores. Because the dimensions of the pores are small, losses occur due to thermal conduction and viscous friction. Acoustic foams are used to sound proof rooms and ducts as well as to treat reverberation problems in rooms (see Ref. 1).

The aim of the model is to characterize the absorption properties — more specifically, the specific surface impedance and the absorption coefficient — of a layer of melamine foam in terms of sound incidence angle and frequency. The melamine foam contains an air inclusion. An analytical solution exists in the case where the layer is uniform. The model uses a 2D geometry of such a system.

Model Definition

Figure 1 depicts the geometry of the modeled system, in which an incident sound field hits the porous melamine foam layer at angle θ . An air inclusion, circular domain of radius a, is present in the porous layer. The incident wave has wave vector **k**. In the figure, the dotted lined indicates the boundaries of the model domain. You only model a portion of width W and apply periodic Floquet boundary conditions on the left and right boundaries to extend the domain to infinity. The incident field is modeled by applying a background pressure field to the air domain. At the top, a perfectly matched layer (PML) domain is used to model an infinitely large air domain. The thickness of the porous melamine layer is $H_p = 10$ cm and the height of the modeled air region is H = 30 cm. The height of the PML domain is H_{pml} .

Model the melamine foam using the Pressure Acoustics interface's Poroacoustics domain feature using the Johnson-Champoux-Allard model with a rigid frame. This is an equivalent fluid model for a rigid frame porous material, a so-called five parameter semiempirical equivalent fluid model. See *About the Poroacoustics Models* in the *Acoustics Module User's Guide*. The surrounding fluid is air, and the material parameters for the foam are as listed in Table 1 (following Ref. 2, material sample number 31).

SYMBOL	VALUE	DESCRIPTION
ε _p	0.995	Porosity
$R_{ m f}$	10,500 Pa·s/m ²	Flow resistivity
S	0.49	Viscous characteristic length parameter
$L_{ m th}$	470 μm	Thermal characteristic length

TABLE I: MELAMINE FOAM MATERIAL PARAMETERS.

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SYMBOL	VALUE	DESCRIPTION	
$L_{ m v}$	240 μm	Viscous characteristic length	
τ	1.0059	Tortuosity factor	

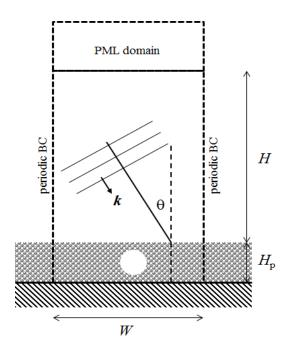


Figure 1: Geometry of the modeled system, the air inclusion has a radius a.

The incident background pressure field is given as

$$p_{\rm inc} = e^{-i(\mathbf{k} \cdot \mathbf{x})} \qquad \mathbf{k} = k_0(\sin\theta, -\cos\theta) \tag{1}$$

where θ is the incidence angle and k_0 is the wave number in the free field (air domain). The pressure *p* solved for in the model is the total field and the scattered field p_{scat} is given as $p_{\text{scat}} = p - p_{\text{inc}}$. Note that this expression for the scattered field is only valid in the air domain, as the incident field is not known a priori in the porous material.

Two parameters that characterize the absorption properties of the porous absorber are the specific surface impedance Z and the absorption coefficient α (see Ref. 1). The absorption coefficient, which represents the ratio of the absorbed and incident energy, is defined as

$$\alpha = 1 - |R|^2 \qquad R = \frac{p_{\text{scat}}}{p_{\text{inc}}}$$
(2)

where R is the pressure reflection coefficient that gives the ratio of the scattered to the incident pressure. The surface specific impedance (normalized by the plane wave characteristic impedance) is defined as

$$Z = \frac{1}{\rho c u_{\rm n}} \tag{3}$$

where ρ is the density of air, *c* is the speed of sound, and $u_n = \mathbf{u} \cdot \mathbf{n}$ is the normal velocity at the surface of the melamine layer. Both coefficients are dependent on frequency and on the incidence angle.

