

Lumped Loudspeaker Driver

This is a model of a moving-coil loudspeaker where a lumped parameter analogy represents the behavior of the electrical and mechanical speaker components. This lumped model is coupled to a 2D axisymmetric pressure acoustics model describing the surrounding air domain. The model is solved with two different configurations, first with an open backvolume and then with a closed back-volume. The main part of the analysis focuses on the open back-volume configuration.

Electric circuit representations of transducers are well known and widely used. In the loudspeaker industry such models have been employed for a long time and with great success. The parameters that characterize the low-frequency performance of a loudspeaker are commonly known as the Thiele-Small or the small-signal parameters. It is common to use these parameters to design and simulate a desired speaker performance in terms of, for example, the on-axis speaker response, the electrical impedance, and the diaphragm velocity.

In the low-frequency regime the motion of the speaker cone can be approximated by the motion of a rigid piston in an infinite baffle. Analytical expressions exist for the sound field radiated from a piston. In the model at hand, the simple piston geometry is replaced by a more realistic speaker cone shape, and the sound field is solved using the finite element method. Of course this is still an approximation as the motion of the speaker cone is still assumed to be rigid; however, the spatial response of the speaker is more realistic. The methodology of lumping certain parts of a complex system is a general and powerful approach that can be applied to other systems. A first extension of the current model can be to only lump the electrical components and model the mechanical components fully.

The output from the model includes, among many things, the total electric impedance, the on-axis sound pressure level at a nominal driving voltage, and the mean speaker cone velocity. The results are compared with an analytical solution based on the flat piston approximation. Finally, it is also shown how to create a so-called directivity plot of the speaker.

Note: This application requires the Acoustics Module and the AC/DC Module.

A schematic representation of a moving coil loudspeaker is given in Figure 1. The figure shows a cross section of a loudspeaker. The speaker driver is placed in an infinite baffle with free space in front and on the back of the speaker. This will be referred to as the open backvolume configuration. In the closed back-volume configuration a closed box is located behind the driver (see Figure 3). The speaker cone consists of the outer suspension, the diaphragm, and a dust cap (not marked in the figure). The mechanical and electrical components of the speaker that are lumped are visualized inside the dotted box. On the electrical side it includes the voice-coil and magnetic system (permanent magnet and pole pieces), and on the mechanical side it includes the moving mass of the voice coil and speaker cone, the spring effect of the spider and outer suspension, as well as possible losses due to damping in these suspensions. In the discussion below the lumped model of the driver itself is not concerned with the open or closed configuration of the back-volume. The effect of where the driver is located or how it is mounted in a system is modeled in the finite element domain.

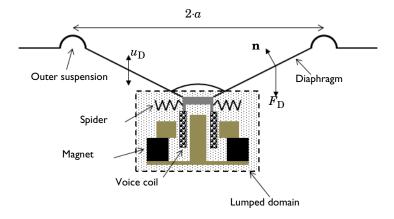


Figure 1: Schematic representation of a moving coil speaker unit.

ELECTROACOUSTIC ANALOGY

In an electroacoustic analogy, mechanical and acoustic physical properties such as force, velocity, pressure, and volume velocity are equated to voltages and currents in an analogous electrical circuit. In this model, the impedance analogy is used for the mechanical components only. This means that the current in the circuit represents the velocity of a moving part (SI unit: m/s) and the voltage represents a force (SI unit: N). In contrast, the acoustics is not represented by a circuit in this model. In general, however,

in an acoustic analogous circuit, a voltage represents pressure (SI unit: Pa) and a current represents volume velocity (SI unit: m³/s).

The passive elements in a circuit — that is, resistors, inductors, and capacitors — represent different processes, namely resistance, mass movement, and compliance, respectively. The components and their analogs are listed in Table 1 (see also Ref. 1 for further details).

TABLE 1: PASSIVE ELEMENTS IN MECHANICAL AND ACOUSTIC ANALOGIES.

ELECTRICAL COMPONENT	MECHANICAL (IMPEDANCE ANALOGY)	ACOUSTIC
ResistorSI unit: Ω	 Mechanical resistance SI unit: N·s/m Losses due to friction, as in a car suspension. 	 Acoustic resistance SI unit: kg/(m⁴·s) Dissipative losses due to viscosity and thermal conduction in the fluid.
• Inductor • SI unit: H	 Mass SI unit: kg Inertial force, acceleration of a mass. 	 Acoustic mass SI unit: kg/m⁴ Internal force of a volume of air that is accelerated but not compressed.
CapacitorSI unit: F	 Compliance SI unit: m/N The inverse of the spring constant. 	 Acoustic compliance SI unit: m⁵/N Compressibility effect of a volume of air that is not accelerated.

The active components in a circuit represent sources; they are external forces (pressures) or applied velocities (volume velocities). The sources are also used for couplings between the electrical, mechanical, and acoustic domains.

Note: To make the units fit in COMSOL, it is necessary to make unit transforms to fit the electrical units. When, for example, inserting a capacitor representing a mechanical compliance, C_{MS} , in the physics interface type: $C_{MS}[F*N/m]$

THE ANALOGOUS CIRCUIT

The analogous circuit for the electrical and mechanical parts of the system sketched in Figure 1 is shown in Figure 2. The upper figure represents the voice coil electrical system and the lower figure the mechanical analogue of the speaker cone, suspensions, and mass of the voice coil. In both figures the node numbers are also shown — they are very useful when setting up the circuit model in COMSOL.

In Figure 2 (top) the external voltage source is denoted by V_0 and the generator output resistance is $R_{\rm g}$, in this model $R_{\rm g}$ = 0 Ω . The voice coil resistance is $R_{\rm E}$, and the voice coil inductance is $L_{\rm E}(\omega)$, which is frequency dependent. The losses in the magnetic circuit are modeled through the frequency dependent resistance $R'_{\rm E}(\omega)$. The current controlled voltage source $BL\cdot u_{\rm D}$, represents the back induced electromagnetic voltage generated when the voice coil (of length L) moves with velocity $u_{\rm D}$ in the magnetic field B. Here BL is the product of the magnetic field strength and the voice coil length L (see also Ref. 2 on how this can be modeled). In the electrical circuit the current is denoted $i_{\rm c}$.

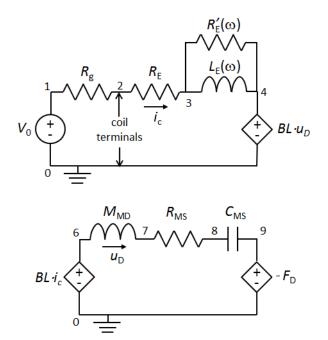


Figure 2: Analogous circuits for the electrical (top) and mechanical (bottom) properties of the speaker driver.

In the mechanical system given in Figure 2 (bottom) the current in the circuit is the velocity of the voice coil and speaker cone in the axial direction (see Figure 1). The force acting on the diaphragm is given by $BL \cdot i_c$. This is the Lorentz force on a voice coil of length L with current i_c , where B is the magnetic flux density. The force acts on a system of mass $M_{\rm MD}$ (voice coil and diaphragm assembly). The resistance $R_{\rm MS}$ models the

damping and $C_{
m MS}$ the mechanical compliance in the speaker suspensions (both spider and outer suspension). Finally, the voltage source -F_D represents the pressure force acting on the speaker diaphragm (in the axial direction). Notice the minus sign that indicates that the force acts against the movement of the diaphragm (see Ref. 1). The force is given by

$$F_D = \int (\Delta p \cdot n_z) dA \tag{1}$$

where Δp is the pressure drop across the diaphragm and n_z is the axial component of the surface normal \mathbf{n} (see Figure 1). This expression gives the couplings from the acoustic finite element model to the lumped circuit model. On the other hand, the coupling from the circuit model to the finite element model comes from specifying the velocity \mathbf{v} on the surface of the diaphragm, which is given by

$$\mathbf{v} = u_{\mathbf{D}} \mathbf{e}_{\mathbf{z}} \tag{2}$$

where $\mathbf{e}_{\mathbf{z}}$ is the unit vector in the axial direction (see Figure 3), $u_{\mathbf{D}}$ is the current in the mechanical circuit analog, and multiplication with $i\omega$ represents differentiation with respect to time.

Note: A version of this model using the *Lumped Mechanical System* interface also exists as a tutorial: Lumped Loudspeaker Driver Using a Lumped Mechanical System. In that model the mechanical part of the system uses the mobility analogy where the present model is based entirely on the impedance analogy. The model additionally requires the Structural Mechanics Module and the Multibody Dynamics Module.

SMALL-SIGNAL PARAMETERS

The fundamental small-signal parameters of the system (or Thiele-Small parameters) are the physical parameters of the loudspeaker driver. They are all constants, as they are given in the low-frequency limit of the model. See Table 2 below.

TABLE 2: FUNDAMENTAL SMALL-SIGNAL PARAMETERS.