UNIFORM POROUS LAYER SOLUTION

In the case of a uniform porous layer (with no air inclusions) of thickness H_p backed by a sound hard wall an analytical solution exists for the surface impedance, reflection coefficient, and absorption coefficient (see Ref. 1). The surface normal impedance (normalized by the characteristic plane wave impedance) is given by

$$Z_{\text{ana}} = \frac{1}{\rho c} \frac{-iZ_c k_c}{k_x} \cot(k_x H_p)$$

$$k_x = \sqrt{k_c^2 - k_y^2}$$

$$k_y = k_0 \sin(\theta)$$
(4)

where a subscript "c" represents complex-valued impedance and wave number variables from the Poroacoustic domain. From the normal impedance the absorption coefficient is deduced.

Results and Discussion

Figure 2 and Figure 3 plot the scattered and total fields for an incidence angle of 45° and the frequency f = 10 kHz. Notice how the wave is absorbed in the porous layer.

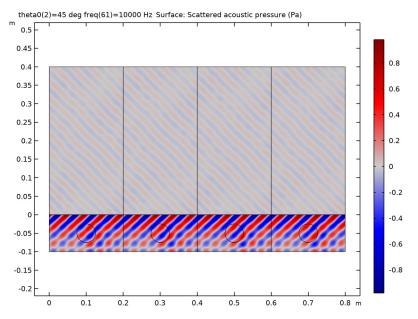


Figure 2: Scattered field for an incidence angle of 45° and frequency f = 10 kHz.

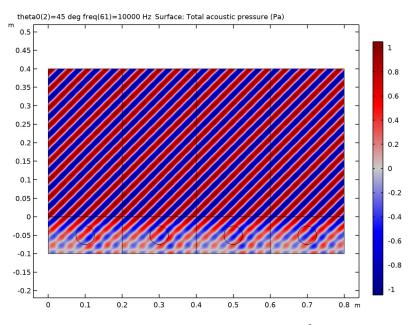


Figure 3: Total acoustic pressure for an incidence angle of 45° and frequency f = 10 kHz.

Figure 4 depicts the total sound pressure level at the surface of the porous melamine layer. Figure 5 plots the specific acoustic impedance at the surface of the porous absorber, and Figure 6 shows the absorption coefficient. The latter are compared to the analytical solution of a uniform porous layer.

The dependency of the surface specific impedance on incidence angle and frequency is important for modeling absorbers as impedance boundary conditions in, for example, a Ray Acoustics model. In larger model systems the present model could be used as a "submodel" to determine appropriate impedance boundary conditions. The real part of the impedance (the resistance) is associated with energy loss whereas the imaginary part (the reactance) is associated with phase changes of the field. The reciprocal value of the impedance is the admittance.

In this system, the absorption coefficient approaches 1 for increasing frequency. This corresponds to the frequency where the product between the porous absorber height H_p and $k_y \pi^{-1}$ of the incident wave is equal to one. This is where half a wavelength fits into the absorbing layer.

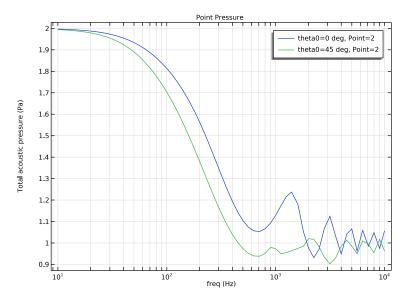


Figure 4: Sound pressure level at the surface of the porous absorber.

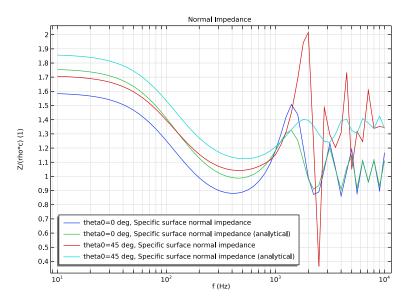


Figure 5: The specific acoustic impedance at the surface of the porous absorber.

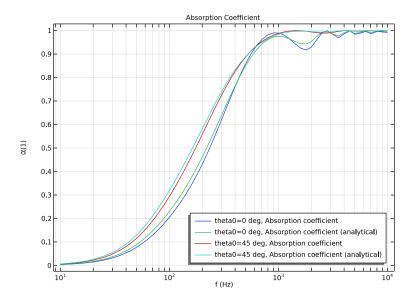


Figure 6: Absorption coefficient for the porous melamine absorber as function of frequency and incidence angle. Compared to the analytical solution of a uniform layer.