SYMBOL	VALUE	DESCRIPTION
$M_{ m MD}$	33.4 g	Moving mass (voice coil and diaphragm)
$C_{ m MS}$	1.18·10 ⁻³ m/N	Suspension compliance
$R_{ m MS}$	1.85 Ns/m	Suspension mechanical losses (damping)
$L_{ m E}$	6.89 mH	Voice coil inductance
$R_{ m E}$	7 Ω	DC resistance of voice coil

TABLE 2: FUNDAMENTAL SMALL-SIGNAL PARAMETERS.

SYMBOL	VALUE	DESCRIPTION
BL	II.4 T·m	Force factor
$S_{ m D}$	$a^2\pi$	Driver equivalent area
a	I2 cm	Piston radius of driver (equivalent)

In Table 2 the radius α is the piston radius of the driver. It is typically taken as half the diaphragm aperture diameter, measured half-way into the outer suspension.

Note: The constants used in this model are all taken from Example 22 in Ref. 1.

Other small-signal parameters may be determined on basis of the fundamental small-signal parameters. Actually, these parameters are the ones measured when characterizing a driver; the fundamental parameters are inferred hereafter. The measured small-signal parameters are given in Table 3. In the table the speed of sound is denoted c_0 and the density of air is ρ_0 . The values of these parameters are determined in the model using the Parameters feature under Global Definitions.

TABLE 3: SMALL-SIGNAL PARAMETERS (MEASURED).

Symbol	Expression	Description
$M_{ m MS}$	$M_{ m MD} + 2S_{ m D}^2 rac{8 ho_0}{3\pi^2 a}$	Moving mass including acoustic load (low frequency approximation).
$F_{ m s}$	$rac{1}{2\pi\sqrt{C_{ m MS}M_{ m MS}}}$	Fundamental resonant frequency.
$Q_{ m ES}$	$\frac{2\pi F_{\rm s} M_{\rm MS} R_{\rm E}}{\left(BL\right)^2}$	Electrical Q factor at $F_{ m S}$
$Q_{ m MS}$	$\frac{2\pi F_{\rm s} M_{\rm MS}}{R_{\rm MS}}$	Mechanical Q factor at $F_{ m s}$
$Q_{ m TS}$	$\frac{Q_{\rm MS}Q_{\rm ES}}{Q_{\rm MS}+Q_{\rm ES}}$	Total Q factor at $F_{ m s}$

TABLE 3: SMALL-SIGNAL PARAMETERS (MEASURED).

Symbol	Expression	Description
$V_{ m AS}$	$\rho_0 c_0^2 S_{\mathrm{D}}^2 C_{\mathrm{MS}}$	Equivalent volume compliance (air volume having the same compliance as the suspension).
η_0	$\frac{4\pi^2 F_{_{\rm S}}^3 V_{{\rm AS}}}{c_0^3 Q_{{\rm ES}}}$	Reference efficiency of the driver.

OTHER PARAMETERS

The model also uses other parameters than the small-signal parameters presented above. Two of them are the frequency dependent voice coil inductance $L_{\rm E}(\omega)$ and the resistance $R'_{\rm E}(\omega)$ associated with the losses in the magnetic system. A model for this is given in Ref. 1, for the higher audio frequencies, defining

$$L_{\rm E}(\omega) = \left[\frac{L_{\rm E}}{\sin\left(n_{\rm e}\frac{\pi}{2}\right)}\right] \omega^{(n_{\rm e}-1)} \qquad R'_{\rm E}(\omega) = \left[\frac{L_{\rm E}}{\cos\left(n_{\rm e}\frac{\pi}{2}\right)}\right] \omega^{n_{\rm e}}$$

where n_e is the so-called voice coil loss factor. For $n_e = 1$ and for in the low frequency limit the loss-less behavior is recovered where $R'_{\rm E}$ is an open circuit and $L_{\rm E}$ is constant. In this model $n_e = 0.7$.

Another expression used in the model is the acoustic radiated power which is given by

$$P_{\rm AR} = \int (\mathbf{n} \cdot \mathbf{I}) dA \tag{3}$$

where **I** is the intensity vector, whose components are defined in COMSOL as (acpr.Ir, acpr.Iz). The integral is taken over the front upper side of the speaker diaphragm. The electric input power (RMS value) is defined as

$$P_{\rm E} = 0.5 \operatorname{Re}(V_0 \cdot i_{\rm c}^*) \tag{4}$$

where * is the complex conjugate operator, and, finally, the speaker efficiency comparing the input electric power to the radiated acoustic power is

$$\eta = \frac{P_{AR}}{P_{E}}$$

A number of the analytical results derived in Ref. 1, which are based on the piston approximation, are also used for comparison with the current more realistic speaker cone shape. They are the low-frequency and high-frequency approximations to the speaker velocity, the on axis pressure at 1 m in front of the piston, and the acoustic radiated power from a piston in an infinite baffle. Expressions for these are given in the reference and under the variable node Component I>Definitions>Analytical approximations in the model.

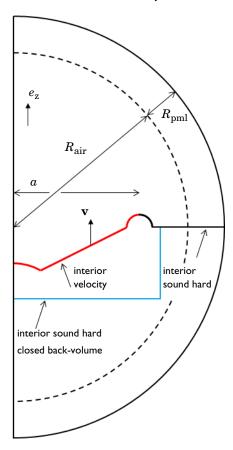


Figure 3: Computational domain and boundary conditions.

FINITE ELEMENT DOMAIN

The computational domain where the pressure acoustics model is solved is sketched in Figure 3. It represents the speaker cone, dust cap, and outer suspension, in an infinite baffle in a 2D axisymmetric model. On the speaker (red line) an interior velocity boundary condition is applied (according to Equation 2) and the rest of the baffle is an interior

sound hard wall. In the closed back-volume configuration the blue line is also modeled an interior sound hard wall. In the open configuration no condition is applied to that boundary (it is transparent). The domain is truncated with a perfectly matched layer (PML) to mimic an infinite open domain. Note that the interior sound hard boundary of the baffle should be applied inside the PML. This is to avoid erroneous energy leaks between the two sides of the infinite baffle. The air domain has a radius of $R_{\rm air}$ while the thickness of the PML layer is $R_{\rm pml}$. For more information about PMLs in acoustics, see the Modeling with the Pressure Acoustics Branch (FEM-Based Interfaces) section in the Pressure Acoustics Module User's Guide.

Results and Discussion

In this section the results of the open back-volume configuration are first discussed in detail. Then selected results are presented for the closed back-volume configuration.

OPEN BACK-VOLUME CONFIGURATION

The generated pressure field is shown in Figure 4 for two frequencies around 1 kHz and 5 kHz. This plot shows the directive characteristic of the speaker cone at increasing frequencies; this nature is discussed more at the end of this section when discussing the directivity plot Figure 11.

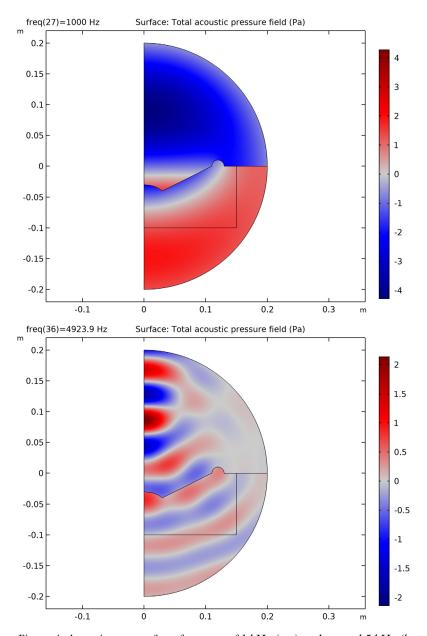


Figure 4: Acoustic pressure for a frequency of 1 kHz (top) and around 5 kHz (bottom).

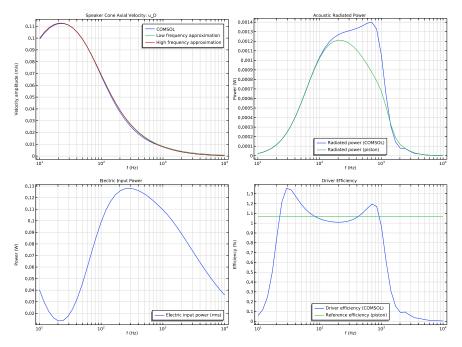


Figure 5: The frequency-dependent diaphragm velocity amplitude (top left), acoustic radiated power (top right), electric input power (bottom left), and efficiency (bottom right).

In Figure 5 and Figure 6 a series of characteristic loudspeaker driver curves are shown. In some figures the results obtained with the hybrid lumped and finite element of COMSOL are compared with theoretical curves that are based on the piston approximation.

In the top left of Figure 5 the calculated speaker cone axial velocity u_D is shown together with two theoretical curves representing the approximate high-frequency and lowfrequency behavior of a piston driver (see Ref. 1). The general trend is that the lowfrequency curve fits well for most of the frequency range. The high-frequency curve is seen to converge toward the higher frequencies. If you increase the frequency range of the model the trend is that the high-frequency approximation is a better fit as expected.

The next two graphs in Figure 5 (top right and bottom left) represent the acoustically radiated power P_{AR} (see Equation 3) and the electric input power P_{E} (see Equation 4). At the low frequencies the behavior is just as the piston model, as expected. The transition to nonmatching models occurs when the speaker diameter becomes comparable to half the wavelength at around 700 Hz. The last graph of Figure 5 represents the driver efficiency given in percent (%), that is, the ratio of the input electric power to the acoustic radiated

power. The actual efficiency is seen to match well with the predicted reference efficiency η_0 of about 1 %.

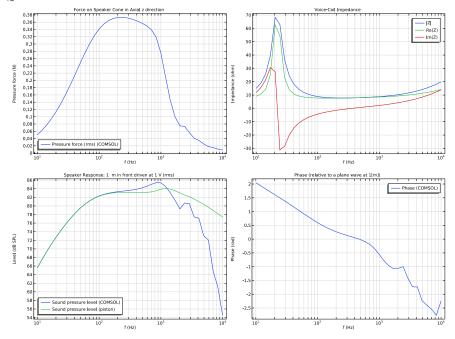


Figure 6: The frequency dependent pressure force on the speaker cone (rms value) (top left), voice coil impedance (top right), speaker sensitivity (bottom left), and phase response (bottom right).

In Figure 6 (top left) the RMS value of the acoustic pressure force $F_{\rm D}$ on the membrane (see Equation 1) is shown as function of frequency. The RMS value is obtained by the usual formula

$$\sqrt{0.5 \cdot F_{\mathrm{D}} \cdot F_{\mathrm{D}}^{*}}$$

where the * represents the complex conjugate.

Figure 6 (top right) represents the voice coil impedance (absolute value, real, and imaginary part) calculated as V_i/i_c . The resonance in the electric system is seen to coincide with the fundamental resonance frequency $F_{\rm S}$ = 22.0 Hz (see Table 3 and the Global **Definitions>Parameters** list in the model).

The two last figures in Figure 6 (bottom left and right) represent the speaker response (sound pressure level) measured 1 m in front and driven at 1 V RMS and the relative phase measured in the same point. The sensitivity is seen to match the piston model at the low frequencies as expected and it has realistic values for a speaker unit. The phase is represented as the phase of the pressure p(0,1 m) relative to the phase a plane wave would have in the same point $\exp(-ik_0z)$ with z = 1 m. In this way the pure distance (phase lag) component of the phase has been removed.

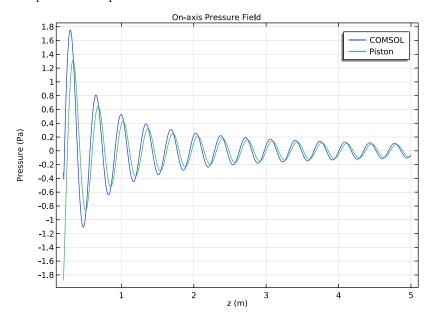


Figure 7: The pressure along the center z-axis from just outside the computational domain to a distance of 5 m. Evaluated for a frequency of 1000 Hz. The pressure is calculated using the exterior field calculation feature.

The pressure field along the z-axis is shown in Figure 7 from $z = R_{air}$ to z = 5 m, evaluated at a frequency of 1000 Hz. In the model a Parametrized Curve 2D is used to evaluate the exterior-field pressure outside the computational mesh. The figure compares the modeled pressure with the analytical on axis pressure from a piston. The agreement is seen to be quite good at this frequency. For lower frequencies the agreement becomes much better while the two curves diverge at higher frequencies, as expected (change the evaluation frequency in the plot to see this trend).

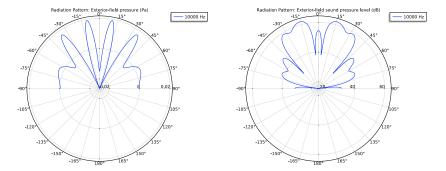


Figure 8: Exterior-field pressure and sound pressure level evaluated at a distance of 1 m the half sphere in front of the speaker and at 10 kHz.

The pressure and the sound pressure level evaluated at 1 m, using the dedicated radiation pattern plots, are shown in Figure 8, here evaluated at 10 kHz. In the figure the 0° mark corresponds to the axial z direction. Both figures show a very strong directive pattern as expected at this high frequency.

In Figure 9 and Figure 10 the radiated intensity is illustrated and evaluated around the four frequencies 100 Hz, 1000 Hz, 5 kHz, and 10 kHz. The color plot represents the magnitude of the intensity vector **I**, the domain vector field represents the components of the intensity vector, and finally the vectors plotted on the edges represent the surface normals. The normals are useful when setting up, for example, the expression for the total radiated power (Equation 3). In this case it is necessary to use the up() operator to get the intensity on the upper side of the speaker cone (seen relative to the normal direction). The four plots clearly show how the acoustic energy is focused for increasing frequencies.

The so-called directivity plot of the speaker unit is shown in Figure 11. This plot represents a contour plot of the sound pressure level $L_{
m p}$ evaluated along a half circle in front of the speaker as function of the angle and the frequency, that is, $L_{\rm p}(\theta, f)$. As this plot is a more advanced and nonstandard plot in COMSOL the axis labels are a bit off. The x-axis represents the angle and actually runs from -90° to 90°. The y-axis is a logarithmic frequency axis running from 10¹ Hz to 10⁴ Hz. The plot illustrates how the spatial response goes from a nearly omni-directional constant value at the low frequencies, through a single lobe response at intermediate frequencies, and ends up as a complex directive pattern at high frequencies. This type of plot is very often used, in industry, to characterize speakers and speaker units.

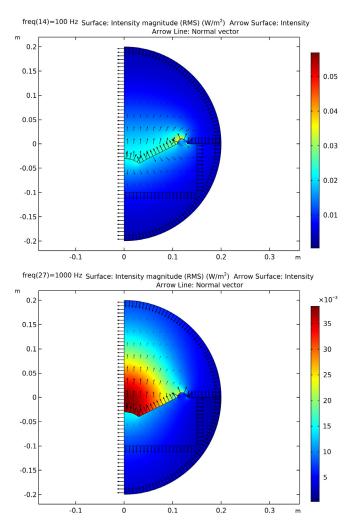


Figure 9: Intensity magnitude (color plot), intensity vector field (domain arrows), and surface normals (edge arrows) for 100 Hz and 1000 Hz.

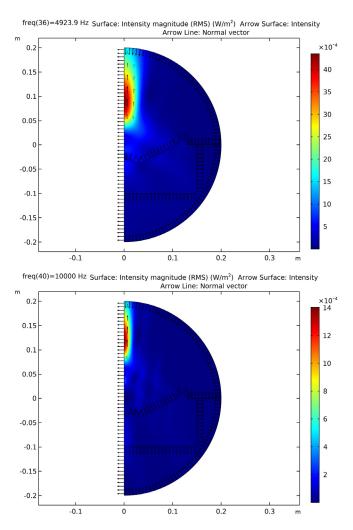


Figure 10: Intensity magnitude (color plot), intensity vector field (domain arrows), and surface normals (edge arrows) around 5 kHz and for $10~\rm kHz$.

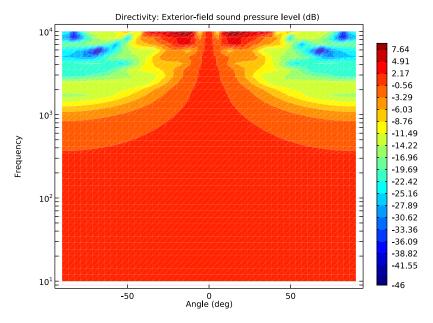


Figure 11: Directivity plot for the speaker. The x-axis is a scaled azimuthal angle that runs from -90° to 90° and the y axis is a logarithmic frequency axis that runs from 10^1 Hz to $10^4 Hz = 10 kHz.$

CLOSED BACK-VOLUME CONFIGURATION

When the model is solved with the closed back-volume configuration, the response of the system changes. This is easily seen in the voice coil impedance curves in Figure 12. The location of the fundamental resonance frequency F_s is seen to increase. This is because the presence of the closed space behind the speaker diaphragm corresponds to an increased mechanical stiffness. The sensitivity of the speaker is depicted in Figure 13. The presence of the closed volume is seen to heavily influence the low frequency performance of the speaker. This is typically also why speaker cabinets have a ventilation port, that is, to reduce this low frequency roll off.

Testing the model with a closed back-volume also has the advantage of testing the entire model setup (the electrical circuit to FEM coupling). For example, a sign error in the applied pressure force on the diaphragm would give more or less correct results for the open configuration, but would not predict the expected resonance shift. This kind of sanity tests are important when setting up numerical models.