Notes About the COMSOL Implementation

PERIODIC FLOQUET BOUNDARY CONDITION

Apply a periodic Floquet boundary condition to model an infinite periodic structure. The periodicity is determined by the wave number of the background (incident) pressure field. The relation between the pressure at the left and right boundaries of the model domain is

$$p(\mathbf{x}) = p(\mathbf{x} + \mathbf{d})e^{-i(\mathbf{k} \cdot \mathbf{d})}$$
(5)

where $\mathbf{d} = (W, 0)$ is a vector extending from the left to the right boundary and \mathbf{k} is the wave vector defined in Equation 1. COMSOL Multiphysics automatically calculates the vector \mathbf{d} when applying the Floquet periodicity.

VISUALIZE PERIODIC SOLUTION

To visualize the periodic solution, create an Array 2D dataset and enable **Floquet-Bloch periodicity** under **Advanced** section. Enter the same **Wave vector** as used in the periodic conditions.

COMPARING TO THE ANALYTICAL SOLUTION

In the results section the simulated absorption coefficient and surface impedance are compared to the analytical solution of a uniform porous layer. To make a verification of the model simply select the inclusion (the circular air domain) as a Poroacoustic domain and run the model again. You will find that the analytical and model results show perfect agreement.

References

1. T.J. Cox and P. D'Antonio, Acoustic Absorbers and Diffusers, Theory, Design and Applications, 2nd ed., Taylor and Francis, 2009.

2. N. Kino and T. Ueno, "Comparison between characteristic lengths and fiber equivalent diameter in glass fiber and melamine foam materials of similar flow resistivity", *J. App. Acoustics*, vol. 69, pp. 325–331, 2008.

Application Library path: Acoustics_Module/Building_and_Room_Acoustics/ porous_absorber

Modeling Instructions

From the File menu, choose New.

NEW

In the New window, click 🙅 Model Wizard.

MODEL WIZARD

- I In the Model Wizard window, click **Q** 2D.
- 2 In the Select Physics tree, select Acoustics>Pressure Acoustics>Pressure Acoustics, Frequency Domain (acpr).
- 3 Click Add.
- 4 Click 🔿 Study.
- 5 In the Select Study tree, select General Studies>Frequency Domain.
- 6 Click **M** Done.

GLOBAL DEFINITIONS

Load the parameters for the model. The list of parameters include geometry definitions, definitions used in the mesh, and material parameters for the melamine foam.

Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- 3 Click 📂 Load from File.
- **4** Browse to the model's Application Libraries folder and double-click the file porous_absorber_parameters.txt.

GEOMETRY I

Rectangle 1 (r1)

- I In the Geometry toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- 3 In the Width text field, type W.
- **4** In the **Height** text field, type H+Hpml.
- 5 Click to expand the Layers section. Clear the Layers on bottom check box.
- 6 Select the Layers on top check box.
- 7 In the table, enter the following settings:

Layer name	Thickness (m)	
Layer 1	Hpml	

Rectangle 2 (r2)

- I In the **Geometry** toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- 3 In the Width text field, type W.
- 4 In the **Height** text field, type Hp.
- **5** Locate the **Position** section. In the **y** text field, type -Hp.

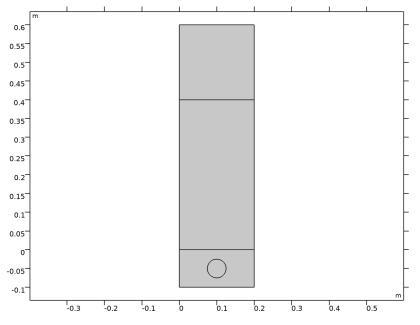
Circle I (c1)

- I In the **Geometry** toolbar, click 🕑 **Circle**.
- 2 In the Settings window for Circle, locate the Size and Shape section.
- 3 In the Radius text field, type a.
- 4 Locate the **Position** section. In the **x** text field, type W/2.

- **5** In the **y** text field, type -Hp/2.
- 6 In the Geometry toolbar, click 🟢 Build All.