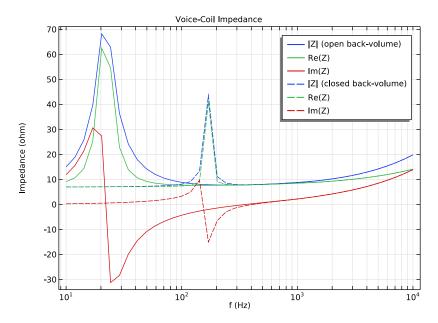


Figure 12: Voice coil impedance curves for both the open (solid) and closed (dashed) back-volume configurations.

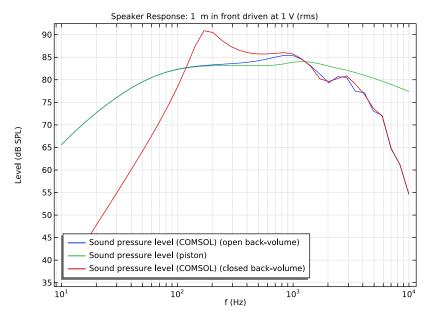


Figure 13: Sensitivity curves for the open back-volume configuration, the analytical results, and the closed back-volume configuration.

References

- 1. W. Marshall Leach, Jr., Introduction to Electroacoustics and Audio Amplifier Design, Kendall Hunt, 2010.
- 2. Loudspeaker Driver Model Documentation, from the COMSOL Application Library.

Application Library path: Acoustics_Module/Electroacoustic_Transducers/ lumped_loudspeaker_driver

Modeling Instructions

From the File menu, choose New.

NEW

In the New window, click Model Wizard.

MODEL WIZARD

- I In the Model Wizard window, click 2D Axisymmetric.
- 2 In the Select Physics tree, select Acoustics>Pressure Acoustics, Frequency Domain (acpr).
- 3 Click Add.
- 4 In the Select Physics tree, select AC/DC>Electrical Circuit (cir).
- 5 Click Add.
- 6 Click Study.
- 7 In the Select Study tree, select General Studies>Frequency Domain.
- 8 Click Done.

GLOBAL DEFINITIONS

Load all the model parameters from a file; they include all the small signal parameters (Table 2 and Table 3) as well as geometry parameters.

Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- 3 Click Load from File.
- 4 Browse to the model's Application Libraries folder and double-click the file lumped_loudspeaker_driver_parameters.txt.

Build the simple 2D axisymmetric geometry of the speaker driver by drawing some circles and lines.

GEOMETRY I

Circle I (c1)

- I In the Geometry toolbar, click Circle.
- 2 In the Settings window for Circle, locate the Size and Shape section.
- 3 In the Radius text field, type 1 [cm].
- 4 In the Sector angle text field, type 180.
- **5** Locate the **Position** section. In the **r** text field, type **a**.

Circle 2 (c2)

- I In the Geometry toolbar, click Circle.
- 2 In the Settings window for Circle, locate the Size and Shape section.
- 3 In the Radius text field, type Rair+Rpml.
- 4 In the Sector angle text field, type 180.
- **5** Locate the **Rotation Angle** section. In the **Rotation** text field, type -90.
- **6** Click to expand the **Layers** section. In the table, enter the following settings:

Layer name	Thickness (m)
Layer 1	Rpml

Polygon I (poll)

- I In the Geometry toolbar, click Polygon.
- 2 In the Settings window for Polygon, locate the Object Type section.
- 3 From the Type list, choose Open curve.
- **4** Locate the **Coordinates** section. In the table, enter the following settings:

r (m)	z (m)
3[cm]	-4[cm]
a-1[cm]	0

Polygon 2 (pol2)

- I In the Geometry toolbar, click Polygon.
- 2 In the Settings window for Polygon, locate the Object Type section.
- **3** From the **Type** list, choose **Open curve**.
- **4** Locate the **Coordinates** section. In the table, enter the following settings:

r (m)	z (m)
a+1[cm]	0
Rair	0

- 5 Click Build All Objects.
- 6 Click the Zoom Extents button in the Graphics toolbar.

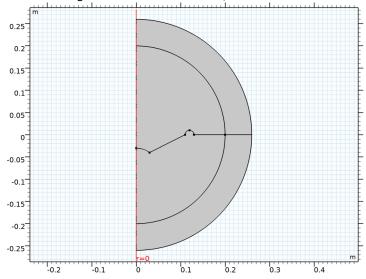
Quadratic Bézier I (qb I)

- I In the Geometry toolbar, click More Primitives and choose Quadratic Bézier.
- 2 In the Settings window for Quadratic Bézier, locate the Control Points section.

- 3 In row 2, set r to 1.8[cm].
- 4 In row 3, set r to 3[cm].
- 5 In row 1, set z to -3[cm].
- 6 In row 2, set z to -3.1[cm].
- 7 In row 3, set z to -4[cm].
- 8 Locate the Weights section. In the 2 text field, type 1.5.
- 9 Click Build All Objects.

Delete Entities I (del I)

- I In the Model Builder window, right-click Geometry I and choose Delete Entities.
- 2 On the object c1, select Boundaries 3 and 4 only.
- 3 In the Settings window for Delete Entities, click Build Selected.



Polygon 3 (pol3)

- I In the Geometry toolbar, click Polygon.
- 2 In the Settings window for Polygon, locate the Object Type section.
- 3 From the Type list, choose Open curve.

4 Locate the **Coordinates** section. In the table, enter the following settings:

r (m)	z (m)
0.15	0
0.15	-0.1
0	-0.1

5 Click Build All Objects.

Now, set up all the variables, selections, and component couplings under the Definitions node. Load the variables from the two variable files (one for model variables and one for the analytical piston expressions). The selections represent the speaker cone surface with an internal acceleration (red line in Figure 3) and the internal sound hard wall.

DEFINITIONS

Variables 1

- I In the Home toolbar, click Variables and choose Local Variables.
- 2 In the Settings window for Variables, type Model variables in the Label text field.
- 3 Locate the Variables section. Click Load from File.
- **4** Browse to the model's Application Libraries folder and double-click the file lumped loudspeaker driver variables 1.txt.

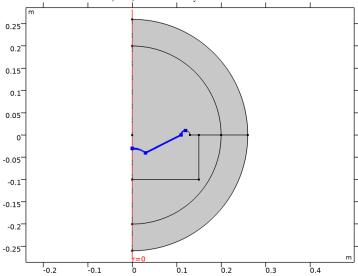
Variables 2

- I In the Home toolbar, click Variables and choose Local Variables.
- 2 In the Settings window for Variables, type Analytical approximations in the Label text field.
- 3 Locate the Variables section. Click Load from File.
- 4 Browse to the model's Application Libraries folder and double-click the file lumped_loudspeaker_driver_variables_2.txt.

Explicit I

- I In the **Definitions** toolbar, click **Explicit**.
- 2 In the Settings window for Explicit, type Speaker in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.

4 Select Boundaries 8, 15, and 18 only.

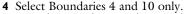


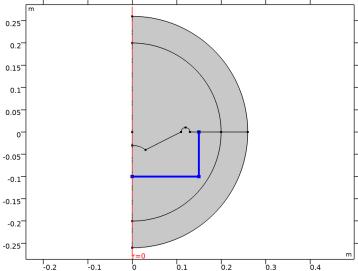
Explicit 2

- I In the **Definitions** toolbar, click **Explicit**.
- 2 In the Settings window for Explicit, type Baffle (interior wall) in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Select Boundaries 9, 11, 12, and 19 only.

Explicit 3

- I In the **Definitions** toolbar, click **Explicit**.
- 2 In the Settings window for Explicit, type Back Volume (interior wall) in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.





Integration | (intob |)

- I In the **Definitions** toolbar, click **Nonlocal Couplings** and choose **Integration**.
- 2 In the Settings window for Integration, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 From the Selection list, choose Speaker.
- **5** In the **Operator name** text field, type intop.
- **6** In the **Operator name** text field, type intop.

Perfectly Matched Layer I (pml1)

- I In the Definitions toolbar, click Perfectly Matched Layer.
- **2** Select Domains 1 and 5 only.
- 3 In the Settings window for Perfectly Matched Layer, locate the Scaling section.
- 4 From the Coordinate stretching type list, choose Rational.
- 5 In the PML scaling factor text field, type 0.5.
- 6 In the PML scaling curvature parameter text field, type 5.

You have now changed the default settings for the perfectly matched layer (PML). The new settings will improve the performance of the PML at very low frequencies. First of all, the acoustic radiated power should be positive. If the default settings had been used you would see a negative radiated power at low frequencies (see Figure 5 (top right) and the instructions on how to create the plot, further down). The issue is that at low frequencies the evanescent waves created by the moving speaker cone extend into the PML layer. The interaction between the scaled coordinate system in the PML and these waves may create an erroneous energy contribution to the model (can be either positive or negative). Note that, the evanescent waves decay in only a fraction of a wavelength.

A good way to investigate the performance of the PML is to make a sensitivity analysis on some parameter (for example the total radiated acoustic power) with respect to changes in the PML parameters. In this model we do not perform such a sensitivity analysis.

In general, increasing the curvature factor effectively shifts the resolving power of the PML toward the physical domain, which is necessary in this case since the evanescent components decay in only a fraction of a wavelength. However, if you increase it too much, you may lose resolution in the other end, that is, of the free space wavelength. Assuming that the PMLs work properly for high frequencies and curvature parameter 1, you can in principle do a convergence study increasing a constant curvature parameter until the low-frequency result converges (for 10 Hz) while making sure that the high-frequency result (for 10 kHz) is not affected. It turns out that a value of 5 yields good results in this model. A scaling factor of 0.5 further improves the results, but only by a small amount. Decreasing the scaling factor corresponds to compressing the PML layer (shortening it), which in turn effectively increases the mesh resolution.

ADD MATERIAL

- I In the Home toolbar, click Add Material to open the Add Material window.
- 2 Go to the Add Material window.
- 3 In the tree, select Built-in>Air.
- **4** Click **Add to Component** in the window toolbar.
- 5 In the Home toolbar, click Add Material to close the Add Material window.

Now, set up the physics and the boundary conditions for the model. Use the interior conditions at the diaphragm and on the infinite baffle. This condition will allow for a discontinuous pressure field.

PRESSURE ACOUSTICS, FREQUENCY DOMAIN (ACPR)

Interior Sound Hard Boundary (Wall) I

I In the Model Builder window, under Component I (compl) right-click Pressure Acoustics, Frequency Domain (acpr) and choose Interior Conditions> Interior Sound Hard Boundary (Wall).

- 2 In the Settings window for Interior Sound Hard Boundary (Wall), locate the Boundary Selection section.
- 3 From the Selection list, choose Baffle (interior wall).

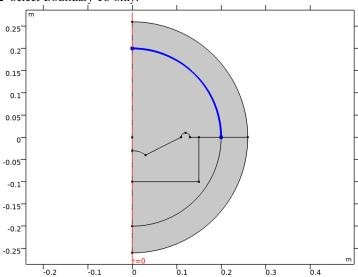
Interior Normal Velocity 1

- I In the Physics toolbar, click Boundaries and choose Interior Normal Velocity.
- 2 In the Settings window for Interior Normal Velocity, locate the Boundary Selection section.
- 3 From the Selection list, choose Speaker.
- **4** Locate the **Interior Normal Velocity** section. Specify the \mathbf{v}_0 vector as

0	r
u_D	z

Exterior Field Calculation 1

- I In the Physics toolbar, click Boundaries and choose Exterior Field Calculation.
- 2 Select Boundary 16 only.



- 3 In the Settings window for Exterior Field Calculation, locate the Exterior Field Calculation section.
- 4 From the Condition in the $z = z^0$ plane list, choose Symmetric/ Infinite sound hard boundary.

Note that, you have applied the exterior-field calculation condition only to the front of the speaker. In reality, the exterior-field condition should be applied to boundaries

surrounding all sources and scatterers. This is of course not possible with an infinite baffle. However, in this specific case, a trick can be used, namely employing the fact that symmetry (in z = 0) is equal to a sound-hard wall in pressure acoustics (as indicated in the UI). So, in the infinite baffle configuration, the exterior-field condition can still be used.

Interior Sound Hard Boundary (Wall) 2

- I In the Physics toolbar, click Boundaries and choose Interior Sound Hard Boundary (Wall).
- 2 In the Settings window for Interior Sound Hard Boundary (Wall), locate the Boundary Selection section.
- 3 From the Selection list, choose Back Volume (interior wall).

This interior wall condition will be deactivated in the first study when modeling the open configuration. In the second study the interior wall is active and the speaker operates with a closed back-volume.

Proceed to set up the electric circuit system for the electric and mechanical model. When building this look at Figure 3 for the references to the node numbers used in the model.

ELECTRICAL CIRCUIT (CIR)

Voltage Source VI

- I In the Model Builder window, under Component I (compl) right-click **Electrical Circuit (cir)** and choose **Voltage Source**.
- 2 In the Settings window for Voltage Source, locate the Node Connections section.
- **3** In the table, enter the following settings:

Label	Node names
P	1
n	0

4 Locate the **Device Parameters** section. In the $V_{\rm src}$ text field, type V0.

Resistor R1

- I In the Electrical Circuit toolbar, click Resistor.
- 2 In the Settings window for Resistor, locate the Node Connections section.

3 In the table, enter the following settings:

Label	Node names
Р	1
n	2

4 Locate the **Device Parameters** section. In the R text field, type R g.

Resistor R2

- I In the Electrical Circuit toolbar, click Resistor.
- 2 In the Settings window for Resistor, locate the Node Connections section.
- **3** In the table, enter the following settings:

Label	Node names
Р	2
n	3

4 Locate the **Device Parameters** section. In the R text field, type R_E.

Inductor LI

- I In the Electrical Circuit toolbar, click Inductor.
- 2 In the Settings window for Inductor, locate the Node Connections section.
- **3** In the table, enter the following settings:

Label	Node names
P	3
n	4

4 Locate the **Device Parameters** section. In the L text field, type L E.

Resistor R3

- I In the Electrical Circuit toolbar, click Resistor.
- 2 In the Settings window for Resistor, locate the Node Connections section.
- **3** In the table, enter the following settings:

Label	Node names
Р	3
n	4

4 Locate the **Device Parameters** section. In the R text field, type Rp_E.

Current-Controlled Voltage Source H1

- I In the Electrical Circuit toolbar, click Current-Controlled Voltage Source.
- 2 In the Settings window for Current-Controlled Voltage Source, locate the **Node Connections** section.
- **3** In the table, enter the following settings:

Label	Node names
Р	4
n	0

4 Locate the Device Parameters section. In the Gain text field, type BL[m/Wb*ohm]. Remember to select the measured current when the mechanical circuit components have been set up.

Current-Controlled Voltage Source H2

- I In the Electrical Circuit toolbar, click Current-Controlled Voltage Source.
- 2 In the Settings window for Current-Controlled Voltage Source, locate the **Node Connections** section.
- **3** In the table, enter the following settings:

Label	Node names
P	6
n	0

- 4 Locate the Device Parameters section. In the Gain text field, type BL[m/Wb*ohm].
- 5 From the Measure current for device list, choose Resistor R2.

Inductor L2

- I In the Electrical Circuit toolbar, click Inductor.
- 2 In the Settings window for Inductor, locate the Node Connections section.
- **3** In the table, enter the following settings:

Label	Node names
Р	6
n	7

4 Locate the **Device Parameters** section. In the L text field, type M_MD[H/kg].

Current-Controlled Voltage Source H1

- I In the Model Builder window, click Current-Controlled Voltage Source HI.
- 2 In the Settings window for Current-Controlled Voltage Source, locate the **Device Parameters** section.
- 3 From the Measure current for device list, choose Inductor L2.

Resistor R4

- I In the Electrical Circuit toolbar, click Resistor.
- 2 In the Settings window for Resistor, locate the Node Connections section.
- **3** In the table, enter the following settings:

Label	Node names
Р	7
n	8

4 Locate the **Device Parameters** section. In the R text field, type R_MS[ohm/kg*s].

Capacitor C1

- I In the Electrical Circuit toolbar, click Capacitor.
- 2 In the Settings window for Capacitor, locate the Node Connections section.
- **3** In the table, enter the following settings:

Label	Node names
P	8
n	9

4 Locate the **Device Parameters** section. In the C text field, type $C_MS[F*N/m]$.

Voltage Source V2

- I In the Electrical Circuit toolbar, click Voltage Source.
- 2 In the Settings window for Voltage Source, locate the Node Connections section.
- **3** In the table, enter the following settings:

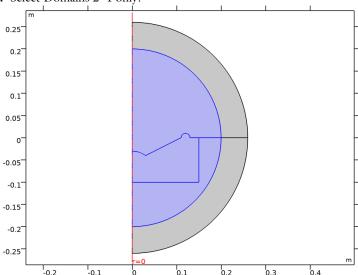
Label	Node names
Р	9
n	0

4 Locate the **Device Parameters** section. In the $V_{\rm src}$ text field, type -F_D[V/N]. The applied force is an external force (pressure acting on the membrane). When the force is increased the velocity of the membrane should decrease and vice versa. Now, build the mesh. It consists of a triangular mesh around the speaker and an extruded mesh in the PML region. In the boundary where the exterior-field is calculated add a single boundary layer. For more details on this look at the model Acoustic Scattering off an Ellipsoid in the Application Library.

MESH I

Free Triangular I

- I In the Model Builder window, under Component I (compl) right-click Mesh I and choose Free Triangular.
- 2 In the Settings window for Free Triangular, locate the Domain Selection section.
- 3 From the Geometric entity level list, choose Domain.
- 4 Select Domains 2-4 only.



Size

- I In the Model Builder window, click Size.
- 2 In the Settings window for Size, locate the Element Size section.
- 3 Click the **Custom** button.

- 4 Locate the Element Size Parameters section. In the Maximum element size text field, type 343[m/s]/fmax/8.
- 5 In the Minimum element size text field, type 343[m/s]/fmax/100.