7 Click the **Zoom Extents** button in the **Graphics** toolbar.

The geometry should look like that in the figure below.



DEFINITIONS

Load the expressions defining the background pressure field, see Equation 1, as well as the surface impedance and absorption coefficient, see Equation 2 and Equation 3, from a file.

Variables I

- I In the Home toolbar, click $\partial =$ Variables and choose Local Variables.
- 2 In the Settings window for Variables, locate the Variables section.
- 3 Click **b** Load from File.
- **4** Browse to the model's Application Libraries folder and double-click the file porous_absorber_variables.txt.

Load the expressions defining the analytical expressions for a single porous layer with a sound hard backing, see Equation 4.

Variables 2

I In the Home toolbar, click a= Variables and choose Local Variables.

- 2 In the Settings window for Variables, locate the Variables section.
- 3 Click 📂 Load from File.
- 4 Browse to the model's Application Libraries folder and double-click the file porous_absorber_analytical.txt.

Define two nonlocal integration couplings that act on points in the geometry. You will use them later to map (or probe) values from these points. One in the porous domain and one in the air domain.

Integration 1 (intop1)

- I In the Definitions toolbar, click *P* Nonlocal Couplings and choose Integration.
- 2 In the Settings window for Integration, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Point.
- 4 Select Point 1 only.

Integration 2 (intop2)

- I In the Definitions toolbar, click 🖉 Nonlocal Couplings and choose Integration.
- 2 In the Settings window for Integration, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Point.
- 4 Select Point 3 only.

Define a nonlocal average coupling on the porous-air interface.

Average 1 (aveop1)

- I In the Definitions toolbar, click Nonlocal Couplings and choose Average.
- 2 In the Settings window for Average, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- **4** Select Boundary 4 only.

Now proceed to set up the material properties. Add air as the default domain material and create a new material to define the melamine foam porosity.

ADD MATERIAL

- I In the Home toolbar, click 🙀 Add Material to open the Add Material window.
- 2 Go to the Add Material window.
- 3 In the tree, select Built-in>Air.
- 4 Click Add to Component in the window toolbar.
- 5 In the Home toolbar, click 🙀 Add Material to close the Add Material window.

MATERIALS

Melamine Foam

- I In the Model Builder window, under Component I (compl) right-click Materials and choose Blank Material.
- 2 In the Settings window for Material, type Melamine Foam in the Label text field.
- **3** Click to expand the Material Properties section. In the Material properties tree, select **Basic Properties>Porosity**.
- 4 Click + Add to Material.
- 5 Locate the Material Contents section. In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Porosity	epsilon	epsilonP0	I	Basic

- 6 Locate the Material Properties section. In the Material properties tree, select Acoustics> Poroacoustics model>Thermal characteristic length (Lth).
- 7 Click + Add to Material.
- 8 Locate the Material Contents section. In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Flow resistivity	Rf	Rf0	Pa·s/m²	Poroacoustics model
Thermal characteristic length	Lth	Lth0	m	Poroacoustics model
Viscous characteristic length	Lv	Lv0	m	Poroacoustics model
Tortuosity factor	tau	tau0	I	Poroacoustics model

Notice that the parameter for the tortuosity is called tau0, not to be confused with the material property of the static viscous tortuosity.

Now set up the physics and the boundary conditions. First, define the incident background pressure field, see Equation 2, then the Floquet condition, see Equation 5, and finally porous material properties for the melamine foam.

DEFINITIONS

Perfectly Matched Layer I (pmll)

- I In the Definitions toolbar, click W Perfectly Matched Layer.
- **2** Select Domain 3 only.

- 3 In the Settings window for Perfectly Matched Layer, locate the Scaling section.
- 4 From the Coordinate stretching type list, choose Rational.
- 5 In the PML scaling factor text field, type 1/cos(theta0).

It is recommended to modify the scaling of the PML to account for the direction of the plane wave.

PRESSURE ACOUSTICS, FREQUENCY DOMAIN (ACPR)

Background Pressure Field 1

- In the Model Builder window, under Component I (compl) right-click Pressure Acoustics, Frequency Domain (acpr) and choose Background Pressure Field.
- **2** Select Domain 2 only.
- **3** In the Settings window for Background Pressure Field, locate the Background Pressure Field section.
- 4 From the Pressure field type list, choose User defined.
- **5** In the $p_{\rm b}$ text field, type p_inc.