Mapped I

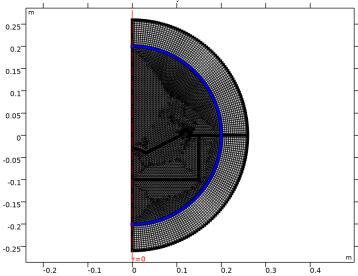
- I In the Model Builder window, right-click Mesh I and choose Mapped.
- 2 Click Build All.

Boundary Layers 1

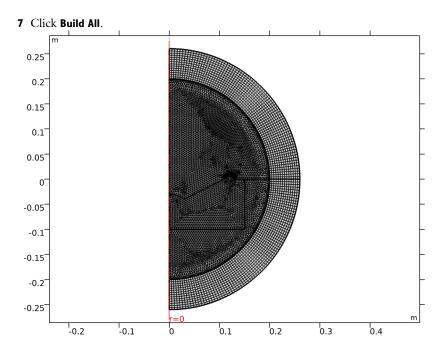
- I Right-click Mesh I and choose Boundary Layers.
- 2 In the Settings window for Boundary Layers, locate the Domain Selection section.
- 3 From the Geometric entity level list, choose Domain.
- **4** Select Domains 2–4 only.

Boundary Layer Properties

- I In the Model Builder window, click Boundary Layer Properties.
- **2** Select Boundaries 14 and 16 only.



- 3 In the Settings window for Boundary Layer Properties, locate the Boundary Layer Properties section.
- 4 In the Number of boundary layers text field, type 1.
- 5 From the Thickness of first layer list, choose Manual.
- 6 In the Thickness text field, type 343[m/s]/fmax/100.



STUDY I

- I In the Model Builder window, click Study I.
- 2 In the Settings window for Study, type Study 1 Open Back-Volume in the Label text field.

Step 1: Frequency Domain

- I In the Model Builder window, under Study I Open Back-Volume click Step 1: Frequency Domain.
- 2 In the Settings window for Frequency Domain, locate the Study Settings section.
- 3 In the Frequencies text field, type 10^{range(1,3/39,4)}. Disable the interior wall that defines the closed back-volume for this first study where we use the open configuration.
- 4 Locate the Physics and Variables Selection section. Select the Modify model configuration for study step check box.
- 5 In the Physics and variables selection tree, select Component I (compl)> Pressure Acoustics, Frequency Domain (acpr)>Interior Sound Hard Boundary (Wall) 2.
- 6 Click Disable.

7 In the Home toolbar, click Compute.

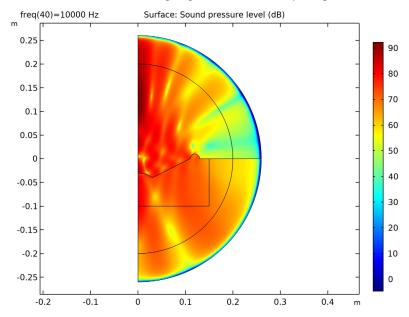
RESULTS

Acoustic Pressure (acpr)

First, look at the default plots. Investigate the 2D Sound Pressure Level (acpr) plot to verify the performance of the perfectly matched layer (PML). After doing this you can disable plotting in the PML region, which is unphysical. Secondly, look at the default exterior-field plots and make a few changes, before setting up a range of plots to investigate the loudspeaker driver performance.

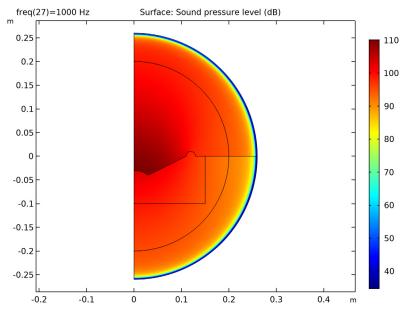
Sound Pressure Level (acpr)

Look at the sound pressure level (SPL) plots at the frequencies of 10 kHz, 1 kHz, and 10 Hz. Note that the SPL decreases nearly 100 dB over the width of the thickness of the PML. This means that the outgoing waves are extremely damped.

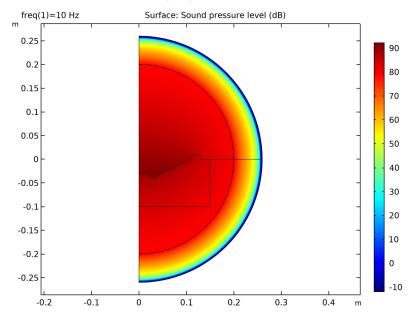


- I In the Model Builder window, click Sound Pressure Level (acpr).
- 2 In the Settings window for 2D Plot Group, locate the Data section.
- 3 From the Parameter value (freq (Hz)) list, choose 1000.

4 In the Sound Pressure Level (acpr) toolbar, click Plot.

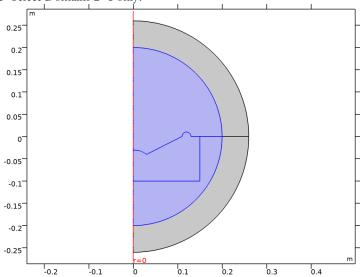


- 5 From the Parameter value (freq (Hz)) list, choose 10.
- 6 In the Sound Pressure Level (acpr) toolbar, click Plot.



Selection

- I In the Model Builder window, expand the Datasets node.
- 2 Right-click Study I Open Back-Volume/Solution I (soll) and choose Selection.
- 3 In the Settings window for Selection, locate the Geometric Entity Selection section.
- 4 From the Geometric entity level list, choose Domain.
- **5** Select Domains 2–4 only.



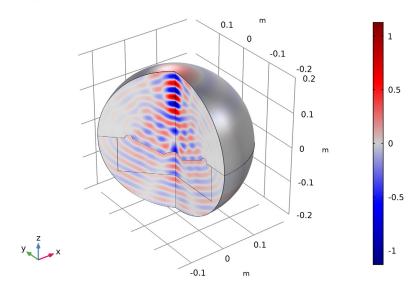
Acoustic Pressure (acpr)

- I In the Model Builder window, click Acoustic Pressure (acpr).
- 2 In the Settings window for 2D Plot Group, locate the Data section.
- 3 From the Parameter value (freq (Hz)) list, choose 1000.
- 4 In the Acoustic Pressure (acpr) toolbar, click Plot.
- 5 Click the Zoom Extents button in the Graphics toolbar.
- 6 From the Parameter value (freq (Hz)) list, choose 4923.9.
- 7 In the Acoustic Pressure (acpr) toolbar, click Plot.

These two plots should reproduce Figure 4.

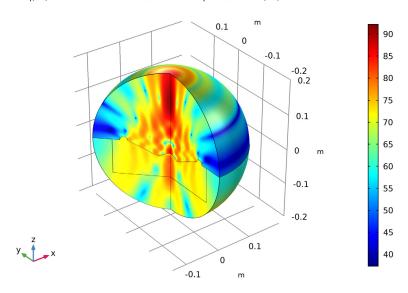
Acoustic Pressure, 3D (acpr)

freq(40)=10000 Hz Surface: Total acoustic pressure field (Pa)



Sound Pressure Level, 3D (acpr)

freq(40)=10000 Hz Surface: Sound pressure level (dB)



Radiation Pattern 1

- I In the Model Builder window, expand the Exterior-Field Sound Pressure Level (acpr) node, then click Radiation Pattern 1.
- 2 In the Settings window for Radiation Pattern, locate the Evaluation section.
- 3 Find the Angles subsection. From the Restriction list, choose Manual.
- 4 In the ϕ start text field, type -90.
- 5 In the ϕ range text field, type 180.
- 6 In the Exterior-Field Sound Pressure Level (acpr) toolbar, click Plot.

Radiation Pattern I

- I In the Model Builder window, expand the Exterior-Field Pressure (acpr) node, then click Radiation Pattern I.
- 2 In the Settings window for Radiation Pattern, locate the Evaluation section.
- 3 Find the Angles subsection. From the Restriction list, choose Manual.
- 4 In the ϕ start text field, type -90.
- **5** In the ϕ range text field, type 180.
- 6 In the Exterior-Field Pressure (acpr) toolbar, click Plot.

These two polar plots should reproduce Figure 8.

Now, create nine 1D plots that depict various speaker characteristic plots. Each plot includes a number of steps to setup the title, the axis, the plot name etc., these are optional and can be skipped. You can also just create the specific plot that is of interest to you. The next steps reproduce the graphs in Figure 5, Figure 6, and Figure 7.

The plots are the following:

- I Membrane Velocity
- **2** Acoustic Radiated Power
- **3** Electric Input Power
- **4** Efficiency
- **5** Force on Speaker Cone
- **6** Voice-Coil Impedance
- **7** Sensitivity
- 8 Phase
- 9 On-axis Exterior-Field Pressure

ID Plot Group 7

- I In the Home toolbar, click Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Diaphragm Velocity in the Label text field.
- 3 Click to expand the **Title** section. From the **Title type** list, choose **Manual**.
- 4 In the Title text area, type Speaker Cone Axial Velocity: u_D.
- **5** Locate the **Plot Settings** section. Select the **x-axis label** check box.
- 6 In the associated text field, type f (Hz).
- **7** Select the **y-axis label** check box.
- 8 In the associated text field, type Velocity amplitude (m/s).

Global I

- I Right-click Diaphragm Velocity and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
abs(u_D)	m/s	COMSOL
$abs(Qlf_D/S_D)$	m/s	Low frequency approximation
abs(Qhf_D/S_D)	m/s	High frequency approximation

- 4 In the Diaphragm Velocity toolbar, click Plot.
- 5 Click the x-Axis Log Scale button in the Graphics toolbar.

This plot should reproduce Figure 5 (top left).

ID Plot Group 8

- I In the Home toolbar, click Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Acoustic Radiated Power in the Label text field.
- 3 Locate the Title section. From the Title type list, choose Manual.
- 4 In the **Title** text area, type Acoustic Radiated Power.
- 5 Locate the Plot Settings section. Select the x-axis label check box.
- **6** In the associated text field, type **f** (Hz).
- 7 Select the y-axis label check box.
- 8 In the associated text field, type Power (W).

9 Locate the Legend section. From the Position list, choose Lower middle.

Global I

- I Right-click Acoustic Radiated Power and choose Global.
- 2 In the Settings window for Global, click Add Expression in the upper-right corner of the y-axis data section. From the menu, choose Component I>Definitions>Variables>P_AR -Radiated power (COMSOL) - W.
- 3 Click Add Expression in the upper-right corner of the y-axis data section. From the menu, choose Component I>Definitions>Variables>P AR ana - Radiated power (piston) -W.
- 4 In the Acoustic Radiated Power toolbar, click Plot.
- 5 Click the x-Axis Log Scale button in the Graphics toolbar.

This plot should reproduce Figure 5 (top right).

ID Plot Group 9

- I In the Home toolbar, click Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Electric Input Power in the Label text field.
- 3 Locate the Title section. From the Title type list, choose Manual.
- 4 In the **Title** text area, type Electric Input Power.
- 5 Locate the Plot Settings section. Select the x-axis label check box.
- 6 In the associated text field, type f (Hz).
- 7 Select the y-axis label check box.
- **8** In the associated text field, type Power (W).
- **9** Locate the Legend section. From the Position list, choose Lower right.

Global I

- I Right-click Electric Input Power and choose Global.
- 2 In the Settings window for Global, click Replace Expression in the upper-right corner of the y-axis data section. From the menu, choose Component I>Definitions>Variables> P_E - Electric input power (rms) - W.
- 3 In the Electric Input Power toolbar, click Plot.
- 4 Click the x-Axis Log Scale button in the Graphics toolbar.

This plot should reproduce Figure 5 (bottom left).

ID Plot Group 10

- I In the Home toolbar, click Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Driver Efficiency in the Label text field.
- 3 Locate the Title section. From the Title type list, choose Manual.
- 4 In the **Title** text area, type Driver Efficiency.
- **5** Locate the **Plot Settings** section. Select the **x-axis label** check box.
- 6 In the associated text field, type f (Hz).
- 7 Select the y-axis label check box.
- 8 In the associated text field, type Efficiency (%).
- **9** Locate the **Legend** section. From the **Position** list, choose **Lower middle**.

Global I

- I Right-click Driver Efficiency and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
eta*100	1	Driver efficiency (COMSOL)
eta0*100		Reference efficiency (piston)

- 4 In the Driver Efficiency toolbar, click Plot.
- 5 Click the x-Axis Log Scale button in the Graphics toolbar.

This plot should reproduce Figure 5 (bottom right).

ID Plot Group II

- I In the Home toolbar, click Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Force on Speaker Cone in the Label text field.
- **3** Locate the **Title** section. From the **Title type** list, choose **Manual**.
- 4 In the Title text area, type Force on Speaker Cone in Axial z direction.
- 5 Locate the Plot Settings section. Select the x-axis label check box.
- 6 In the associated text field, type f (Hz).
- **7** Select the **y-axis label** check box.
- 8 In the associated text field, type Pressure force (N).

9 Locate the **Legend** section. From the **Position** list, choose **Lower left**.

Global I

- I Right-click Force on Speaker Cone and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
sqrt(0.5*F_D*conj(F_D))	N	Pressure force (rms) (COMSOL)

- 4 In the Force on Speaker Cone toolbar, click Plot.
- 5 Click the x-Axis Log Scale button in the Graphics toolbar.

This plot should reproduce Figure 6 (top left).

ID Plot Group 12

- I In the Home toolbar, click Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Voice-Coil Impedance in the Label text field.
- 3 Locate the Title section. From the Title type list, choose Manual.
- **4** In the **Title** text area, type Voice-Coil Impedance.
- 5 Locate the Plot Settings section. Select the x-axis label check box.
- 6 In the associated text field, type f (Hz).
- 7 Select the y-axis label check box.
- 8 In the associated text field, type Impedance (ohm).

Global I

- I Right-click Voice-Coil Impedance and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
abs(VO/cir.R1_i)	ohm	Z
real(VO/cir.R1_i)	ohm	Re(Z)
imag(V0/cir.R1_i)	ohm	Im(Z)

4 In the Voice-Coil Impedance toolbar, click Plot.

5 Click the x-Axis Log Scale button in the Graphics toolbar.

This plot should reproduce Figure 6 (top right).

ID Plot Group 13

- I In the Home toolbar, click Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Sensitivity in the Label text field.
- 3 Locate the Title section. From the Title type list, choose Manual.
- 4 In the Title text area, type Speaker Response: 1†m in front driven at 1 V (rms).
- 5 Locate the Plot Settings section. Select the x-axis label check box.
- 6 In the associated text field, type f (Hz).
- 7 Select the y-axis label check box.
- **8** In the associated text field, type Level (dB SPL).
- **9** Locate the **Legend** section. From the **Position** list, choose **Lower left**.

Octave Band I

- I In the Sensitivity toolbar, click More Plots and choose Octave Band.
- 2 In the Settings window for Octave Band, locate the Selection section.
- 3 From the Geometric entity level list, choose Global.
- 4 Locate the y-Axis Data section. In the Expression text field, type pext(0,1).
- 5 Locate the Plot section. From the Style list, choose Continuous.
- 6 Click to expand the **Legends** section. Select the **Show legends** check box.
- 7 From the Legends list, choose Manual.
- **8** In the table, enter the following settings:

Legends				
Sound	pressure	level	(COMSOL)	

9 In the Sensitivity toolbar, click Plot.

Octave Band 2

- I Right-click Octave Band I and choose Duplicate.
- 2 In the Settings window for Octave Band, locate the y-Axis Data section.
- 3 In the Expression text field, type sqrt(2)*prms.

4 Locate the **Legends** section. In the table, enter the following settings:

Legends				
Sound	pressure	level	(piston)	

5 In the Sensitivity toolbar, click Plot.

This plot should reproduce Figure 6 (bottom left).

ID Plot Group 14

- I In the Home toolbar, click Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Phase in the Label text field.
- 3 Locate the Title section. From the Title type list, choose Manual.
- 4 In the Title text area, type Phase (relative to a plane wave at 1[m]).
- **5** Locate the **Plot Settings** section. Select the **x-axis label** check box.
- 6 In the associated text field, type f (Hz).
- 7 Select the y-axis label check box.
- 8 In the associated text field, type Phase (rad).

Global I

- I Right-click Phase and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
arg(pext(0,1)/exp(-i*k0*1[m]))	rad	Phase (COMSOL)

- 4 In the Phase toolbar, click Plot.
- 5 Click the x-Axis Log Scale button in the Graphics toolbar.

This plot should reproduce Figure 6 (bottom right).

Set up a parameterized curve used to evaluate the exterior-field outside of the computational mesh.

Parameterized Curve 2D I

- I In the Results toolbar, click More Datasets and choose Parameterized Curve 2D.
- 2 In the Settings window for Parameterized Curve 2D, locate the Expressions section.
- 3 In the z text field, type s*5[m]+(1-s)*Rair.
- 4 Select the Only evaluate globally defined expressions check box.

ID Plot Group 15

- I In the Results toolbar, click ID Plot Group.
- 2 In the Settings window for ID Plot Group, type On-axis Pressure in the Label text field.
- 3 Locate the Data section. From the Dataset list, choose Parameterized Curve 2D 1.
- 4 Locate the Title section. From the Title type list, choose Manual.
- 5 In the **Title** text area, type On-axis Pressure Field.
- **6** Locate the **Plot Settings** section. Select the **x-axis label** check box.
- 7 In the associated text field, type z (m).
- 8 Select the y-axis label check box.
- **9** In the associated text field, type Pressure (Pa).
- 10 Locate the Data section. From the Parameter selection (freq) list, choose From list.
- II In the Parameter values (freq (Hz)) list, select 1000.