Now, add the poroacoustic domain defining the melamine foam.

Poroacoustics 1

- I In the Physics toolbar, click **Domains** and choose Poroacoustics.
- 2 Select Domain 1 only.
- 3 In the Settings window for Poroacoustics, locate the Poroacoustics Model section.
- 4 From the Poroacoustics model list, choose Johnson-Champoux-Allard (JCA).
- 5 Locate the Porous Matrix Properties section. From the Porous elastic material list, choose Melamine Foam (mat2).

It is good practice to add a periodic condition for each type of domain, in this case, one for the PML, one for the pressure acoustics, and one for the porous domain. This is especially the case when using a background pressure field.

Periodic Condition 1

- I In the **Physics** toolbar, click **Boundaries** and choose **Periodic Condition**.
- **2** Select Boundaries 1 and 8 only.
- 3 In the Settings window for Periodic Condition, locate the Periodicity Settings section.
- **4** From the **Type of periodicity** list, choose **Floquet periodicity**.

5 Specify the \mathbf{k}_{F} vector as

kx	x
ky	у

Periodic Condition 2

- I In the Physics toolbar, click Boundaries and choose Periodic Condition.
- **2** Select Boundaries **3** and **9** only.
- 3 In the Settings window for Periodic Condition, locate the Periodicity Settings section.

4 From the **Type of periodicity** list, choose **Floquet periodicity**.

5 Specify the \mathbf{k}_{F} vector as

kx	x
ky	у

Periodic Condition 3

- I In the **Physics** toolbar, click **Boundaries** and choose **Periodic Condition**.
- 2 Select Boundaries 5 and 10 only.
- 3 In the Settings window for Periodic Condition, locate the Periodicity Settings section.
- **4** From the **Type of periodicity** list, choose **Floquet periodicity**.
- **5** Specify the \mathbf{k}_{F} vector as

kx x ky y

MESH I

Free Triangular 1

- I In the Mesh toolbar, click Kree Triangular.
- 2 In the Settings window for Free Triangular, locate the Domain Selection section.
- 3 From the Geometric entity level list, choose Domain.
- **4** Select Domains 1 and 4 only.

Size

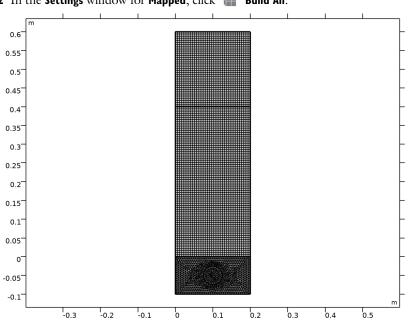
- I In the Model Builder window, click Size.
- 2 In the Settings window for Size, locate the Element Size section.
- 3 Click the **Custom** button.

4 Locate the **Element Size Parameters** section. In the **Maximum element size** text field, type lambda_min/6.

This mesh resolves the smallest wavelength of the study lambda_min with 6 elements.

Mapped I

I In the Mesh toolbar, click Mapped.



2 In the Settings window for Mapped, click 📗 Build All.

STUDY I

Step 1: Frequency Domain

- I In the Model Builder window, under Study I click Step I: Frequency Domain.
- 2 In the Settings window for Frequency Domain, locate the Study Settings section.
- 3 Click Range.
- 4 In the Range dialog box, choose ISO preferred frequencies from the Entry method list.
- 5 In the Start frequency text field, type 10.
- 6 In the Stop frequency text field, type 10000.
- 7 From the Interval list, choose 1/6 octave.

8 Click Replace.

You thus solve the model on a logarithmic frequency axis from 10 Hz to 10^4 Hz. Add a parametric sweep over the incidence angle theta0 for the values 0^o and 45^o .