Line Grabh I

- I Right-click On-axis Pressure and choose Line Graph.
- **2** Select Boundary 4 only.
- 3 In the Settings window for Line Graph, locate the y-Axis Data section.
- 4 In the Expression text field, type pext(r,z).
- 5 Locate the x-Axis Data section. From the Parameter list, choose Expression.
- **6** In the **Expression** text field, type **z**.
- 7 Click to expand the Quality section. From the Resolution list, choose Extra fine.
- **8** Click to expand the **Legends** section. Select the **Show legends** check box.
- 9 From the Legends list, choose Manual.
- **10** In the table, enter the following settings:

Legends

COMSOL

Line Graph 2

- I Right-click Line Graph I and choose Duplicate.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- 3 In the Expression text field, type acpr.iomega*rho0*u_D*S_D*exp(-i*k0*z)/(2* pi*z).

4 Locate the **Legends** section. In the table, enter the following settings:

Legends Piston

5 In the On-axis Pressure toolbar, click Plot.

This plot should reproduce Figure 7.

Create a 2D intensity plot that includes the magnitude of the intensity vector acpr. I rms as well as an arrow surface (vector field plot) of the intensity vector, with the components (acpr.Ir,acpr.Iz).

2D Plot Group 16

- I In the Home toolbar, click Add Plot Group and choose 2D Plot Group.
- 2 In the Settings window for 2D Plot Group, type Intensity in the Label text field.

Surface I

- I Right-click Intensity and choose Surface.
- 2 In the Settings window for Surface, locate the Expression section.
- 3 In the Expression text field, type acpr. I rms.

Arrow Surface 1

- I In the Model Builder window, right-click Intensity and choose Arrow Surface.
- 2 In the Settings window for Arrow Surface, click Replace Expression in the upper-right corner of the Expression section. From the menu, choose Component 1> Pressure Acoustics, Frequency Domain>Intensity>acpr.lr,acpr.lz - Intensity.
- 3 Locate the Coloring and Style section. From the Color list, choose Black.

Arrow Line 1

- I Right-click Intensity and choose Arrow Line.
- 2 In the Settings window for Arrow Line, click Replace Expression in the upper-right corner of the Expression section. From the menu, choose Component I>Pressure Acoustics, Frequency Domain>Geometry>acpr.nr,acpr.nz - Normal vector.
- 3 Locate the Coloring and Style section. From the Color list, choose Black.
- 4 In the Intensity toolbar, click Plot.
- 5 Click the **Zoom Extents** button in the **Graphics** toolbar.

This plot should reproduce the last frame in Figure 10, now change the evaluation frequency to 5000 Hz, 1000 Hz, and 100 Hz. This will reproduce the remaining frames in Figure 9 and Figure 10.

Intensity

- I In the Model Builder window, click Intensity.
- 2 In the Settings window for 2D Plot Group, locate the Data section.
- 3 From the Parameter value (freq (Hz)) list, choose 4923.9.
- 4 In the Intensity toolbar, click Plot.
- 5 From the Parameter value (freq (Hz)) list, choose 1000.
- 6 In the Intensity toolbar, click Plot.
- 7 From the Parameter value (freq (Hz)) list, choose 100.
- 8 In the Intensity toolbar, click Plot.

Next create the directivity plot of the speaker.

ID Plot Group 17

- I In the Home toolbar, click Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Directivity in the Label text field.

Directivity I

- I In the Directivity toolbar, click More Plots and choose Directivity.
- 2 In the Settings window for Directivity, locate the Evaluation section.
- 3 Find the Angles subsection. From the Restriction list, choose Manual.
- 4 In the ϕ start text field, type -90.
- 5 In the ϕ range text field, type 180.
- 6 Click to expand the Coloring and Style section. From the Layout list, choose Frequency on y-axis.
- 7 In the Directivity toolbar, click Plot.
- 8 Click the y-Axis Log Scale button in the Graphics toolbar.

This should reproduce the directivity plot depicted in Figure 11. You can tailor the plot to your needs using the normalization options or defining the specific levels to use in the contour plot.

Create an additional plot that shows the sensitivity and the phase in one plot using two y-axes.

Global I

In the Model Builder window, right-click Global I and choose Copy.

Sensitivity I

I In the Model Builder window, right-click Sensitivity and choose Duplicate.

- 2 In the Settings window for ID Plot Group, type Sensitivity and Phase in the Label text field.
- 3 Locate the Title section. From the Title type list, choose None.

Octave Band 2

- I In the Model Builder window, expand the Results>Sensitivity and Phase node.
- 2 Right-click Octave Band 2 and choose Delete.

Octave Band I

- I In the Settings window for Octave Band, locate the Plot section.
- 2 From the Style list, choose 1/3 octave bands.

Global I

- I In the Model Builder window, right-click Sensitivity and Phase and choose Paste Global.
- 2 In the Settings window for Global, click to expand the Coloring and Style section.
- 3 In the Width text field, type 2.
- 4 From the Color list, choose Red.

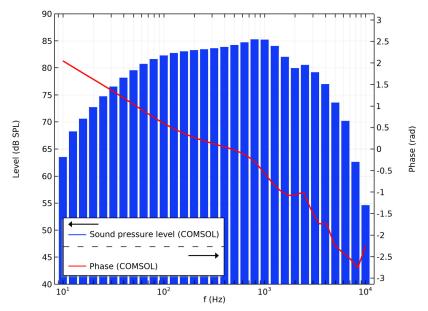
Sensitivity and Phase

- I In the Model Builder window, click Sensitivity and Phase.
- 2 In the Settings window for ID Plot Group, locate the Plot Settings section.
- 3 Select the Two y-axes check box.
- **4** In the table, enter the following settings:

Plot	Plot on secondary y-axis
Global I	$\sqrt{}$

- 5 Locate the Axis section. Select the Manual axis limits check box.
- **6** In the **y minimum** text field, type 40.
- 7 In the y maximum text field, type 90.
- 8 In the Secondary y minimum text field, type -pi.
- 9 In the Secondary y maximum text field, type pi.
- 10 Locate the Plot Settings section. Select the Secondary y-axis label check box.
- II In the associated text field, type Phase (rad).





It is time to solve the model with the closed back-volume configuration. After solving add a few extra graphs to the existing plots to see how the system response has changed. The closed back-volume acts as a spring and thus modifies the mechanical compliance of the system. The effect is, for example, seen in the electrical impedance plot by moving the resonance up in frequency.

ADD STUDY

- I In the Home toolbar, click Add Study to open the Add Study window.
- **2** Go to the **Add Study** window.
- 3 Find the Studies subsection. In the Select Study tree, select General Studies> Frequency Domain.
- 4 Click Add Study in the window toolbar.
- 5 In the Home toolbar, click Add Study to close the Add Study window.

STUDY 2

Step 1: Frequency Domain

- I In the Settings window for Frequency Domain, locate the Study Settings section.
- 2 In the Frequencies text field, type 10^{range(1,3/39,4)}.

- 3 In the Model Builder window, click Study 2.
- 4 In the Settings window for Study, type Study 2 Closed Back-Volume in the Label text field.
- 5 Locate the Study Settings section. Clear the Generate default plots check box.
- 6 In the Home toolbar, click Compute.

Finally, modify two of the existing plots to understand the effect of a closed-back volume. To see the results of the closed configuration, in any of the plots, simply change the **Data set** to the one from study 2.

RESULTS

Global I

- I In the Model Builder window, under Results>Voice-Coil Impedance click Global I.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
abs(V0/cir.R1_i)	ohm	Z (open back-volume)

Global 2

- I Right-click Results>Voice-Coil Impedance>Global I and choose Duplicate.
- 2 In the Settings window for Global, locate the Data section.
- 3 From the Dataset list, choose Study 2 Closed Back-Volume/Solution 2 (sol2).
- **4** Locate the **y-Axis Data** section. In the table, enter the following settings:

Expression	Unit	Description
abs(VO/cir.R1_i)	ohm	Z (closed back-volume)

- 5 Locate the Coloring and Style section. From the Color list, choose Cycle (reset).
- 6 Find the Line style subsection. From the Line list, choose Dashed.
- 7 In the Voice-Coil Impedance toolbar, click Plot.

This plot should reproduce Figure 12.

Octave Band I

- I In the Model Builder window, click Octave Band I.
- 2 In the Settings window for Octave Band, locate the Legends section.

3 In the table, enter the following settings:

Legends Sound pressure level (COMSOL) (open back-volume)

Octave Band 3

- I Right-click Results>Sensitivity>Octave Band I and choose Duplicate.
- 2 In the Settings window for Octave Band, locate the Data section.
- 3 From the Dataset list, choose Study 2 Closed Back-Volume/Solution 2 (sol2).
- **4** Locate the **Legends** section. In the table, enter the following settings:

Legends				
Sound pressure	level	(COMSOL)	(closed	back-volume)

5 In the Sensitivity toolbar, click Plot.

This plot should reproduce Figure 13.