Parametric Sweep

- I In the Study toolbar, click **Parametric Sweep**.
- 2 In the Settings window for Parametric Sweep, locate the Study Settings section.
- 3 Click + Add.
- 4 In the table, enter the following settings:

Parameter name	Parameter value list	
theta0 (Incident wave angle)	0[deg] 45[deg]	

5 In the **Study** toolbar, click **= Compute**.

Create an array dataset that will help you plot the Floquet periodic solution on several unit cells. Add a selection to not show the unphysical solution in the PML domain.

RESULTS

Study I/Parametric Solutions I (sol2)

In the Model Builder window, expand the Results>Datasets node, then click Study I/ Parametric Solutions I (sol2).

Selection

- I In the **Results** toolbar, click 🐐 **Attributes** and choose **Selection**.
- 2 In the Settings window for Selection, locate the Geometric Entity Selection section.
- 3 From the Geometric entity level list, choose Domain.
- 4 Select Domains 1, 2, and 4 only.

Array 2D I

- I In the **Results** toolbar, click **More Datasets** and choose **Array 2D**.
- 2 In the Settings window for Array 2D, locate the Data section.
- 3 From the Dataset list, choose Study I/Parametric Solutions I (sol2).
- 4 Locate the Array Size section. In the X size text field, type 4.

Enable **Floquet-Bloch periodicity** and enter the **Wave vector** to visualize the periodic solution.

- 5 Click to expand the Advanced section. Select the Floquet-Bloch periodicity check box.
- 6 Find the Wave vector subsection. In the X text field, type kx.

7 In the Y text field, type ky.

Total Acoustic Pressure

- I In the Model Builder window, under Results click Acoustic Pressure (acpr).
- 2 In the Settings window for 2D Plot Group, type Total Acoustic Pressure in the Label text field.
- 3 Locate the Data section. From the Dataset list, choose Array 2D I.

Surface 1

- I In the Model Builder window, expand the Total Acoustic Pressure node, then click Surface I.
- 2 In the Total Acoustic Pressure toolbar, click 💽 Plot.
- **3** Click the **Comextents** button in the **Graphics** toolbar.

Compare the resulting plot with that in Figure 3.

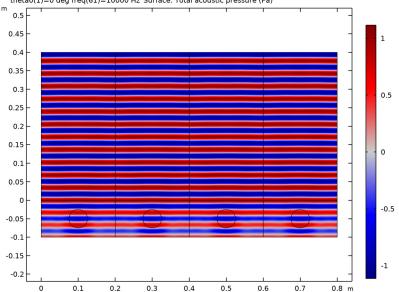
Now change the incidence angle from 45° to 0° .

Total Acoustic Pressure

- I In the Model Builder window, click Total Acoustic Pressure.
- 2 In the Settings window for 2D Plot Group, locate the Data section.
- 3 From the Parameter value (theta0 (deg)) list, choose 0.
- **4** In the **Total Acoustic Pressure** toolbar, click **O** Plot.

5 Click the **F Zoom Extents** button in the **Graphics** toolbar.

The result should look like that in the following figure.



theta0(1)=0 deg freq(61)=10000 Hz Surface: Total acoustic pressure (Pa)

Scattered Acoustic Pressure

- I Right-click Total Acoustic Pressure and choose Duplicate.
- 2 In the Settings window for 2D Plot Group, type Scattered Acoustic Pressure in the Label text field.

Now, plot the scattered acoustic pressure.

Surface 1

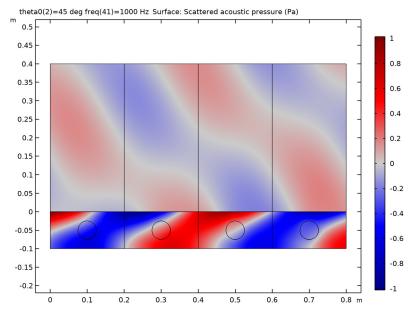
- I In the Model Builder window, expand the Scattered Acoustic Pressure node, then click Surface I.
- 2 In the Settings window for Surface, locate the Expression section.
- **3** In the **Expression** text field, type acpr.p_s.

Scattered Acoustic Pressure

- I In the Model Builder window, click Scattered Acoustic Pressure.
- 2 In the Settings window for 2D Plot Group, locate the Data section.
- 3 From the Parameter value (theta0 (deg)) list, choose 45.

- 4 In the Scattered Acoustic Pressure toolbar, click Plot.Compare the resulting plot with that in Figure 2.Now change the frequency from 10 kHz to 1 kHz.
- 5 From the Parameter value (freq (Hz)) list, choose 1000.
- 6 In the Scattered Acoustic Pressure toolbar, click 🗿 Plot.

The result should look like that in the figure below.



Total Acoustic Pressure

Next, plot the incident acoustic pressure field at 10 kHz for an incidence angle of 0°

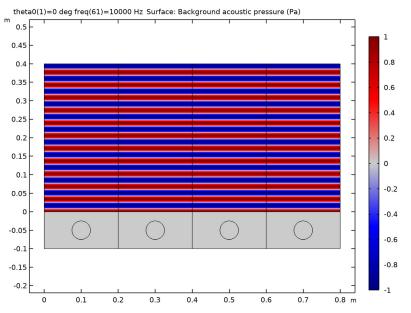
Incident Acoustic Pressure

- I In the Model Builder window, right-click Total Acoustic Pressure and choose Duplicate.
- 2 In the Settings window for 2D Plot Group, type Incident Acoustic Pressure in the Label text field.

Surface 1

- I In the Model Builder window, expand the Incident Acoustic Pressure node, then click Surface I.
- 2 In the Settings window for Surface, locate the Expression section.
- 3 In the Expression text field, type acpr.p_b.

4 In the **Incident Acoustic Pressure** toolbar, click **I** Plot.



Next, create 1D plots to depict the absorption properties of the melamine absorber.

First, reproduce the plot in Figure 4, which shows the sound pressure level at the surface of the porous melamine layer.

Point Pressure

- I In the Home toolbar, click 📠 Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Point Pressure in the Label text field.
- 3 Locate the Data section. From the Dataset list, choose Study I/ Parametric Solutions I (sol2).
- 4 Click to expand the Title section. From the Title type list, choose Label.

Point Graph 1

- I Right-click Point Pressure and choose Point Graph.
- **2** Select Point 2 only.
- 3 In the Settings window for Point Graph, click to expand the Legends section.
- 4 Select the Show legends check box.
- 5 In the Point Pressure toolbar, click 🗿 Plot.

6 Click the **x-Axis Log Scale** button in the **Graphics** toolbar.

Proceed by plotting the acoustic normal impedance at the surface of the porous melamine layer. The plot should look like that in Figure 5.

Normal Impedance

- I In the Home toolbar, click 🚛 Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Normal Impedance in the Label text field.
- 3 Locate the Data section. From the Dataset list, choose Study I/ Parametric Solutions I (sol2).
- 4 Locate the Title section. From the Title type list, choose Label.
- 5 Locate the Plot Settings section. Select the x-axis label check box.
- 6 In the associated text field, type f (Hz).
- 7 Select the y-axis label check box.
- 8 In the associated text field, type Z/(rho*c) (1).
- 9 Locate the Legend section. From the Position list, choose Lower left.

Global I

- I Right-click Normal Impedance and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
Z	1	Specific surface normal impedance
Z_ana	1	Specific surface normal impedance (analytical)

4 Click to expand the Legends section. In the Normal Impedance toolbar, click 💽 Plot.

5 Click the **x-Axis Log Scale** button in the **Graphics** toolbar.

Finally, plot the absorption coefficient of the porous melamine layer for the two studied incidence angles (Figure 6).

Absorption Coefficient

- I In the Home toolbar, click 📠 Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Absorption Coefficient in the Label text field.
- 3 Locate the Data section. From the Dataset list, choose Study I/ Parametric Solutions I (sol2).

- 4 Locate the Title section. From the Title type list, choose Label.
- 5 Locate the Plot Settings section. Select the x-axis label check box.
- 6 In the associated text field, type f (Hz).
- 7 Select the y-axis label check box.
- 8 In the associated text field, type \alpha (1).
- 9 Locate the Legend section. From the Position list, choose Lower right.

Global I

- I Right-click Absorption Coefficient and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
alpha	1	Absorption coefficient
alpha_ana	1	Absorption coefficient (analytical)

4 In the **Absorption Coefficient** toolbar, click **I Plot**.

5 Click the **u x-Axis Log Scale** button in the **Graphics** toolbar.